AD-A104 090 UNCLASSIFIED	SOUTHWEST MULTICYLI JUN 81 C	RESEARCH INSI NDER DIESEL EN W COON	SAN AN IGINE TES	ITONIO ITS WIT	TX H UNSTA	MBILIZE	D WATER	F/6 -IN-FU- 587	13/10 -ETC(U)	`
1 16 2 AUX 090 1	o									
					15/1					
					Į.Ją					

1-50

LEVEL

12

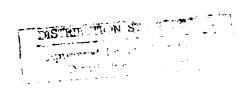
CG-D-27-80

AD A104090

# MULTICYLINDER DIESEL ENGINE TESTS WITH UNSTABILIZED WATER-IN-FUEL EMULSIONS



REPRINT JUNE 1981



U.S. DEPARTMENT OF TRANSPORTATION

RESEARCH AND SPECIAL PROGRAMS ADMINISTRATION
TRANSPORTATION SYSTEMS CENTER ● CAMBRIDGE MA 02142



PREPARED FOR UNITED STATES COAST GUARD
OFFICE OF RESEARCH AND DEVELOPMENT • WASHINGTON DC 20593

REPRINT JUNE 198

81 9 01 059

# NOTICE

This document is disseminated under the sponsorship of the Department of Transportation in the interest of information exchange. The United States Government assumes no liability for its contents or use thereof.

# NOTICE

The United States Government does not endorse products or manufacturers. Trade or manufacturers' names appear herein solely because they are considered essential to the object of this report.

بالأ		750141	ICAL DEBORT STA	NOADO TITI E BACE		
1. Report No.	2. Government Ac		Recipient's Catalog	NDARD TITLE PAGE		
CG-D-27-80		.,				
	AD-A10					
4. Title and Subtitle	TEL ENGINE ME	1	Report Date Reprint Jun	<b>e-19</b> 81		
MULTICYLINDER DIES UNSTABILIZED WATE			Performing Organiza			
ONSTABILIZED WATER	V-IN-LOEF FW	DESTONS 4	-	gtion code		
7 0 110 110		9	11-5477 Performing Organiz	ation Report No.		
7. Author(s) C. W./Coon, Jr		8.	- or forming Organiza	ation responding		
or / coon, or /			DOT-TSC-USCG	-80-7		
9. Performing Organization Name and Address	i	10.	Work Unit No.			
Southwest Research Inst:	itute*		CG 107/R 101	4		
6220 Culebra Road		11.	Contract or Grant	<del>1</del> 0,		
San Antonio TX 78284			DOT-TSC-158	7 —		
		13.	Type of Report and	Period Covered		
12. Sponsoring Agency Name and Address	_	· · · · · · · · · · · · · · · · · · ·	<u>F</u> INAL	REP <b>CR</b> T _		
U.S. Department of Trans U.S. Coast Guard	_	Sep	tember 1978-	Jul <b>y 19</b> 80		
Office of Research and I Washington DC 20593	Development	14.	Sponsoring Agency G-DMT-3	Code		
15. Supplementary Notes U.	S. Department	of Transportation	1			
		ecial Programs Adm		1		
		Systems Center (D)	CS-332)			
Ca	mbridge MA (	)2142				
16. Abstract						
Two diesel engines repre						
main propulsion units install						
a test environment in an atte						
with water-in-fuel emulsions laboratory test cell. A prof						
stabilized emulsions for which						
percent of the total volume of						
boat operation was performed						
settings, and both engines we						
load performance.	•					
The test results for the	e four-stroke	cycle engine indi	cated that a	n average		
diesel fuel saving of about 2						
encountered operating conditi						
tistical analysis procedures						
Significant reductions in exh						
stream opacity was low through						
statistically significant rec						
Measurements of gaseous general, the emissions increa						
ments of particulate emission						
slight effect of water concer						
associated with the presence						
17. Key Words		18. Distribution Statement				
			A 0. E TO THE DUE	uc.		
Diesel engines Water-in-fuel emulsions		DOCUMENT IS AVAIL THROUGH THE NATION	ONAL TECHNICAL			
Fuel consumption		INFORMATION SERVI		ì		
Exhaust emissions		VIRGINIA 22161				
EMIGROT CHIZOTONIC	r	<u> </u>	F 22 - 2 - 2 - 2 - 2 - 2 - 2 - 2 - 2 - 2			
19. Security Classif. (of this report)	20. Security Clas		21. No. of Pages	22. Price		
Unclassified	Uncl	assified	154			

#### PREFACE

This work was performed for the U.S. Department of Transportation, U.S. Coast Guard, Office of Research and Development, under a contract issued by the Transportation Systems Center. The Technical Monitors were Fred Weidner (USCG) and Robert Walter (TSC). The laboratory tests were performed by Rodney Bauer of the Department of Engine and Vehicle Research, Southwest Research Institute.

Engines were made available to the program by the Cummins Engine Company, Inc., and by the Detroit Diesel Allison Division of General Motors Corporation. The cooperation of these organizations is sincerely appreciated.

Accession For
NTIS GRA&I
DTIC TAB
Unannounced []
Justification
Distribution/ Availability Codes
Myail and/or Dist Special
Special
A

METRIC CONVERSION FACTORS

	į		•	•	e 1	Łŧ			7	· ``	'n.				*	•			1		<b>s</b> i	i 2	<b>3</b>			•		<b>\</b>	# <del>-</del>	] }	1
etric Measures	. Fi	ŀ	arches	riches	<b>1</b>				1		seres aries	9639		Ĺ					! [		Ī		1		nect)				150 0\$1		
Appreximate Conversions from Matric Moasures	Maltiphy by	LENGTH	8	7	3.3	2;	;	AREA		#. 6.78 	7.	2.5 1, w. 1		MASS (weight	***					8	•	<b>%</b>	g <u></u>		TEMPERATURE (onset)		1 R		2 - S	12	
Approximate Co.	What You Know	•		Continuite	meters	Ę	L. Homeston		•	September Construction		hectores (10,000 m <sup>2</sup>		•	į				•	An Hebitara	į	interes	Cubic meters				Colours		•	} *} *	
	24872			ŧ!	Ŝε	•	5			<b>}</b> -	ر" <u>،</u>	2				- 3	•			Ŧ.		_	`e ^e				ب				
       ES	22		<b>&gt;2</b>		 	lin			91 	Helli	83 		111111	i Idili	111111	Allai		ot III					\    12 <b>1</b> 013	,		HIDA	) Interdes				
]  -	11711		ļ.i.	'  ' 	' ' <b>!</b>	`['	, I.Î.I	']''	. <b>.</b> 1.1.		44	`[']'   	.İ.1.	' '       	ļ,1,	'l'  	'I'    	'l'  •	\\ \ \	l' <b> </b> '	]']'     	' '	נ'נ''   	<b>'</b>	l' '   	' '	1.1.1.	ייין.	''' '   	ייים    ני!ו!י!	1111 Act
		į				5	8 €	5		•		reî				•	<b>1</b> .			Ē	Ē	<b>i</b> -			`£	è		٠			
Matric Mossures		7. 2.				Centimeters	Centimaters meters	h : lgmeter s			Square Cartumeters	Square melers	square kelamaters hecterss			-	h: logiens			and the first own	m.Hitisers	m.H.Inters		<b>:</b>	Cubic meters	Cubic meters		Cetews	temporatura.		
		Mettiph by	7	LEMBIR		5.5	g 5	4.	AREA		<b>9</b> 1	8 3	9.7	:	MASS (weight)	2	\$ .	}	V. UME		. 5	2	0.47	X.	. 8	<b>K</b> :	TEMPERATURE (exact)	Av9 (after	(A)		
Apprezimete Conversions to	:	When You Know		1		mchos	ijį	e e		1	sensor suges		square miles		- <sub>1</sub>	100	-	(3000)		,		fluid Gunche	įį		Sellen.	Culture yands	21				
		ļ				9	<u>.</u> 1	1			٦,	ኔ ን	ı"ı								<u>.</u>	1	. 1	<b>.</b> %	is	·			•		

# TABLE OF CONTENTS

Section	<u> </u>	Page
1.	INTRODUCTION	1
	1.1       Background	1 2 3
2.	EXPERIMENTAL APPARATUS AND TECHNIQUES	6
	2.1 Equipment	6
	2.1.1 Engines	6 7 7 13
	2.2 Test Procedure	25 31
3.	RESULTS	34
	3.1 Fuel Consumption	34 45 45 45 55 58 63
4.	SUMMARY AND CONCLUSIONS	64
5.	RECOMMENDATIONS	68
APPENDI	X A - FUEL PROPERTIES	69
APPENDI	X B - SAMPLE CALCULATIONS	73
APPENDI	X C - TEST RESULTS	98
APPENDI	X D - REPORT OF NEW TECHNOLOGY	141
DEFEREN	OEC	142

# LIST OF ILLUSTRATIONS

Figure		Page
2-1	ENGINE INSTALLATION; CUMMINS ENGINE TESTS	9
2-2	ENGINE INSTALLATION; DETROIT DIESEL TESTS	10
2-3	TYPICAL HYDROSHEAR	11
2-4	PROTOTYPE EMULSIFIED FUEL SYSTEM	12
2-5	HYDROCARBON ANALYZER	16
2-6	ANALYZER SYSTEM FOR CARBON MONOXIDE AND CARBON DIOXIDE	17
2-7	ANALYZER SYSTEM FOR OXIDES OF NITROGEN	18
2-8	EMISSION INSTRUMENT CONSOLE, FRONT VIEW	23
2-9	EMISSION INSTRUMENT CONSOLE, END VIEW	24
2-10	PARTICULATE MEASURING SYSTEM	26
2-11	ENGINE SPEED AND POWER OUTPUT, MAXIMUM HORSEPOWER AND PROP LOAD, CUMMINS ENGINE	27
2-12	OPERATING TIME AS FUNCTION OF ENGINE SPEED FOR CUMMINS ENGINE-POWERED CUTTERS	29
2-13	ENGINE SPEED AND POWER OUTPUT, MAXIMUM HORSEPOWER AND PROP LOAD, DETROIT DIESEL TWELVE-CYLINDER ENGINE	30
2-14	COMPARISON OF CARBON BALANCE AND MEASURED FUEL-AIR RATIO, CUMMINS ENGINE TESTS	33
3–1	FUEL CONSUMPTION, CUMMINS ENGINE, 1800 RPM	35
3-2	FUEL CONSUMPTION, CUMMINS ENGINE, 1200 RPM	36
3-3	FUEL CONSUMPTION, CUMMINS ENGINE, 900 RPM	38
3-4	FUEL CONSUMPTION, DETROIT DIESEL ENGINE, 1000 RPM	39
3-5	FUEL CONSUMPTION, DETROIT DIESEL ENGINE, FOUR SPEEDS	40
3-6	FUEL INJECTION TIMING AS FUNCTION OF ADJUSTMENT DIMENSION, DETROIT DIESEL ENGINE, 12V-149TI (180 INJECTORS)	42

# LIST OF ILLUSTRATIONS (CONTINUED)

Figure		Page
3-7	EFFECT OF INJECTION TIMING ON FUEL CONSUMPTION, DETROIT DIESEL ENGINE, 1000 RPM	43
3-8	EFFECT OF INJECTION TIMING ON FUEL CONSUMPTION, DETROIT DIESEL ENGINE, 1400 RPM	44
3-9	EXHAUST SMOKE, CUMMINS ENGINE, 1800 RPM	46
3-10	EXHAUST SMOKE, CUMMINS ENGINE, 1200 RPM	47
3-11	EXHAUST PARTICULATE EMISSIONS AS FUNCTION OF PERCENT WATER, DETROIT DIESEL ENGINE	48
3–12	EXHAUST PARTICULATE EMISSIONS AS FUNCTION OF ENGINE SPEED, DETROIT DIESEL ENGINE	49
3-13	EMISSIONS OF NITRIC OXIDE, CUMMINS ENGINE, 1800 RPM	51
3-14	EMISSIONS OF NITRIC OXIDE, CUMMINS ENGINE, 1200 RPM	52
3-15	TIMING OF THE BEGINNING OF FUEL INJECTION, CUMMINS ENGINE	53
3–16	EMISSIONS OF OXIDES OF NITROGEN, DETROIT DIESEL ENGINE, FIVE SPEEDS	54
3–17	EMISSIONS OF UNBURNED HYDROCARBONS, CUMMINS ENGINE, 1800 RPM	56
3-18	EMISSIONS OF UNBURNED HYDROCARBONS, CUMMINS ENGINE, 1200 RPM	57
3-19	EMISSIONS OF UNBURNED HYDROCARBONS, DETROIT DIESEL ENGINE, FIVE SPEEDS	59
3-20	EMISSIONS OF CARBON MONOXIDE, CUMMINS ENGINE, 1800 RPM	60
3-21	EMISSIONS OF CARBON MONOXIDE, CUMMINS ENGINE, 1200 RPM	61
3-22	EMISSIONS OF CARBON MONOXIDE, DETROIT DIESEL ENGINE,	62

# LIST OF TABLES

<u>Table</u>		Page
2-1	ENGINE SPECIFICATIONS	8
2-2	DATA OBTAINED FROM TEST CELL	14
2-3	INSTRUMENTS AND RANGES ON L-4 EMISSIONS CART	19
2-4	SWRI HEATED HYDROCARBON ANALYZER FLOW SCHEMATIC COMPONENT DESCRIPTION	20
2-5	NDIR CO AND CO <sub>2</sub> FLOW SCHEMATIC COMPONENT DESCRIPTION	21
2-6	HEATED CHEMILUMINESCENT NO <sub>X</sub> ANALYZER FLOW SCHEMATIC COMPONENT DESCRIPTION	22
3-1	DETROIT DIESEL 12V-149TI ENGINE FUEL INJECTION TIMING	41
A-1	FUEL ANALYSIS DATA	70
A-2	MARINE ENGINE PERFORMANCE CURVE	71
A-3	ESTIMATED PERFORMANCE SERIES V-149TI MARINE 16V-149TI CREW BOAT, JACKET WATER INTERCOOLER, 150 INJECTORS	72
B-1	TEST DATA	74
B-2	WATER FLOWMETER CURVE COEFFICIENTS	79
B-3	AIR TEMPERATURE CORRECTION FACTORS	84
B-4	POPULATION SAMPLES	94
B-5	CUMULATIVE DISTRIBUTION	96
C-1	ENGINE TEST RESULTS, CUMMINS ENGINE, 900 RPM, BASELINE	99
C-2	ENGINE TEST RESULTS, CUMMINS ENGINE, 900 RPM, 5% WATER	100
C-3	ENGINE TEST RESULTS, CUMMINS ENGINE, 900 RPM, 10% WATER	101
C-4	ENGINE TEST RESULTS, CUMMINS ENGINE, 900 RPM, 15% WATER	102
C-5	ENGINE TEST RESULTS. CHMMINS ENGINE. 900 RPM. 20% WATER	103

# LIST OF TABLES (CONTINUED)

<u>Table</u>		Page
C-6	ENGINE TEST RESULTS, CUMMINS ENGINE, 900 RPM, 25% WATER	104
C-7	ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, BASELINE	105
C-8	ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, 5% WATER	106
C-9	ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, 10% WATER	107
C-10	ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, 15% WATER	108
C-11	ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, 20% WATER	109
C-12	ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, 25% WATER	110
C-13	ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, BASELINE	111
C-14	ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, 5% WATER	113
C-15	ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, 10% WATER	114
C-16	ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, 15% WATER	115
C-17	ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, 20% WATER	116
C-18	ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, 25% WATER	117
C-19	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 500 RPM	118
C-20	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 800 RPM, BASELINE	119
C-21	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 800 RPM, WITH WATER ADDITION	120
C-22	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, BASELINE	121
C-23	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, 5, 10, 15% WATER	122
C-24	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, 20, 25% WATER	123
C-25	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, FUEL INJECTION TIMING RETARDED 2.4 DEGREES	124

# LIST OF TABLES (CONTINUED)

<u>Table</u>		Page
C-26	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, FUEL INJECTION TIMING RETARDED 4.1 DEGREES	125
C-27	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, FUEL INJECTION TIMING ADVANCED 2.8 DEGREES	126
C-28	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, FUEL INJECTION TIMING ADVANCED 5.5 DEGREES	127
C-29	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1200 RPM, BASELINE	128
C-30	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1200 RPM, WITH WATER ADDITION	129
C-31	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1400 RPM, BASELINE	130
C-32	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1400 RPM, WITH WATER ADDITION	131
C-33	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1400 RPM, FUEL INJECTION TIMING RETARDED 4.1 DEGREES	132
C-34	ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1600 RPM, BASELINE	133
C-35	PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE, 600 RPM	134
C-36	PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE, 800 RPM	135
C-37	PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM	136
C-38	PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE, 1200 RPM	137
C-39	PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE, 1200 RPM, 15, 20, and 25% WATER	138
C-40	PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE, 1400 RPM	139
C-41	PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE,	140

# 1. INTRODUCTION

The current emphasis on fuel conservation has prompted the study of many devices and techniques oriented toward a reduction in engine fuel consumption. This report describes the procedures used and the results obtained during a study of unstabilized water-in-fuel emulsions as fuels for engines representative of U.S. Coast Guard main propulsion systems.

The complete program involved an investigation of two high-speed diesel engines with nominal maximum power ratings in the 1000 hp range. A Cummins VTA-1710 engine was used to represent the military version (VT12-900M) that is utilized by the USCG. In addition, a Detroit Diesel 12V-149TI engine was employed for the acquisition of data representative of the 16 cylinder version installed in the USCG cutters.

#### 1.1 BACKGROUND

The use of water-in-fuel emulsions for fuel conservation has been a subject of continuing interest for several years. During 1977, Southwest Research Institute conducted a program for the Department of Transportation in which fuel-water emulsions were examined in the context of a single-cylinder test engine. The results obtained during that study indicated that a reduction in fuel consumption on the order of five percent might be available for engines representative of marine propulsion. It was recommended that the testing effort be continued using multi-cylinder engines, and the present study describes the partial fulfillment of that recommendation.

Various investigators have recommended the use of different devices and philosophies for the production of water-in-fuel emulsions used as engine fuels. One approach suggests the addition of surfactant compounds to the fuel-water mixture. The surfactants stabilize the emulsions and allow batch mixing of fuel supplies. For the present study, however, this approach was not considered feasible, since the USCG would prefer to avoid the requirement for precise blending of fuel additives with the large quantities of fuel utilized for patrol boats. Furthermore, it was considered necessary to view

water of Miller .

the emulsified fuel as an option to clear diesel fuel, and the storage of a second bulk fuel aboard existing cutters was not considered appropriate.

This study deals with the use of unstabilized emulsions as engine fuels. The water and fuel were emulsified immediately prior to use by the engine, and emulsified fuel that was supplied to the engine but not burned was recycled through the emulsifier circuit.

#### 1.2 OBJECTIVES

The initial goal of the study was the identification and the specification of an appropriate emulsification device. It had previously been observed that some devices offered for use as fuel emulsification systems were inappropriate to shipboard operation by virtue of physical size or power requirements. Furthermore, it was not clear that all devices were equally capable of providing useful emulsions. Therefore, the initial program phase was devoted to selection of an emulsification device.

Following the specification and purchase of a suitable emulsifier, the program objective was the determination of the effect of the emulsified fuels on engine operation. All tests were performed in a laboratory environment; the engines were connected to a dynamometer. Particular emphasis was placed upon fuel conservation, but measurements of exhaust emissions were also obtained. A further objective was the determination of results on a statistically significant basis in order that a numerical confidence could be placed in the results.

As an additional program objective, it was specified that encouraging results from the test bed studies would yield design information appropriate to the assembly of a prototype fuel emulsification system for shipboard use. The final program phase would involve field testing of the prototype on a USCG cutter.

#### 1.3 APPROACH

The program was initiated with a selection process devoted to the definition of emulsification systems appropriate to the study. Invitations were sent to all individuals or companies known to be involved in the development of emulsification systems, and an advertisement was placed in the Commerce Business Daily that outlined the program requirements. Six prospective suppliers responded to the invitation and offered devices for evaluation in the SwRI laboratory.

A system was provided by SwRI that would supply metered quantities of fuel and water to the prototype systems on a uniform basis. The emulsification systems were exercised within their performance limits as defined by the supplier, and samples of water-in-fuel emulsion were obtained over the 0 to 25 percent concentration range that was of interest. Immediately following collection of each sample, the time required for accumulation of an obvious separation layer was observed. This process allowed the assessment of the capability of each device to produce an emulsion that would be useful during the engine studies.

In addition, each prospective emulsification device was evaluated on the basis of energy usage, physical size, complexity, compatibility with the shipboard environment, and the need for auxiliary hardware such as pumps and controllers. Individual evaluations were performed by representatives of SwRI, the Transportation Systems Center, and the USCG. As a result of the evaluation process, two emulsification systems were selected for use during the engine operation phase of the program, and purchase orders for units of an appropriate size were executed.

Since the response of the engine fuel system to the presence of water was unknown, a brief sequence of test runs was performed using stabilized emulsions containing 5 and 20 percent water by volume. The single purpose of these tests was the determination of any observable detrimental effects on engine operation as a result of water in the fuel system. No detrimental effects were observed, therefore the testing with unstabilized mixtures was initiated.

and the same of the same

During all of the test sequences the laboratory engine was operated at controlled speeds and loads representative of the cutter prop load at each speed. On this basis, the maximum engine output would be observed only at the engine rated speed. Although some test runs were performed at maximum speed and load, the data were of interest only for certification of the fact that the engine met the ratings specified by the manufacturer. At the other test speeds the prop loads were considerably below the maximum possible engine output.

Throughout the bulk of the test program, the engine remained in the configuration appropriate to the use of diesel fuel alone. Since the fuel-water emulsion was regarded as an alternate fuel for boat operation, it was not considered appropriate to optimize the engine operating parameters for emulsified fuel use. It is possible that fuel consumption results different from those obtained could have been achieved by adjusting the engine operating parameters; the injection timing is particularly significant in this context, and some tests were performed with timing variation on the two-stroke cycle engine.

The emulsifier purchased for use during the tests was integrated into a fuel system capable of supplying the engine demand at any speed and load. The fuel system included ample provision for maintaining the state of the emulsions and for re-emulsifying fuel returned from the engine. In addition, a provision was included to allow sampling and subsequent observation of quantities of emulsified fuel immediately prior to introduction into the engine. Observation of these samples allowed additional verification of the water concentration of the emulsion.

During the test runs the engine was operated at selected points on the prop load curve with various water concentrations. Each test sequence, which occupied one day, involved operation of the engine at a single speed-load point. The first test run was performed with clear diesel fuel, followed by tests at 5, 10, 15, 20, and 25 percent water. Upon completion of these runs the fuel system was flushed and the test with clear diesel fuel was

repeated. During the performance of each test run, extensive observations of fuel consumption and other engine operating parameters were made.

The complete body of data includes information at speeds along the entire prop load curve for each test engine. In addition, the data include extensive testing at two speed-load points for the Cummins engine; the two points represent high utilization by operating USCG cutters. The data allow the determination of the optimum water concentration at each speed-load point; this information would be useful for the design of a shipboard control system.

# EXPERIMENTAL APPARATUS AND TECHNIQUES

The performance of the experiments associated with this study involved specific equipment, test procedures, and data evaluation routines. The salient features of these items are described in this section.

# 2.1 EQUIPMENT

The equipment utilized for the evaluation of water-in-fuel emulsions consisted of engines installed in a laboratory test cell, a fuel system uniquely appropriate for the production and distribution of emulsified fuel, and a broad variety of instrumentation used for the acquisition of data. Each of these categories is worthy of an expanded description.

## 2.1.1 Engines

The engines used aboard USCG cutters of the 82-ft WPB class are manufactured by the Cummins Engine Company and characterized by Model No. VT12-900M. The units have a total displacement of 1710 cubic inches and are rated at 800 shaft horsepower at 2300 rpm. This rating corresponds to approximately 875 brake horsepower at the same speed.

The engine available to this effort was a Cummins Model VTA-1710-C800, the industrial version of the USCG engine described above. The engine is ordinarily rated at 800 brake horsepower at 2100 rpm. The engine was equipped with an automotive type AFC fuel pump rather than the older style marine pump. For the purposes of this study, the fuel pump and governor were modified to allow engine operation up to 2300 rpm, and a maximum power capability of 845 horsepower at 2300 rpm was demonstrated in the laboratory. This value is within the five percent tolerance specified by the manufacturer.

The military and industrial versions of the 1710 engine differ slightly in the timing at which the fuel injection event begins. This difference amounts to approximately three crank angle degrees, and this deviation was not deemed sufficient to warrant the major effort involved in a timing change.

The engines used to power 95-ft WPB cutters are manufactured by the Detroit Diesel Allison Division of the General Motors Corporation; the specific model designation is 16V-149TI. The displacement of each unit is 2384 cubic inches, and each engine is rated at 1235 shaft horsepower at 1800 rpm.

The engine that was available for use during this study was a Detroit Diesel Model 12V-149TI; this unit is a twelve-cylinder version of the USCG engine. During the testing program, the engine was operated at approximately the same horsepower output per cylinder that the marine version would produce. Thus, although the total output of the twelve-cylinder engine was low, the details of engine operation were quite representative of the sixteen-cylinder counterpart.

The major specifications of the engines used during the test program are outlined in Table 2-1, and data from the manufacturers is shown in Appendix A.

## 2.1.2 Dynamometer and Test Cell

Each engine was installed in a test cell at the SwRI laboratory and connected to an eddy-current dynamometer capable of absorbing up to 1000 horsepower. The engine installation is shown in Figures 2-1 and 2-2. The dynamometer utilized was an absorbing unit only; no motoring capability was available.

The engine speed was determined through the use of a magnetic pickup and a 60-tooth gear installed in the engine-dynamometer coupling. The speed signal was transmitted to a digital counter used as an output device, and, in addition, the signal was supplied to a dynamometer controller capable of maintaining engine speed within a tolerance of one rpm. The dynamometer beam load was measured through the use of a strain gauge type load-cell connected to an output device at the control console. The load-cell was subjected to a weekly deadweight calibration.

## 2.1.3 Fuel System

A fuel supply system was assembled that would meter, premix, and emulsify the fuel and water in concentrations that were of interest. Although certain

and the State of the last

#### TABLE 2-1. ENGINE SPECIFICATIONS

Cummins Engine Company, Inc. Model: VTA-1710-C800 (VT12-900M)

Type: Four Stroke Cycle Bore and Stroke: 5.5 x 6 No. of Cylinders: 12

Displacement: 1710 Cubic Inches Rated Horsepower: 800 at prop shaft

Rated Speed: 2300 RPM

General Motors Corporation
Detroit Diesel Allison Division
Model: 12Y-16TT (16Y-16TT)

Model: 12V-149TI (16V-149TI)
Type: Two Stroke Cycle

Bore and Stroke: 5.75 x 5.75 No. of Cylinders: 12 (16)

Displacement: 1788 Cubic Inches

Rated Horsepower: 900 (1200)

Rated Speed: 1800 RPM



FIGURE 2-1. ENGINE INSTALLATION; CUMMINS ENGINE TESTS

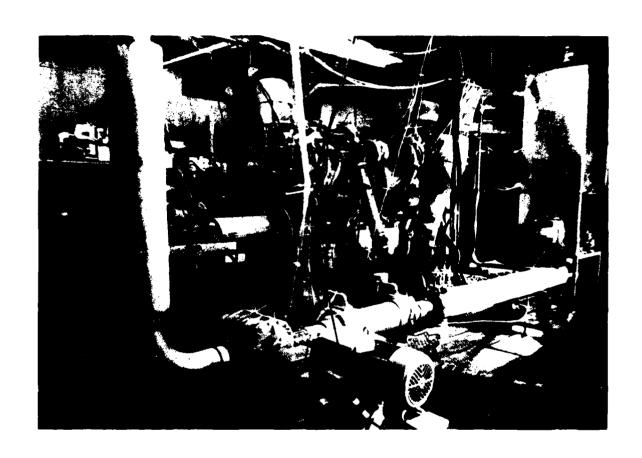


FIGURE 2-2. ENGINE INSTALLATION; DETROIT DIESEL TESTS

features of the system were designed specifically to accommodate the Cummins engine, the same system proved useful for both of the engines tested.

In ordinary operation the fuel would be supplied directly to the injection pump of the engine, and fuel not used by the engine would be returned to a storage tank. The Cummins injection system is unique in that the returned fuel typically contains quantities of gas which must be removed prior to recycling of the unburned fuel through the engine. In usual installations, this capability is provided by a vented storage tank.

For the purposes of this study, it was necessary to assemble a fuel system that would generate the fuel-water emulsion while simultaneously satisfying the requirement for degasification of the return fuel. A schematic diagram of the system used is shown in Figure 2-4, and the fuel system is visible in Figure 2-1. Fuel and water were supplied independently to a mixing tee; this device provided a crude mixture prior to emulsification. water was utilized throughout, and the line pressure provided the driving force. Fuel was pumped from a storage tank into the mixing arrangement. A constant fuel level was maintained in a float-controlled tank having a volume of approximately one-half gallon. This open tank allowed gases trapped in the return fuel to escape prior to fuel recycling. Fuel was removed from the float-controlled tank by a one horsepower gear pump which supplied a pressure of approximately 100 psi to the fuel-water emulsifier. The emulsifier used in this system was a Hydroshear device supplied by Gaulin Corporation; the unit operates by subjecting the fuel-water mixture to an extremely high shear state. A drawing of a typical Hydroshear after Lawson 11 is shown in Figure 2-3. The pressure at the outlet of the emulsifier was typically 20 to 25 psi.

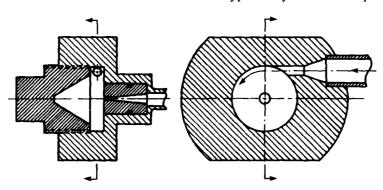
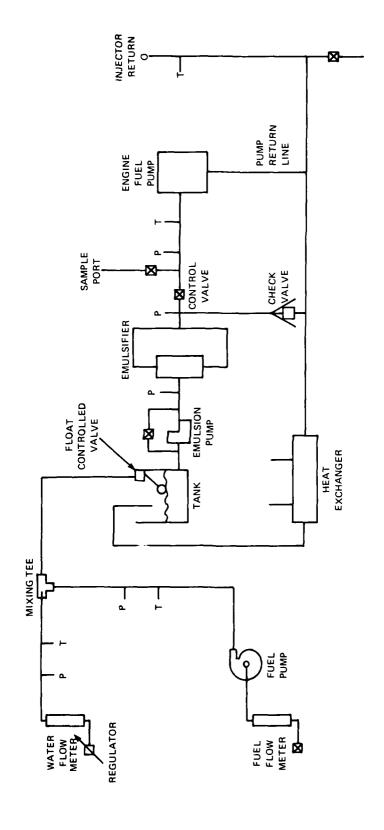


FIGURE 2-3. TYPICAL HYDROSHEAR



ļ

FIGURE 2-4. PROTOTYPE EMULSIFIED FUEL SYSTEM

At the emulsifier outlet, the fuel was directed either to the engine fuel pump or to a by-pass loop. Fuel directed toward the engine passed through a control valve which lowered the pressure to a value below 5 psi in order to meet the requirements of the engine fuel system. Fuel returned from either the engine fuel pump or the engine fuel injectors was routed into the by-pass portion of the system. The unused emulsion was conducted through a heat exchanger for cooling prior to return to the float-controlled tank. Pressures and temperatures were measured at points of interest throughout the fuel supply system, and a sample port was provided at the engine fuel pump for use in the verification of water concentrations.

During the tests, the fuel system was operated at a continuous flow rate approximately equal to the engine maximum demand. Thus, a substantial flow rate was always present in the by-pass loop, and the emulsifier was not subjected to varying conditions as the engine load changed. During steady-state operation, the flow rate of the fuel-water mixture to the float-controlled tank was equal to the rate at which the fuel was consumed by the engine, but the flow through the emulsifier loop was constant.

#### 2.1.4 Instrumentation

The documentation of engine performance using emulsified fuels required the measurement of a number of quantities during engine operation. The individual parameters for which data were recorded during each test run are listed in Table 2-2.

The dry bulb and wet bulb temperatures used for calculation of humidity were measured using mercury-in-glass thermometers. Exhaust temperatures were measured with type K thermocouples, and other temperatures were measured using type J thermocouples. All of the thermocouple readings were obtained through the use of multi-point switches and readout devices appropriate to the thermocouple calibration.

Pressures were measured using Bourdon tube gauges, mercury manometers, or water manometers as appropriate for the value and range of the metered quantity. The value of barometric pressure was obtained during each test run.

TABLE 2-2. DATA OBTAINED FROM TEST CELL

Speed Load Fuel Rate

Pressures:
Barometer
Oil
Fuel Rail
Turbocharger Boost
Exhaust
Turbine Inlet
Inlet Depression
Fuel Inlet
Air Flowmeter
Air Filter
Emulsifier
Fuel Supply
Water Supply

Emissions:
Hydrocarbons
Carbon Monoxide
Nitric Oxide
Oxides of Nitrogen
Carbon Dioxide
Oxygen
Smoke

Water Flow Rate
Water Concentration

Temperatures:
Engine Coolant
Inlet
Outlet
Oil Sump
Fuel Inlet
Fuel Mixture
Return Fuel
Intake Air
Cylinder Exhaust
Exhaust Manifold
Turbine Inlet
Compressor Outlet

Compressor Outlet
Charge Air
Water
Cell Air
Dry Bulb
Wet Bulb
Return Fuel Cooler

The flow rate of diesel fuel was continuously monitored through the use of a commercial linear mass flowmeter. However, the primary technique for determination of fuel flow was a direct mass measurement obtained using a platform balance and a stop watch. Using this technique, the time required for consumption of a known mass of fuel was recorded. The mass was adjusted in such a way that typical fuel times were on the order of two minutes, and several readings were obtained during each test run.

The water flow was monitored through the use of a variable area flowmeter installed in the water inlet line. The meter was calibrated prior to the beginning of the test program, and tables were prepared which listed the water flowmeter reading for each desired water concentration over a range of fuel rates applicable to each test point. To establish a particular water concentration in the fuel, the engine operator would read the fuel mass flowmeter, consult the table, and set the water flow rate accordingly. The water concentration was then verified by obtaining a sample of the emulsion at the engine inlet and allowing separation of the water and diesel fuel to occur.

The air flow to the engine was measured using a laminar flow element rated at 2000 cfm. The pressure drops across the flowmeter filter and across the metering element were measured using inclined water manometers, and the air flow rate was established from the meter calibration using corrections for ambient temperature and pressure.

During tests of the Detroit Diesel engine, additional air flowmetering capability was required. The 2000 cfm laminar flow element was used in the air supply to one-half of the engine (one bank of six cylinders). The air flow to the remaining engine cylinders was metered with an ASME flow nozzle installed in an inlet plenum chamber.

Instruments appropriate to diesel engine testing were used for the measurement of gaseous emissions. The concentration of unburned hydrocarbons in the exhaust stream was monitored using a heated flame ionization detector. Non-dispersive infrared analyzers were used for measurement of carbon monoxide and carbon dioxide, and a chemiluminescent analyzer was used to establish levels of nitric oxide and oxides of nitrogen. The oxygen level in the exhaust was monitored using a polarographic analyzer. Schematic diagrams of the components of the emissions instrumentation system are shown in Figures 2-5, 2-6, and 2-7, and descriptions of the individual hardware items are provided in Tables 2-3, 2-4, 2-5, and 2-6. Photographs of the instrument console are provided as Figures 2-8 and 2-9.

The exhaust smoke was measured through the use of a USPHS type opacity meter incorporated in the exhaust system at the boundary of the test cell.

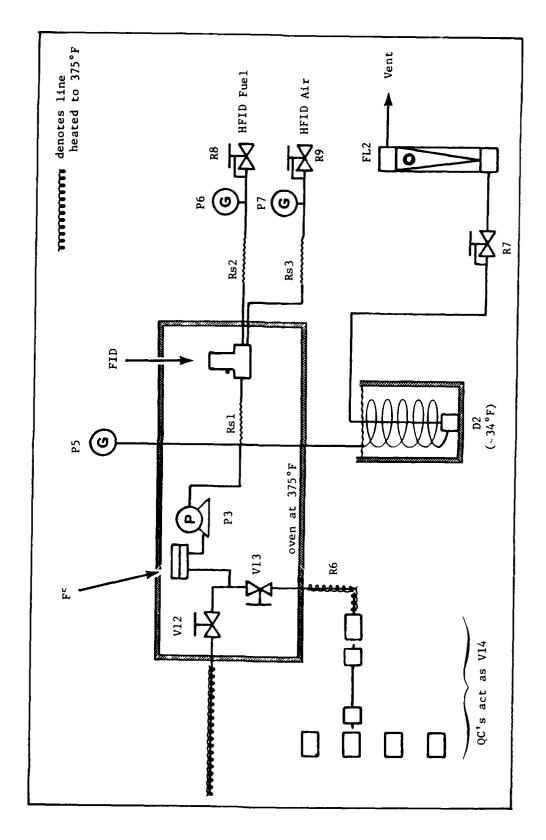


FIGURE 2-5. HYDROCARBON ANALYZER

a continue single

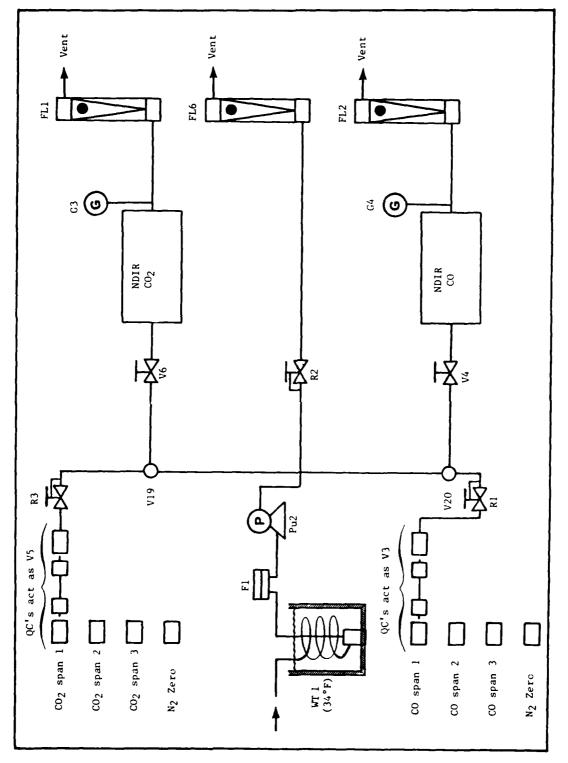


FIGURE 2-6. ANALYZER SYSTEM FOR CARBON MONOXIDE AND CARBON DIOXIDE

could of michigan

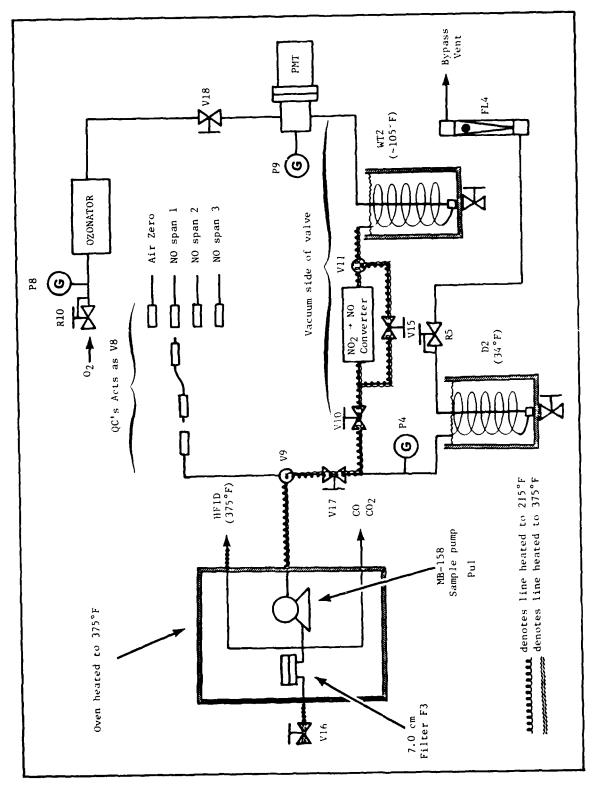


FIGURE 2-7. ANALYZER SYSTEM FOR OXIDES OF NITROGEN

TABLE 2-3. INSTRUMENTS AND RANGES ON L-4 EMISSIONS CART

Emission	Detection Method	Instrument	Range	Nominal Concentration
Carbon Monoxide (S/N AIA-23)	NDIR	Horiba OPE-15	1 2 3	0 - 1000 ppm CO 0 - 3000 ppm CO 0 - 6000 ppm CO
Carbon Dioxide (S/N 15395)	NDIR	Horiba OPE-15	1 2 3	0 - 16% CO <sub>2</sub> 0 - 6% CO <sub>2</sub> 0 - 2% CO <sub>2</sub>
Oxides of Nitrogen (S/N LOAR-9691-110)	CL	TECO 10	1 2 3	0 - 250 ppm 0 - 1000 ppm 0 - 2500 ppm
Hydrocarbons (S/N 10010)	FID	Beckman 402	1 2 3	0 - 500 ppm C 0 - 1000 ppm C 0 - 5000 ppm C
Oxygen (S/N 271-001)	Polaro- graphic	Beckman OM-11EA	1 2	0 - 25% 0 <sub>2</sub> 0 - 5% 0 <sub>2</sub>

TABLE 2-4. SWRI HEATED HYDROCARBON ANALYZER FLOW SCHEMATIC COMPONENT DESCRIPTION

Component	Description	Description of Function
Valve	V14	QC's act as span/zero selector valve
Valve	V13	Span/zero gas flow control valve
Valve	V12	Sample flow control valve
Gage	P5	Sample pressure
Gage	P6	HFID fuel pressure
Gage	<b>P</b> 7	HFID air pressure
Restrictor	Rs1	Sample capillary (Beckman)
Restrictor	Rs2	HFID fuel restrictor (Beckman)
Restrictor	Rs3	HFID air restrictor (Beckman)
Detector	HFID	Beckman 402 HFID detector
Water trap	D2	Bypass flow water trap (~34°F)
Flowmeter	FL3	Bypass flowmeter (~5 CFH)
Filter	F5	7.0 cm stainless steel flip top filter
Pump	Pu3	Metal bellows MB-158 pump
Regulator	R7	Sample backpressure regulator
Regulator	R8	HFID fuel regulator
Regulator	R9	HFID air regulator

TABLE 2-5. NDIR CO AND  $\text{CO}_2$  FLOW SCHEMATIC COMPONENT DESCRIPTION

Component	Description	Description of Function
Valve	V3	QC's act as CO selector valve V
Valve	V4	CO flow control valve
Valve	<b>V</b> 5	QC's act as CO <sub>2</sub> selector valve
Valve	V6	CO <sub>2</sub> flow control valve
Valve	V19	CO <sub>2</sub> sample/calibrate selector valve
Valve	V20	CO sample/calibrate selector valve
Gage	G3	CO <sub>2</sub> instrument pressure
Gage	G4	CO instrument pressure
Gage	P2	CO sample/span pressure
Gage	Р3	CO2 sample/span pressure
Regulator	R1	CO span/zero pressure regulator
Regulator	R2	Bypass backpressure regulator
Regulator	R3	CO <sub>2</sub> span/sero pressure regulato
Flowmeter	FL1	CO <sub>2</sub> instrument flow
Flowmeter	FL2	CO/CO <sub>2</sub> bypass flow
Flowmeter	FL6	CO instrument flow
Water trap	WT1	Water trap (34°F) for CO/CO <sub>2</sub> instrument
Filter	F1	7.0 cm stainless steel flip top filter holder
Pump	Pu2	Sample pump

TABLE 2-6. HEATED CHEMILUMINESCENT NO  $_{\mbox{\scriptsize K}}$  ANALYZER FLOW SCHEMATIC COMPONENT DESCRIPTION

Component	Description	Description of Oven
Valve	V8	QC's act as selector valve V8
Valve	V9	Sample/calibrate selector valve
Valve	V10	Sample flow control valve
Valve	V11	$NO/NO_X$ selector valve
Valve	V15	NO flow control valve
Valve	V16	System leak check valve
Valve	V17	NO/NO <sub>x</sub> total flow controller
Valve	V18	Ozone flow control valve
Gage	P4	Sample backpressure
Gage	; P8	Oxygen pressure gage
Gage	P9	Reaction chamber vacuum
Flowmeter	FL4	Sample bypass flowmeter
Regulator	R5	Sample backpressure regulator
Regulator	R10	Oxygen pressure regulator
Dryer	D2	Bypass flow water trap (~34°F)
Water Trap	WT2	NO <sub>X</sub> /NO water trap (IPA·CO <sub>2</sub> @ -105°F)
Filter	F3	7.0 stainless steel flip top filter holder
Pump	Pul	Sample pump

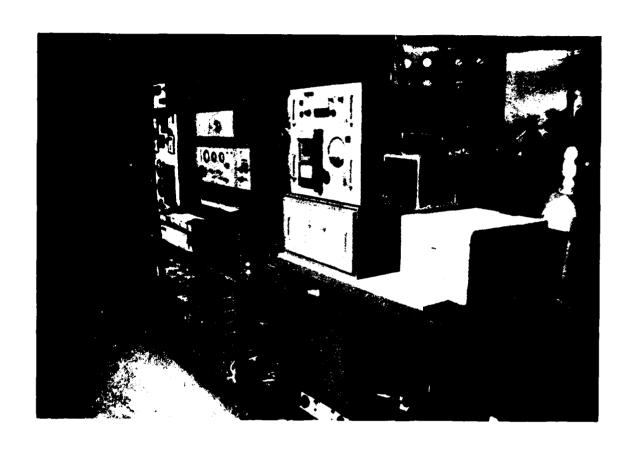


FIGURE 2-8. EMISSION INSTRUMENT CONSOLE, FRONT VIEW

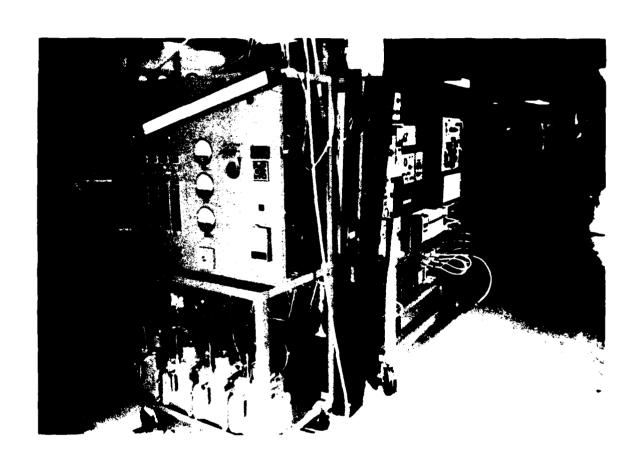
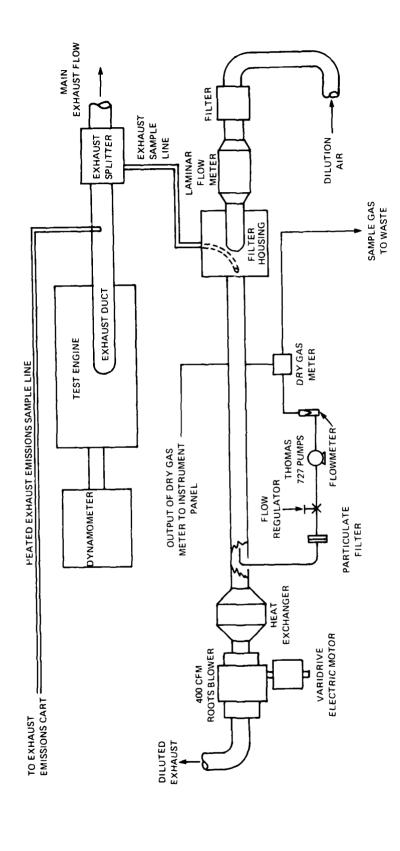


FIGURE 2-9. EMISSION INSTRUMENT CONSOLE, END VIEW

Measurements of exhaust particulate emissions were obtained during some of the tests of the Detroit Diesel engine. The primary tool utilized for this series of measurements was a dilution tunnel of the type shown in Figures 2-2 and 2-10; the dilution of the sample stream is utilized for cooling and mixing prior to the accumulation of a particulate sample. In order to obtain a sample of the exhaust, probes were located in each of the engine exhaust ducts at a point downstream from the turbocharger outlets. A regulating valve was located in each sample line, and the pressure drop across the valve was used as a means of equating the sample line flow rates. Thus, a single sample representative of both engine exhaust ducts was obtained and supplied to the particulate tunnel. The tunnel had a nominal diameter of eight inches, and air flow rates sufficient for a dilution ratio of 10 to 20 were utilized. Within the tunnel, the exhaust sample was mixed with the dilution air and cooled to 125°F. A metered sample of the diluted stream was obtained and applied to a 47 millimeter Pallflex T60A20 filter that was weighed prior to the beginning of the test. Subsequent weighing, along with the measured flow of the air stream, allowed the calculation of the particulate weight per standard cubic foot of engine exhaust. In general, only one sample filter was used during this test series; the multiple filters shown in Figure 2-10 would be utilized when more elaborate analyses of the particulate matter were required.

#### 2.2 TEST PROCEDURE

The general philosophy that governed the performance of the alternate fuel tests was closely related to the ultimate use of fuel-water emulsions on USCG cutters; thus, it was desired to obtain data that would be representative of boat operation. A sample of engine speeds and loads was obtained for one USCG cutter powered by Cummins engines, and the prop load curve for the engine was calculated. This curve is shown, along with the engine maximum output, in Figure 2-11. The specific test points for consideration during the evaluation program were selected from locations along the prop load curve.



Transfer of the second

FIGURE 2-10. PARTICULATE MEASURING SYSTEM

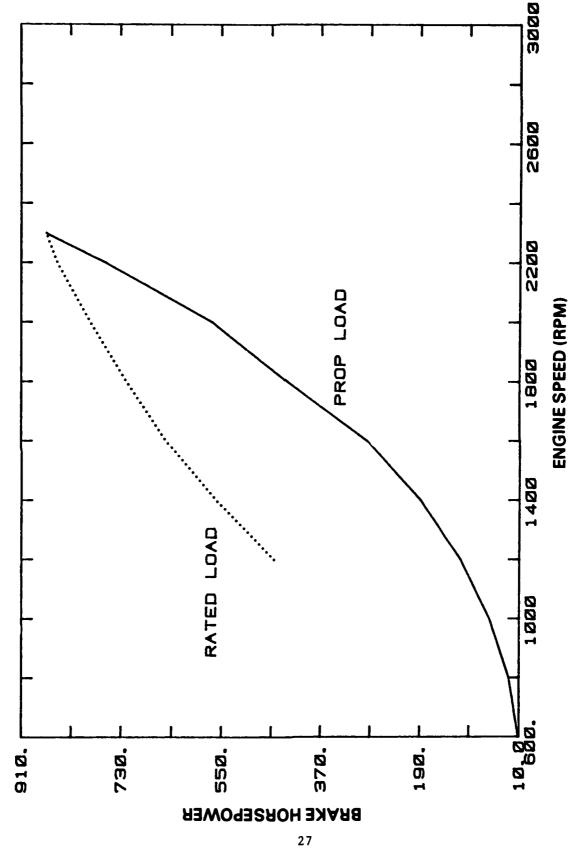


FIGURE 2-11. ENGINE SPEED AND POWER OUTPUT, MAXIMUM HORSEPOWER AND PROP LOAD, CUMMINS ENGINE

In order to establish appropriate test points for comprehensive evaluation, records representative of over 4900 hours of operation of 14 cutters powered by Cummins engines were evaluated. Figure 2-12 shows engine speed plotted against the percentage of the total operating time that was spent at each speed. It is apparent from this representation that engine speeds of 1200 and 1800 rpm are particularly important during boat operation; a fuel system designed for conservation of diesel fuel should exhibit a significant effect at these speeds in order to be effective from an overall viewpoint. The most comprehensive testing, therefore, was performed at speeds of 1200 and 1800 rpm and the prop load associated with each speed. In addition, some test data were obtained at 900 rpm and prop load.

For the tests involving the Detroit Diesel engine, the propeller load curve supplied by the manufacturer for the 16V-149TI engine was used. The loads were multiplied by 0.75 in order to account for the difference in the number of cylinders between the test engine and the marine engine; the result is shown in Figure 2-13. Tests were performed at 200 rpm increments along the prop load curve, but the comprehensive evaluation of specific points was not conducted. No data was available to evaluate the 95-ft. WPB duty-cycle with Detroit Diesel engines.

The procedure used during each test run began with a check of the instruments associated with data acquisition. In each case calibrations were performed as required. The engine was started and allowed to warm up, and then the speed and load selected for that day of testing were established by adjusting the diesel fuel-flow rate and the dynamometer controller. The initial engine operation was performed with clear No. 2 diesel fuel having specifications as shown in Appendix A, and baseline data were recorded prior to the introduction of water into the fuel system.

Upon completion of the baseline data acquisition, a water flow appropriate to a five percent concentration was initiated. The dynamometer load was adjusted to the prop load test point by adjusting the output of the engine fuel pump, and a sample of the emulsion was obtained for verification of water concentration. Data were recorded at the five percent concentration;

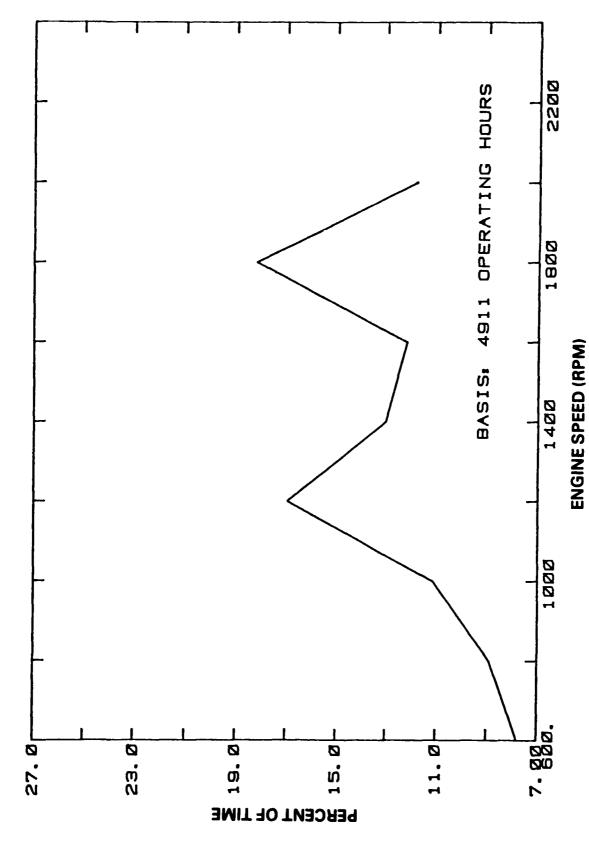


FIGURE 2-12. OPERATING TIME AS FUNCTION OF ENGINE SPEED FOR CUMMINS ENGINE-POWERED CUTTERS

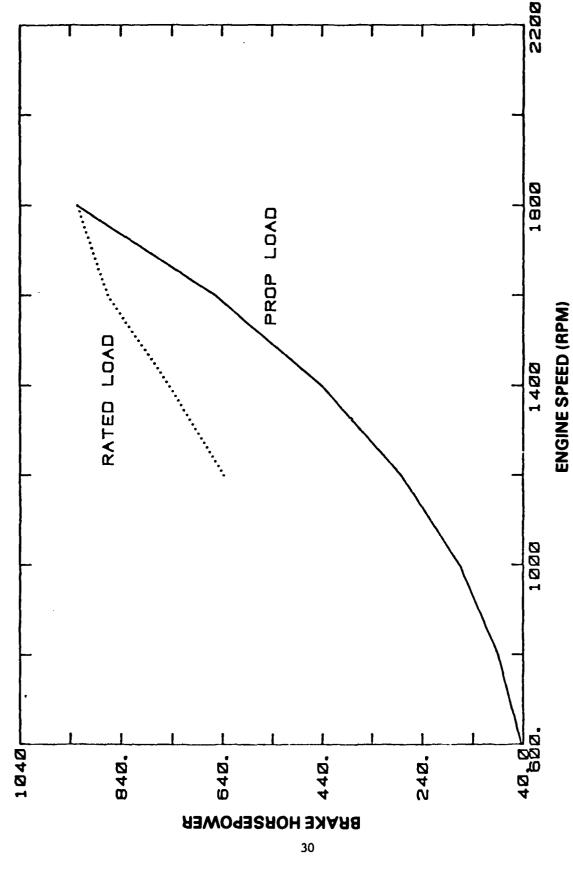


FIGURE 2-13. ENGINE SPEED AND POWER OUTPUT, MAXIMUM HORSEPOWER AND PROP LUAD, DETROIT DIESEL TWELVE-CYLINDER ENGINE

this process involved repeated measurements of the fuel rate. This sequence was then repeated at water concentrations of 10, 15, 20, and 25 percent by volume. Upon completion of the test run at the highest water concentration, the fuel system was flushed with clear diesel fuel, and the baseline test run was repeated. Subsequent days of testing involved repetition of this entire process at other speed and load conditions.

All data were recorded on a permanent record sheet, and individual values were subsequently introduced into a computer data reduction program.

## 2.3 DATA REDUCTION AND CALCULATIONS

A computer routine was utilized for the calculation of performance quantities and for the comparison of data obtained under the same operating conditions. A set of sample calculations is included in Appendix B. The sample calculations reflect the computations made by the computer program for each test run.

The basic performance quantities, such as horsepower, torque, and specific fuel consumption, were calculated using conventional relationships and constants appropriate to the specific instruments employed. These basic parameters are listed, along with measured quantities, in the tabulations of the results shown in Appendix C.

At the test points described by 1200 rpm and 1800 rpm for the Cummins engine, the test sequence over the spectrum of water concentrations was repeated several times in order to build a statistical basis for the data. Thus, a single point, such as 1200 rpm and 15 percent water, was evaluated on several test days, and three to five individual runs were performed at that point. Since each individual test run included several fuel rate measurements, the flow rate of diesel fuel specified for each run in Appendix C represents an average of several measurements. These averages for each run were then included in an overall average applicable to each test point defined by speed, load, and water concentration.

... roate-win catego

The raw data from the emission measurement procedures was interpreted through calibration curves developed for each instrument in terms of the concentration of the contaminant species in the exhaust stream. The values for each test run are reported in terms of parts per million or percent in Appendix C. During each test run for which emissions were measured, the data on exhaust emissions allowed the calculation of a carbon balance fuel-air ratio. This value was compared to the fuel-air ratio obtained by direct measurement of fuel flow rate and air flow rate. The results of the comparison for the Cummins engine are shown in Figure 2-14, which describes the error between the two values using the measured value as a standard. It should be noted that the Federal procedure for certification of diesel engines allows a tolerance of 10 percent in the comparison between calculated and measured fuel-air ratios.

During both the Cummins and GM engine tests, comprehensive data were obtained at selected test points. The average diesel fuel flow rate at each test point defined by speed, load, and water concentration was obtained from the collection of fuel rates for the individual test runs. Then, the standard deviation at each test point was calculated. As a further step, a 90 percent confidence band was calculated for each test point using the Student's t-distribution. This step allowed an indication of the significance that could be attached to the test results. The percentage change in diesel fuel flow rate was then calculated for each test point using the zero water concentration data as the baseline for each test point.

A further statistical test based upon the Student's t-distribution was applied to the fuel consumption data. For those test points where the presence of water indicated a significant change in diesel fuel consumption, a test was applied which measured the confidence with which it could be stated that the mean of the fuel consumption measurements was, in fact, different from the mean of the fuel flow measurements at zero water concentration. The results of this test allow the attachment of a numerical confidence to the statement that the presence of water in the fuel actually causes a change in the flow rate of diesel fuel.

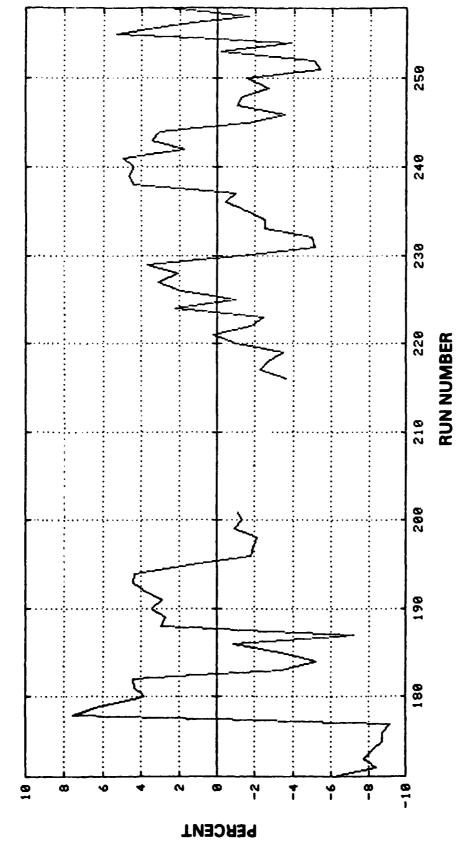


FIGURE 2-14. COMPARISON OF CARBON BALANCE AND MEASURED FUEL-AIR RATIO, CUMMINS ENGINE TESTS

# 3. RESULTS

The results of the testing program are phrased in terms of fuel consumption, smoke, gaseous emissions, and engine operating parameters. Each of these qualities are discussed in detail in the following sections.

### 3.1 FUEL CONSUMPTION

The primary thrust of the entire program was an assessment of the effect of water in the fuel on the quantity of diesel fuel consumed by the engine at a particular speed and load. The fuel consumption results, therefore, are of particular interest in the context of the overall program goals.

The fuel consumption measurements at the 1800 rpm prop load test point for the Cummins engine are shown in Figure 3-1. The solid curve is drawn through the mean values of all test runs at each water concentration, and the 90 percent confidence band is shown by broken lines. The curve indicates that the maximum effect of the presence of water was obtained at a concentration of 20 percent by volume, where the 20 percent figure represents the comparison between water volume and the total volume of liquid entering the engine. With 20 percent water present in the fuel, the diesel fuel flow rate was reduced by 1.7 percent by comparison with the flow rate of clear diesel fuel without water addition. The statistical analysis allows the statement that a 95 percent confidence exists that the means of the test runs at zero percent water and 20 percent water are actually different. Or, in other words, there is a 95 percent confidence that the sample populations at zero percent water and 20 percent water are different, and the observed change did in fact occur.

Results for the test runs at 1200 rpm and prop load are shown in Figure 3-2. The presentation format is identical to that used for the 1800 rpm tests. In this case, the minimum diesel fuel consumption occurred at a water concentration of 15 percent, and the reduction in diesel fuel flow by comparison to test runs during which no water was added was 3.3 percent. Again, a 95 percent confidence exists that the mean values of the samples

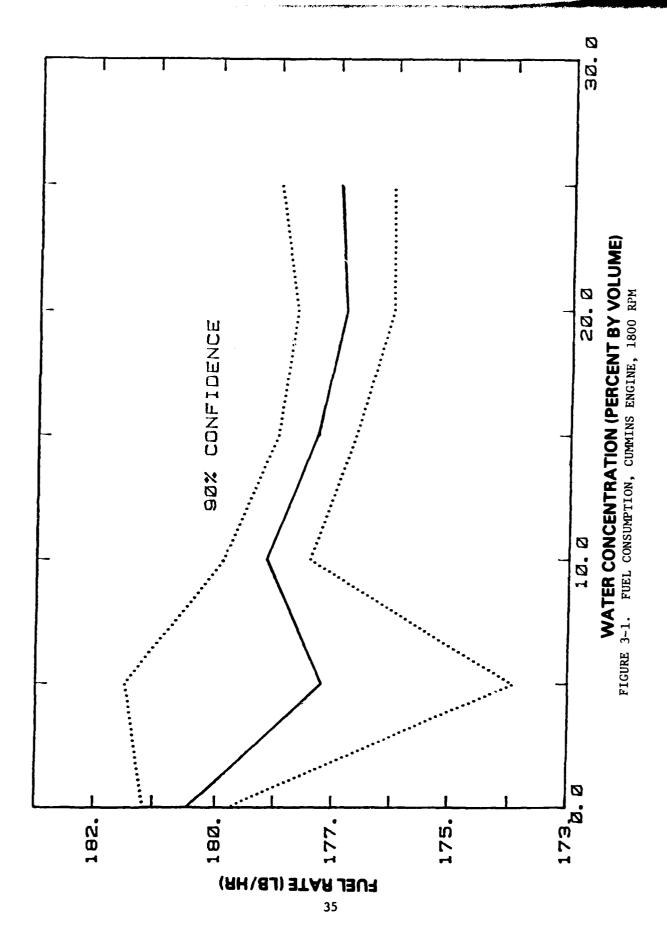


FIGURE 3-2. FUEL CONSUMPTION, CUMMINS ENGINE, 1200 RPM

obtained without water addition and with 15 percent water addition in fact represent different populations.

Some data were obtained for evaluation of the effect of water addition on fuel consumption at a speed of 900 rpm and prop load. The results are shown in Figure 3-3, using the same format as that described above. During these tests the reduction in diesel fuel flow was found to be 2.5 percent.

For the Detroit Diesel engine, fuel consumption results were obtained at several points along the prop load curve. At the 1000 rpm test point, the body of data was sufficiently extensive to allow statistical analysis; the results are shown in Figure 3-4. For this case, the general tendency was for the water to increase fuel consumption. The same trend was observed for the tests conducted at other speeds; the results are shown in Figure 3-5. No significant improvement in the rate of diesel fuel consumption could be inferred from these tests.

The configuration of the Detroit Diesel engine did allow an assessment of the effect of injection timing on the performance of water-in-fuel emulsions. Since the timing change can be effected through an injector adjustment, rather than a camshaft change, it was possible to obtain data at several values of the injection timing. Figure 3-6 describes the relationship between the fuel injector adjustment dimension and injection timing; the standard value for the engine was 2.205 inches. Tests were performed for values of the beginning of injection from about 25° BTDC to about 15° BTDC; the specific dimensions and timing angles are shown in Table 3-1. Most of the tests were performed at 1000 rpm, and examination of Figure 3-7 indicates that the timing change did not affect the relationship between fuel consumption and water addition. One series of tests was performed at 1400 rpm (Figure 3-8); the results again indicate that the timing change did not improve the ability of the engine to benefit from the addition of water to the fuel. In both Figure 3-7 and Figure 3-8, the curves designated as baseline are reproduced from Figures 3-4 and 3-5.

FIGURE 3-3. FUEL CONSUMPTION, CUMMINS ENGINE, 900 RPM

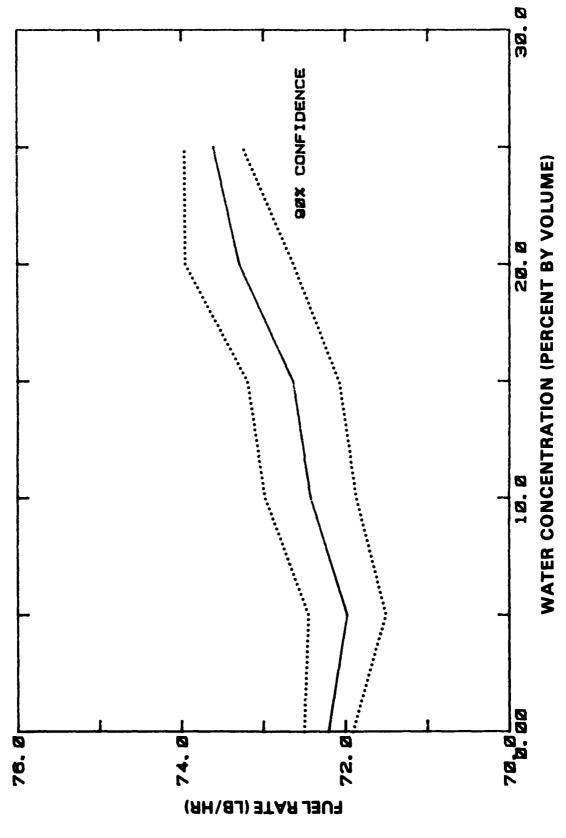


FIGURE 3-4. FUEL CONSUMPTION, DETROIT DIESEL ENGINE, 1000 RPM

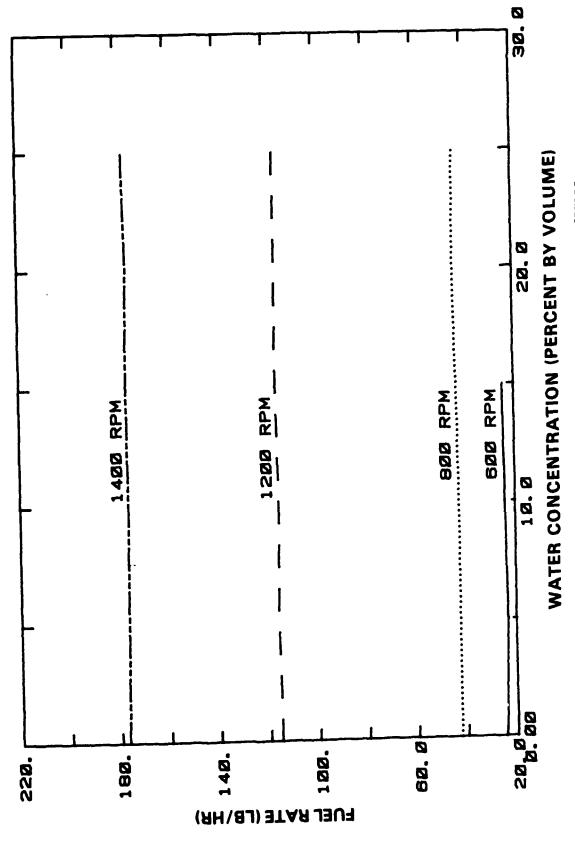


FIGURE 3-5. FUEL CONSUMPTION, DETROIT DIESEL ENGINE, FOUR SPEEDS

TABLE 3-1. DETROIT DIESEL 12V-149TI ENGINE FUEL INJECTION TIMING

Injector	Timing of
Adjustment	Injection
Dimension	Event
(inches)	(degrees)
2.165 2.185 2.205 2.223 2.235	5.5 advance 2.8 advance 0 2.4 retard 4.1 retard

- Chief State Control

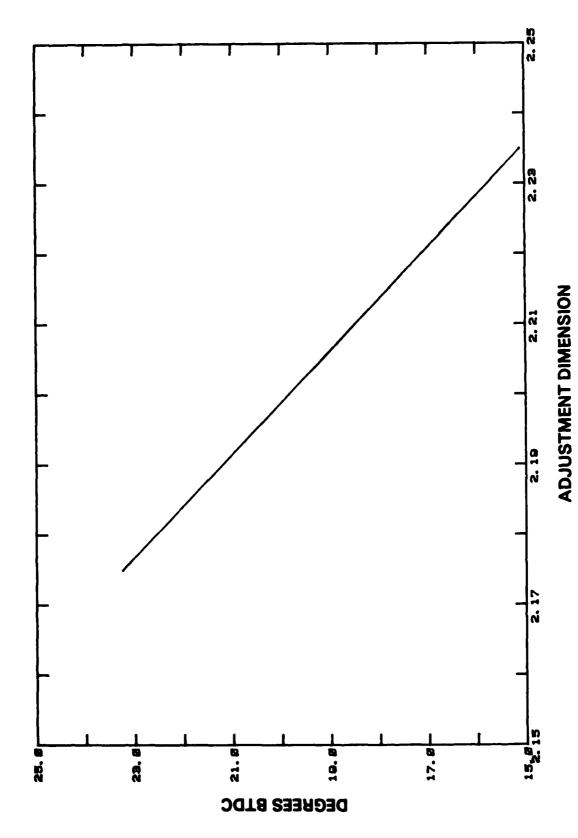


FIGURE 3-6. FUEL INJECTION TIMING AS FUNCTION OF ADJUSTMENT DIMENSION, DETROIT DIESEL ENGINE, 12V-149TI (180 INJECTORS)

42

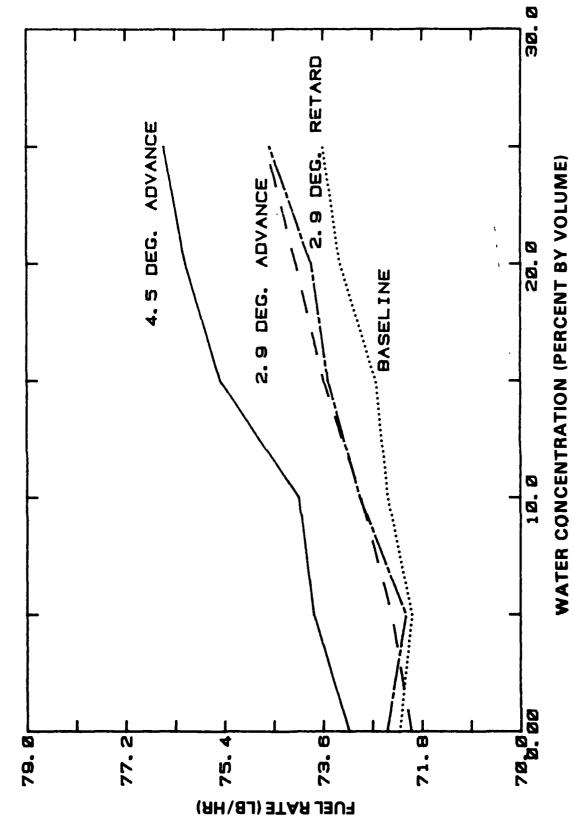


FIGURE 3-7. EFFECT OF INJECTION TIMING ON FUEL CONSUMPTION, DETROIT DIESEL ENGINE, 1000 RPM

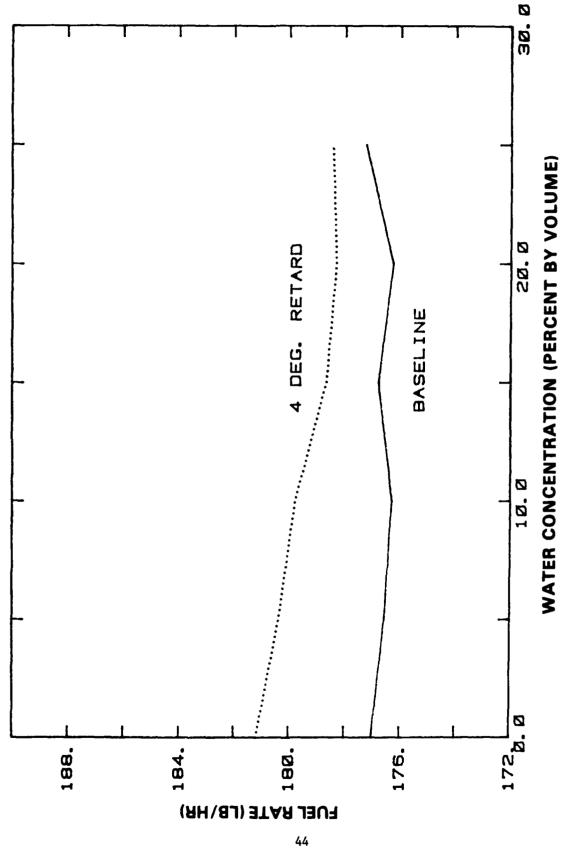


FIGURE 3-8. EFFECT OF INJECTION TIMING ON FUEL CONSUMPTION, DETROIT DIESEL ENGINE, 1400 RPM

## 3.2 EXHAUST SMOKE

During the performance of the test runs on the Cummins engine, it was observed that the presence of water in the fuel caused a significant percentage reduction in the presence of exhaust smoke. The test results are shown in Figure 3-9 for the test point at 1800 rpm, and in Figure 3-10 for the test run at 1200 rpm. In both cases, it may be observed that the smoke reduction increased as water was added to the fuel. Although the percentage reductions are dramatic, it must be noted that the opacity of the exhaust stream was quite low even without water addition. Therefore, the effect of water addition on smoke reduction is questionable from a practical viewpoint, although the magnitude of the effect is statistically significant.

#### 3.3 PARTICULATE EMISSIONS

During some of the Detroit Diesel engine tests, measurements were made of the particulate emissions using the procedures outlined in Section 2. A sample of the exhaust was obtained from each of the engine exhaust pipes, diluted with air, and passed through a pre-weighed filter. The difference in filter weights, combined with gas flow measurements, provided an assessment of the particulate loading per standard cubic foot of exhaust.

The results obtained from the particulate measurements are shown in Figures 3-11 and 3-12 as a function of both water concentration and engine speed. It may be observed from the data presented that the addition of water to the fuel has no positive effect on the particulate emissions.

## 3.4 OXIDES OF NITROGEN

The potential of water addition in terms of reduction of emissions of oxides of nitrogen from an operating engine was of particular interest at the outset of the program; other investigators have suggested that the use of water-in-fuel emulsions can provide a significant change in the emission levels of this particular contaminant.



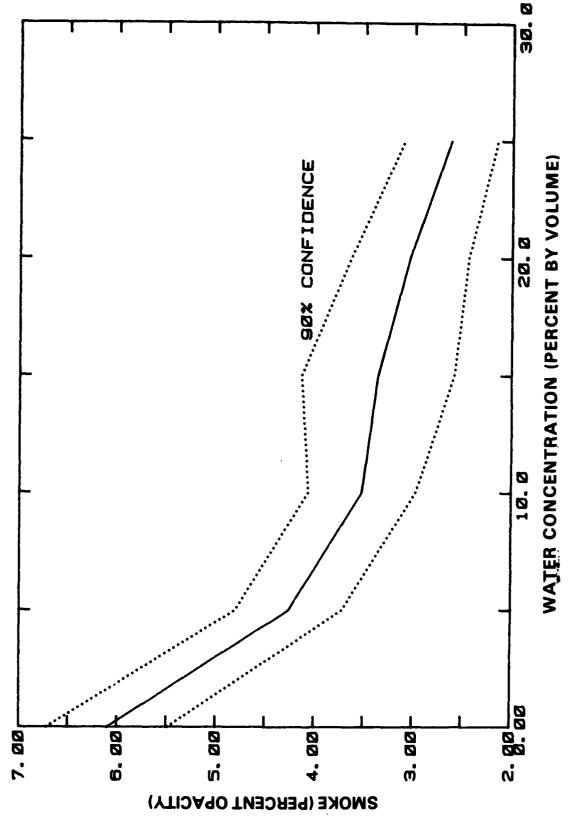


FIGURE 3-9. EXHAUST SMOKE, CUMMINS ENGINE, 1800 RPM

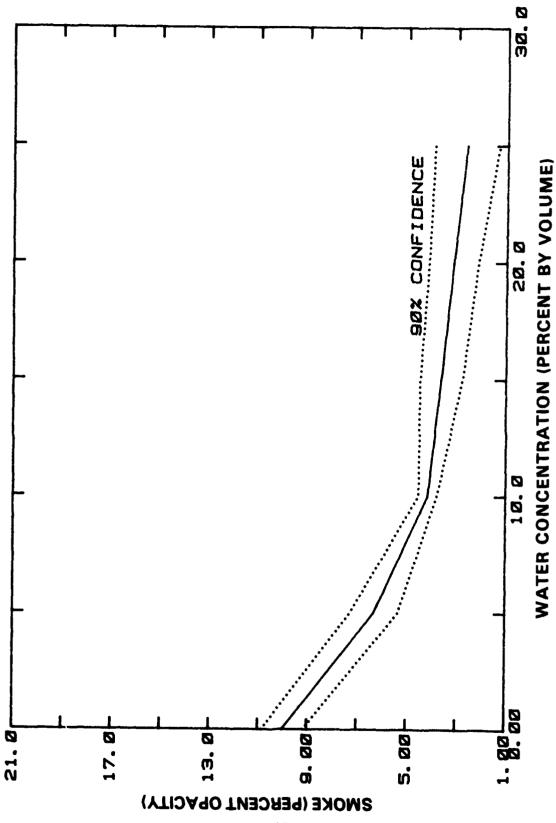


FIGURE 3-10. EXHAUST SMOKE, CUMMINS ENGINE, 1200 RPM

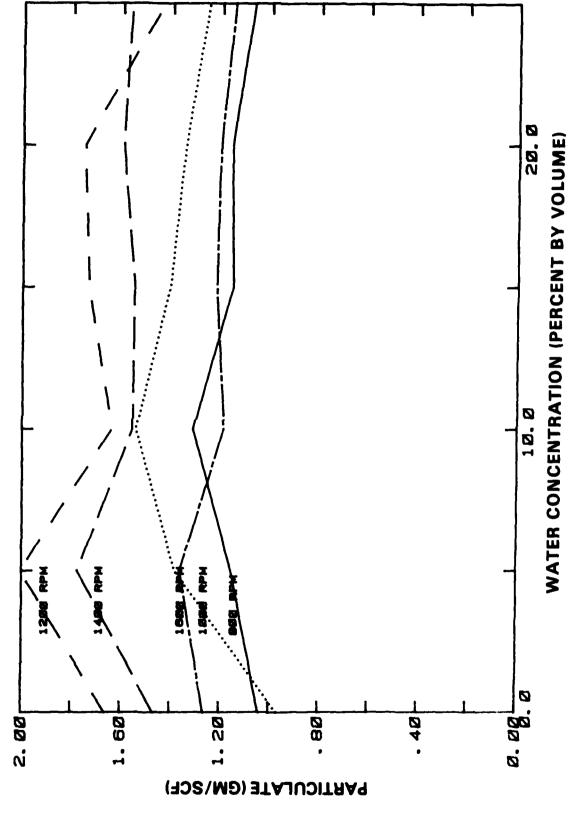


FIGURE 3-11. EXHAUST PARTICULATE EMISSIONS AS FUNCTION OF PERCENT WATER, DETROIT DIESEL ENGINE

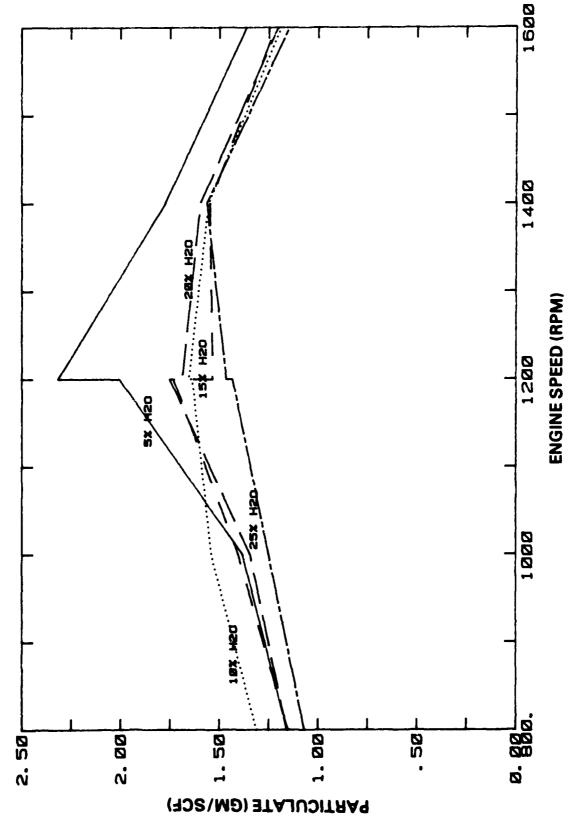


FIGURE 3-12. EXHAUST PARTICULATE EMISSIONS AS FUNCTION OF ENGINE SPEED, DETROIT DIESEL ENGINE

For the Cummins engine, the results of the testing program are summarized in Figure 3-13 for the 1800 rpm test point, and in Figure 3-14 for the 1200 rpm test point. Neither case allows the inference of a reduction in emissions of oxides of nitrogen as phrased in terms of nitric oxide concentrations. In fact, the results at 1800 rpm suggest that water addition did in fact increase the nitric oxide emissions. This result is due in part to a unique characteristic of the cummins fuel injection system. The PT system utilized on the test engine provides for the end of fuel injection at a fixed crank angle location. In other words, the fuel injection event ends at the same point during each engine cycle regardless of the quantity of fuel supplied to the injector. An increase in liquid volume, such as that obtained by the addition of water to the fuel, results in an advance of the point at which injection begins. Thus, the injection event begins earlier in the engine cycle as water is added to the fuel. The effect of injection advance on emissions of oxides of nitrogen is well documented; in general, injection advance tends to increase the emissions of this substance. It is likely, therefore, that the effect of the injection advance offset the tendency toward NO reduction afforded by the presence of water, and the result was the observed constancy or slight increase of nitric oxide levels.

Some specific experiments were performed to assess the degree of injection advance associated with the addition of water to the fuel. Through the use of a strain-gauged component in the injection linkage, a signal was obtained that allowed display of the injection event on an oscilloscope. A series of tests was performed at the 1800 rpm test point; the results are shown in Figure 3-15. The injection event is defined by the point indicated. It may be observed that as the water concentration increases the point at which injection begins tends to advance with respect to the top dead center position. From evaluation of the oscilloscope photographs, it was determined that an injection advance of approximately one degree could be associated with the addition of each five percent water added to the fuel.

Emissions of oxides of nitrogen for the tests of the Detroit Diesel engine are summarized in Figure 3-16; the effect is similar to that observed for the

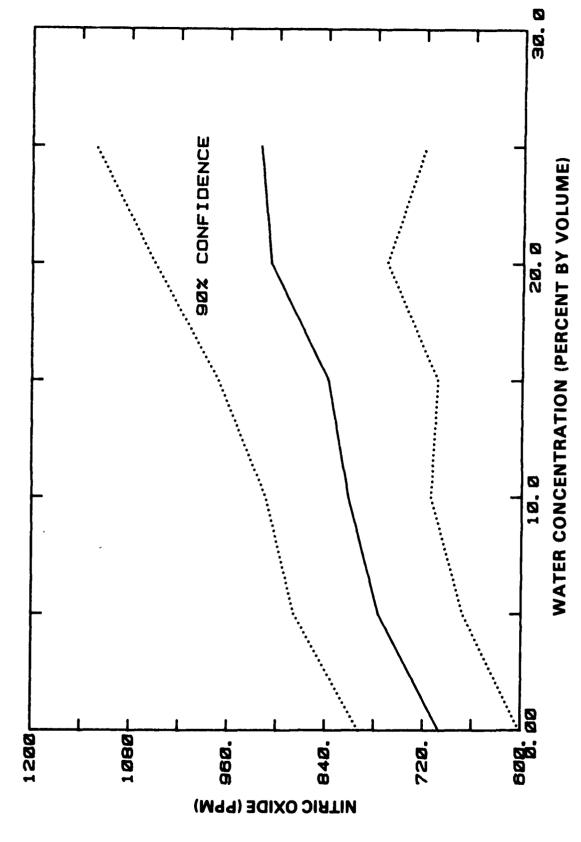
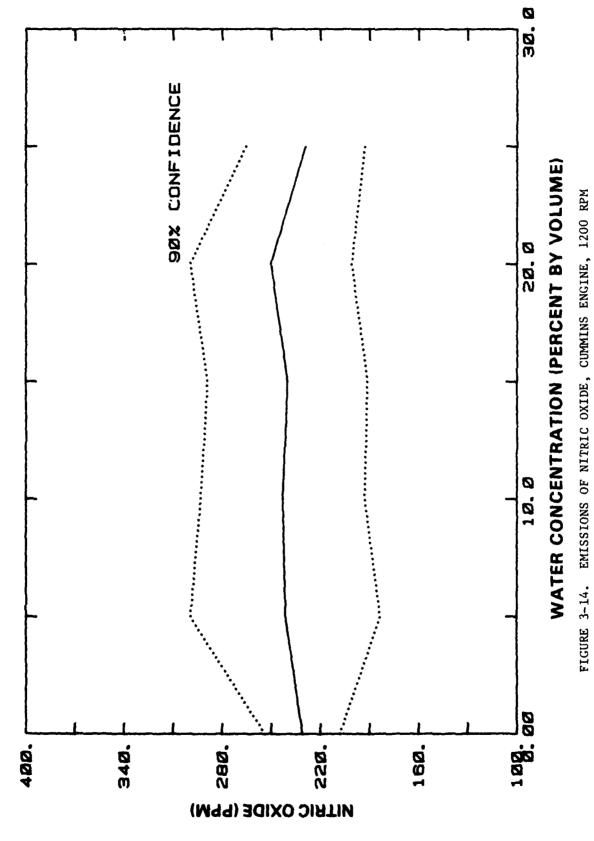
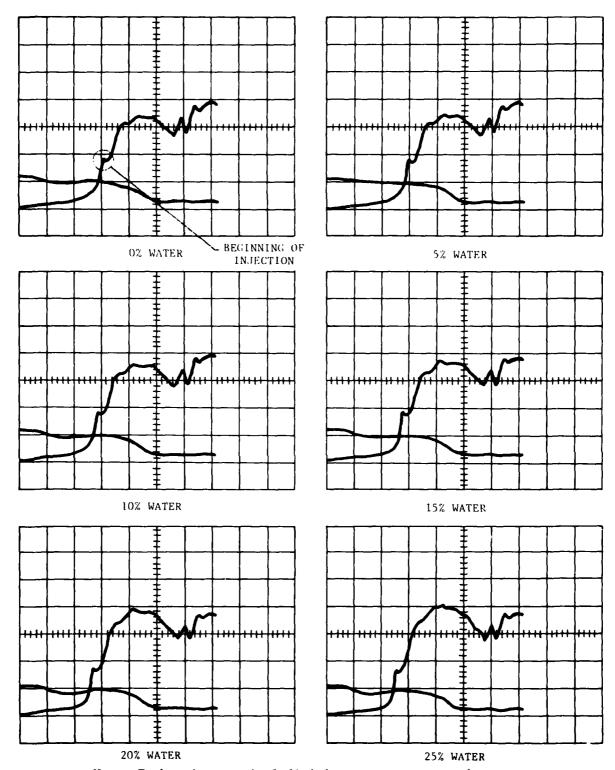


FIGURE 3-13. EMISSIONS OF NITRIC OXIDE, CUMMINS ENGINE, 1800 RPM





Note: Each major vertical division represents ten degrees. FIGURE 3-15. TIMING OF THE BEGINNING OF FUEL INJECTION, CUMMINS ENGINE

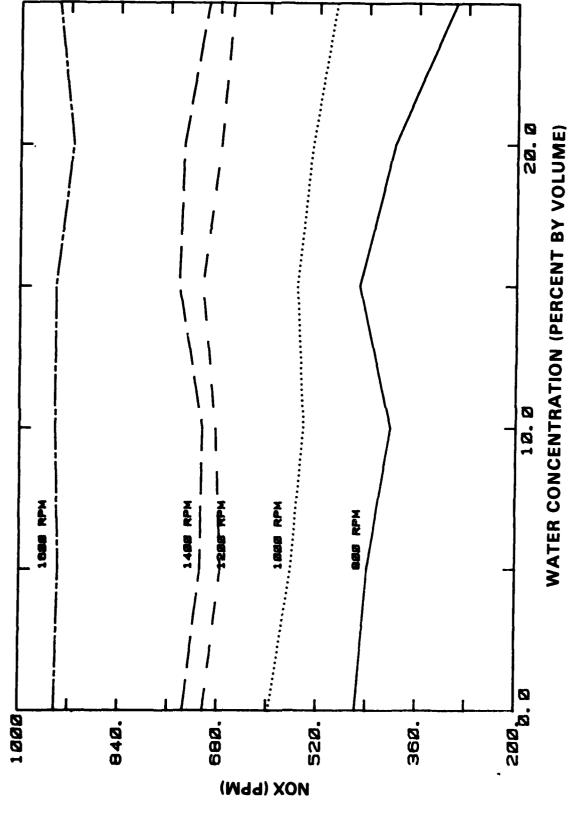


FIGURE 3-16. EMISSIONS OF OXIDES OF NITROGEN, DETROIT DIESEL ENGINE, FIVE SPEEDS

Cummins engine. The emissions increase as the load increases; this result is the usual consequence of increased cycle temperatures. Although some reductions seem to occur at low rpm (800 and 1000) and high water concentrations, in general, the addition of water does not appear to be effective for the reduction of emissions of oxides of nitrogen at any concentration examined during these tests. The explanation used for the lack of influence of water addition on emissions of oxides of nitrogen for the Cummins engine is not applicable in this case; increased liquid quantities do not affect the timing of the beginning of injection for the Detroit Diesel engine.

Two mechanisms may be postulated for the control of emissions of oxides of nitrogen through water addition. First, the water tends to absorb energy from the combustion process, and lower peak cycle temperatures might be attained. In addition, the presence of water tends to increase the ignition delay period; the net effect in this case would be a retarded combustion event. Since both cycle temperature reduction and retarded injection timing have previously been demonstrated as effective control techniques, it would appear that water addition should provide the desired results. However, the data obtained during this program indicate that, if the mechanisms described were operative, they were not sufficient in magnitude to provide effective control. In other words, at the water concentration levels employed and at the engine power levels utilized, the ignition delay increase and the cycle temperature decrease were not sufficient to cause an appreciable decrease in the emissions of oxides of nitrogen.

## 3.5 UNBURNED HYDROCARBONS

Unburned hydrocarbons are another exhaust contaminant of particular interest in engine exhaust streams. In general, it has been found that the presence of water in the fuel tends to increase the occurrence of unburned hydrocarbons in the exhaust due to a reduction in the cycle temperatures. The hydrocarbon results for the Cummins engine are shown in Figures 3-17 and 3-18 for the test points at 1800 rpm and 1200 rpm. The effect of water addition

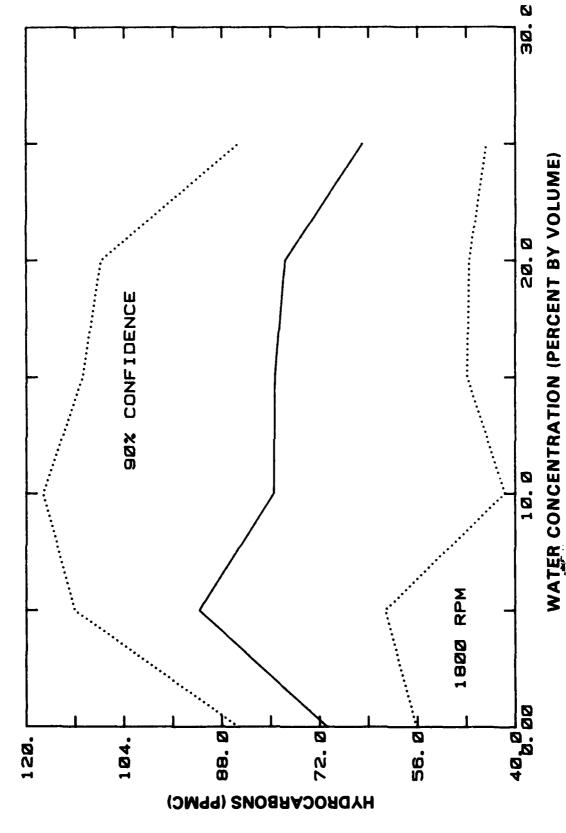


FIGURE 3-17. EMISSIONS OF UNBURNED HYDROCARBONS, CUMMINS ENGINE, 1800 RPM

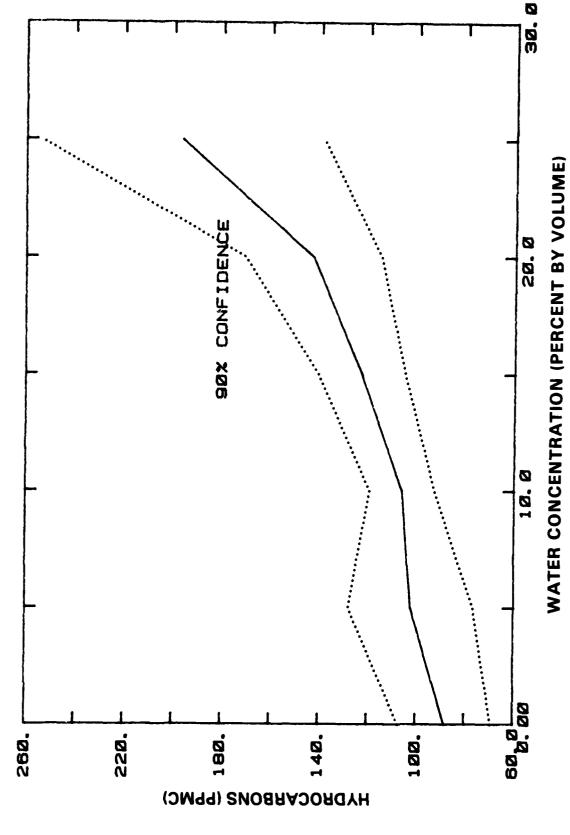


FIGURE 3-18. EMISSIONS OF UNBURNED HYDROCARBONS, CUMMINS ENGINE, 1200 RPM

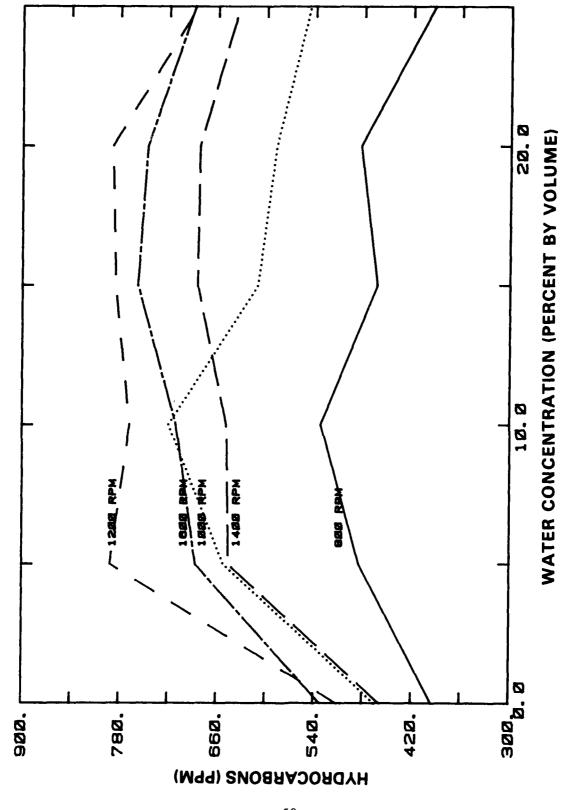
is largely inconclusive at 1800 rpm; at 1200 rpm an increase in hydrocarbons may be observed. Again, it is likely that the effect of injection advance counteracted the effect of the presence of water to some extent.

For the Detroit Diesel engine, the values of unburned hydrocarbon emissions are shown in Figure 3-19. The magnitudes of the emission levels are higher than those observed for the Cummins engine; this result is not unusual for comparisons between four-stroke cycle and two-stroke cycle engines. The presence of water in the fuel appears to yield an increase in hydrocarbon emissions; the levels associated with five percent water concentration are increased by about thirty percent compared to the baseline values. Water concentrations above five percent, however, had little additional effect on hydrocarbon emissions.

## 3.6 CARBON MONOXIDE

Carbon monoxide is a contaminant of the same character as unburned hydrocarbons in that both substances occur as a result of incomplete oxidation of the fuel. The situations that cause high hydrocarbon emissions typically cause high carbon monoxide emissions in addition. For the Cummins engine, it may be observed in Figures 3-20 and 3-21 that the character of the carbon monoxide emissions for these tests is similar to that for the emissions of unburned hydrocarbons. The results at 1800 rpm do not indicate a significant change in carbon monoxide emissions, and a moderate increase is apparent for the 1200 rpm test point at the high water concentration.

For the Detroit Diesel engine, the emissions of carbon monoxide are described in Figure 3-22. The results for 1200 rpm suggest a small influence of water concentration on carbon monoxide emissions. At speeds below 1200 rpm, an increasing tendency toward higher emissions may be observed, while an increasing tendency toward emission reduction can be associated with speeds above 1200 rpm. An explanation of this trend may be postulated in terms of the effect of water addition on the ignition delay of the fuel and on the mixing of fuel and air within the cylinder. At the low speeds, the effect of the



EMISSIONS OF UNBURNED HYDROCARBONS, DETROIT DIESEL ENGINE, FIGURE 3-19. FIVE SPEEDS

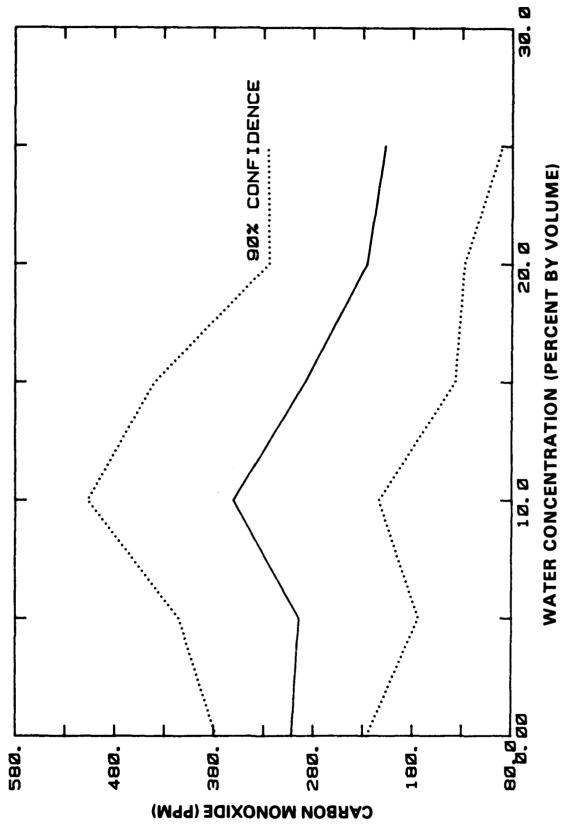


FIGURE 3-20. EMISSIONS OF CARBON MONOXIDE, CUMMINS ENGINE, 1800 RPM



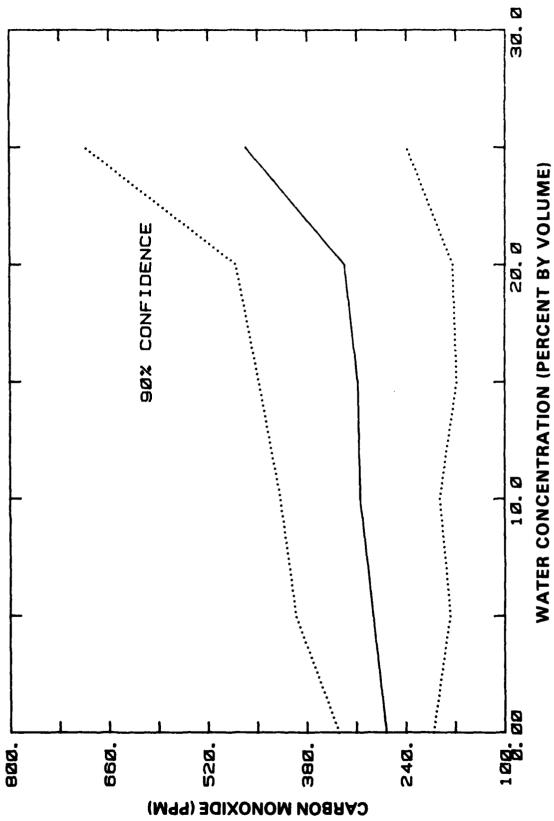


FIGURE 3-21. EMISSIONS OF CARBON MONOXIDE, CUMMINS ENGINE, 1200 RPM

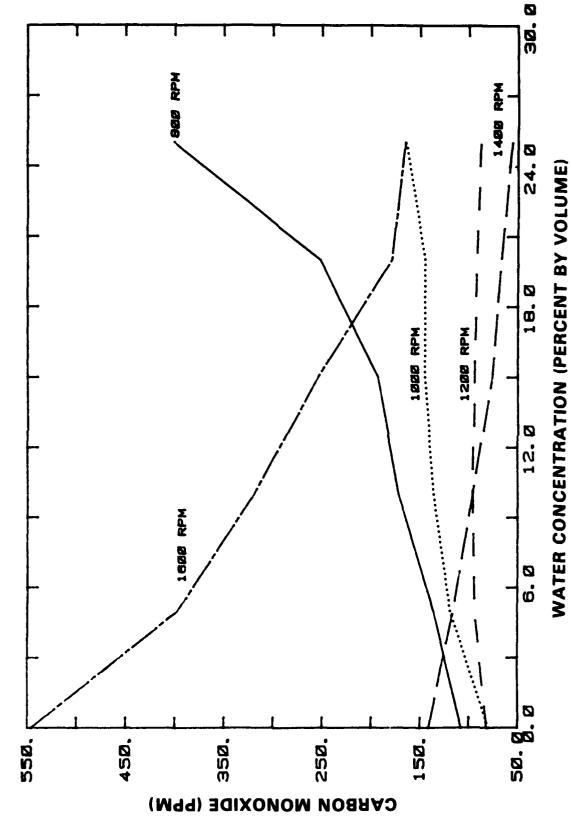


FIGURE 3-22. EMISSIONS OF CARBON MONOXIDE, DETROIT DIESEL ENGINE, FIVE SPEEDS

addition of water on the ignition delay might be sufficient to cause increased emissions of carbon monoxide, although no effect on emissions of hydrocarbons and oxides of nitrogen was discernible.

It may be more appropriate to examine the carbon monoxide emissions from the Detroit Diesel engine in the context of mixing within the engine cylinder. During the 800 rpm tests, the fuel rate was quite small at the prop load condition. The addition of an inert component to the fuel stream would tend to diversify the jet of injected fuel with respect to the interior of the cylinder; the local fuel-air ratio in the vicinity of a fuel droplet would tend to become leaner. Since successful combustion depends upon ignition at points within the chamber and subsequent mixing of burning and unburned materials, it is possible that the addition of water allowed portions of the charge to escape complete inflammation. At the higher fuel rates, this effect of water addition would be reduced, and the effect of water addition on carbon monoxide emissions would be reduced. This argument does not explain the high carbon monoxide levels at the 1600 rpm test point; the baseline carbon monoxide emissions at that point seem uncharacteristically high. Since the fuel-air ratio at this point is well within customary limits for good combustion, poor mixing of air and fuel could be the cause of poor combustion. It is possible that the injection of an increased volume of liquid allowed improved penetration of the fuel injection jet, and increased mixing caused a reduction in carbon monoxide levels to values typical of lower speeds.

#### 3.7 CARBON DIOXIDE AND OXYGEN

The emissions of carbon dioxide and oxygen are recorded in the test data shown in Appendix C. These substances, although not regulated contaminants, are of interest in the generalized context of engine testing. The carbon dioxide measurement is particularly important to carbon balance fuel-air ratio calculations, and results for these estimates have been presented in Figure 2-14.

# 4. SUMMARY AND CONCLUSIONS

During the testing program described by this document, Cummins and Detroit Diesel engines representative of USCG main propulsion units were operated under loading conditions typical of marine service using water-in-fuel emulsions having various concentrations. Measurements of engine performance and emissions were obtained in an effort to define optimum points for further exploitation of the benefits of emulsified fuels.

The engines were evaluated on a laboratory test bed that included a dynamometer capable of absorbing the maximum engine output. Instrumentation was provided to allow measurement of speed, load, and pertinent temperatures and pressures throughout the installation. Fuel was supplied through a system capable of blending water with diesel fuel in amounts up to 25 percent of the total volume of liquid supplied to the engine; fuel not used by the engine was cooled and recycled through the blending system. No surfactants or stabilizers were employed. A Gaulin "Hydroshear" emulsifier was used to accomplish the mixing of fuel and water, and visual observation of samples obtained from various points suggested that separation of fuel and water did not occur within the fuel system. The fuel consumption measurements were performed using a direct weight method.

Measurements of gaseous exhaust emissions were obtained for both engines using instruments appropriate to the type and level of the individual contaminant substances. In addition, a dilution tunnel was used to measure particulate emissions from the Detroit Diesel engine, and exhaust smoke measurements were obtained for the Cummins engine.

Data from cutter logs were used to define the prop load test points of particular interest for the Cummins engine, and tests were repeated several times for those points in order that a statistical basis for the results could be constructed. For the Detroit Diesel engine, tests were performed at more points along the prop load curve, but the repetition of tests was less extensive.

The fuel consumption tests for the Cummins engine suggested that diesel fuel savings averaging two to three percent could be obtained using emulsion concentrations of fifteen to twenty percent water. No significant fuel saving could be associated with the use of emulsions in the Detroit Diesel engine. Since the laboratory test conditions were generally more favorable than those that would prevail in actual marine use, it is necessary to conclude that the use of water-in-fuel emulsions would not be beneficial to USCG operations.

Measurements of exhaust smoke were performed for the Cummins engine, and particulate emissions were measured for the Detroit Diesel engine. Although dramatic reductions in exhaust plume opacity were observed, the smoke levels for engine operation without water addition were not excessive. Thus, although the data suggest that water-in-fuel emulsions could be used for smoke control, the observation of excessive smoke at any operating point other than full rated load is probably indicative of defective engine components or poor adjustment of engine systems, and smoke control should be effected through correction of those conditions. The addition of water to the fuel did not have a significant effect on the emission of exhaust particulates, although the Detroit Diesel engine was generally insensitive to the presence of water at all test points.

In terms of gaseous exhaust emissions, the expected effects of water addition were not generally observed. The addition of water to the fuel should yield an increase in the emissions of oxides of nitrogen. Although some trends toward these effects could be observed in the test results, no definitive conclusions can be drawn concerning the effect of water addition on emissions.

From a theoretical viewpoint, the addition of water to diesel fuel can result in a mixture which would exhibit unique properties at the onset of combustion. Specifically, it is believed that the vaporization of the water phase causes a "micro-explosion" that is capable of shattering a fuel droplet; the result of this process would be improved mixing of fuel and air and enhanced combustion quality. In addition to improving combustion in a diesel engine, the presence of water in the fuel should lower combustion temperatures, and

emissions of unburned hydrocarbons and carbon monoxide should increase. Also, the lower temperature and an increase in the ignition delay period should reduce the emissions of oxides of nitrogen. The micro-explosion phenomenon has been demonstrated for burning of single droplets, and the addition of water to fuel has been suggested for application to a wide range of combustion processes. 2,3,4,5,6

Several investigators have obtained encouraging test results using water-in-fuel emulsions in diesel engines; 1,6,7,8 others have been less successful in demonstrating benefits associated with the emulsion use. 9,10 The basic engine configuration apparently affects the results; four-stroke cycle engines have generally produced more noticeable effects.

During the present study, all of the observations were macroscopic in nature, and no attempt was made to observe the details of emulsion quality or the micro-explosion event. However, the tests were performed in such a way that the effects of emulsions on engine performance would be revealed.

For the four-stroke cycle (Cummins) engine, the observed reductions in fuel consumption and exhaust stream opacity are indicative of improved mixing between the fuel spray and the air charge within the combustion chamber. The results for unburned hydrocarbons and carbon monoxide were inconclusive, and the emissions of oxides of nitrogen were probably affected by the change in injection timing due to increased fuel quantity. Thus, some evidence can be associated with support of the micro-explosion theory, although the effects were not sufficiently large to be of practical interest.

In the case of the two-stroke cycle (Detroit Diesel) engine, no improvement in fuel consumption or particulate emissions could be observed. Some indication of mixing quality could be inferred from the data on carbon monoxide emissions, but the presumed micro-explosion effects could not be separated from alterations of the fuel spray due to increased injection quantity. In general, the two-stroke cycle engine was quite tolerant of, and insensitive to, the addition of water to the fuel. This result is consistent with the observations of other investigators; for example, some observers have found that water concentrations exceeding 40 percent were necessary to obtain

significant changes in engine performance. 11 Such water concentrations lie beyond the range of practical interest for USCG operations.

Both the data obtained during this study and the results reported by other investigators indicate that the effect of water-in-fuel emulsions on engine performance is dependent upon the engine system configuration. Although inferences can be drawn from the body of accumulated information, it is not possible, as yet, to predict the response of an untested engine to the addition of water to the fuel. Additional information must be obtained to define the specific mechanisms which are operative and the effect that these mechanisms exert on the combustion process.

It is possible that further investigation would reveal significant differences between techniques for the production of water-in-fuel emulsions, both in the microstructure of the emulsion product and in the effect on engine operation. Aside from the assurance of a stability sufficient for transit through the fuel system, this study did not address the details of emulsion production. An investigation of the effects of different production techniques, if attempted, should be closely coupled with a study designed to reveal the dominant mechanisms of combustion process control.

# RECOMMENDATIONS

On the basis of the results obtained during this study, it is recommended that no future effort be directed toward the use of water-in-fuel emulsion in 85 foot and 95 foot WPB class cutters. Although a small fuel consumption benefit was observed for the Cummins engine in a laboratory environment, it is unlikely that a benefit of 2-3 percent could be translated to field use.

The use of water-in-fuel emulsions for smoke control is possible for the Cummins engine, but emulsion use should not be pursued as a control strategy. Excessive smoke at conditions other than engine rated speed and load is indicative of a hardware failure or poor control system adjustment, and correction of the defect is indicated in preference to an auxiliary control strategy.

The data obtained on the Cummins and Detroit Diesel engines do not preclude the successful use of emulsions in other USCG power units. If emulsions are candidates for application to, for example, larger medium-speed diesel engines, then specific tests should be conducted to determine the response of those engines to the presence of water in the fuel system.

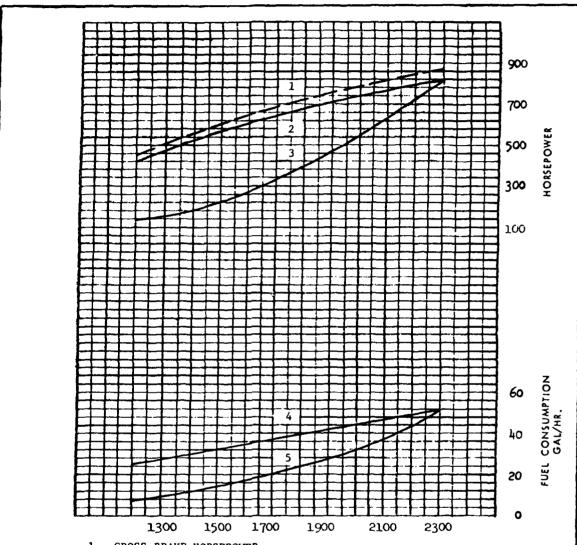
Additional basic work will be required in order to formulate general statements concerning the necessary properties of emulsions and the effect of those emulsions on engine performance. While the execution of this basic work is beyond the scope of USCG interests, it is recommended that performance of the basic research efforts be encouraged. Until the general results for emulsion use are understood and documented, tests of specific enginemulsification system combinations will be necessary for the assessment of practicality.

APPENDIX A FUEL PROPERTIES AND ENGINE DATA

TABLE A-1. FUEL ANALYSIS DATA

<del></del>			·
PROPERTY	CUMMINS TESTS RUNS 171-282	DETROIT DIESEL TESTS RUNS 1-106	DETROIT DIESEL TESTS RUNS 107-150
Heat of Combustion (BTU/LB)	20,038	20,050	19,400
Hydrogen (% by Weight)	12.83	12.92	13.52
Carbon (% by Weight)	85.75	84.71	84.31
Oxygen (% by Weight)	2.37	1.60	0.96
Nitrogen (% by Weight)	0.68	0.77	0.48
Sulfur (% by Weight	0.18	0.12	0.075
API Gravity at 60°F	35.3	33.9	35.1
Reid Vapor Pressure	1.78	2.60	0.31
Cetane Number (Calc.)	47.5	49.7	50.6
Flash Point (°F)	147	163	162
Viscosity (centistokes)			
50°F 100°F 150°F 210°F	4.91 2.59 1.61 1.07	7.30 3.55 2.18 1.35	5.51 3.14 1.95 1.26
Distillation			
IBP (°F) 10% (°F) 20% (°F) 30% (°F) 40% (°F) 50% (°F) 60% (°F) 70% (°F) 80% (°F) 90% (°F) 95% (°F) EP (°F) Recovery (%) Residue (%) Loss (%)	360 420 446 464 481 499 514 533 552 581 606 630 98.5 1.5 0.0	387 463 489 505 520 534 549 568 595 640 682 714 96.75 1.25 2.00	385 446 473 482 507 522 536 554 579 621 657 700 98 2

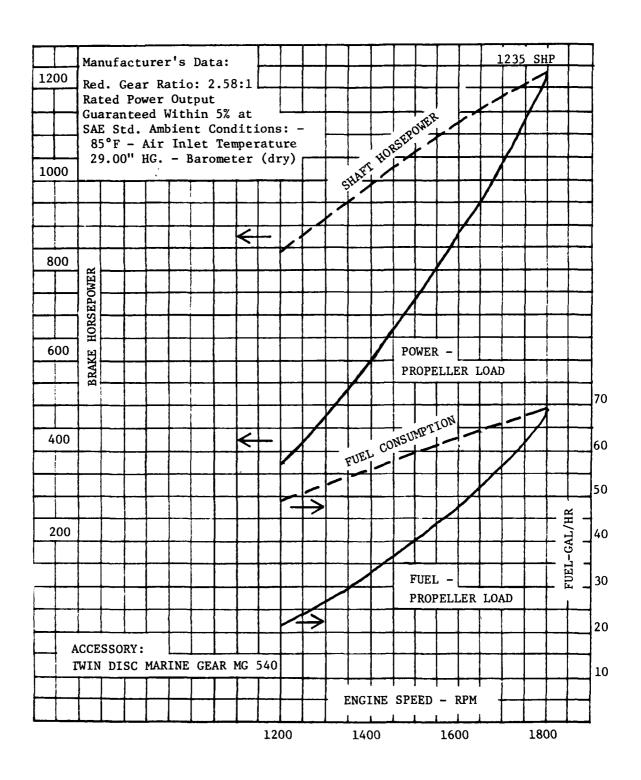
TABLE A-2. MARINE ENGINE PERFORMANCE CURVE



- 1. GROSS BRAKE HORSEPOWER.
- NET HORSEPOWER WITH REVERSE REDUCTION GEAR, GENERATOR AND RAW WATER PUMP.
- 3. HYPOTHETICAL PROPELLER POWER CURVE (3.0 EXPONENT).
- 4. FUEL CONSUMPTION FOR NET SHAFT HORSEPOWER.
- 5. FUEL CONSUMPTION FOR HYPOTHETICAL PROPELLER.

The above curves are based on 500 ft. altitude (29.38" HG.) and 85°F intake air temperature; fuel consumption curves are based on fuel weight of 7.0 lb/US gal. Manufacturer's data for Model VT12-900M engine (turbocharged-aftercooled, 12 cylinders, 1710 cu. in. displacement, with 5-1/2 in. bore and 6 in. stroke, military version).

TABLE A-3. ESTIMATED PERFORMANCE SERIES V-149TI MARINE 16V-149TI CREW BOAT, JACKET WATER INTERCOOLER, 150 INJECTORS



# APPENDIX B SAMPLE CALCULATIONS

During each individual test run, engine data were entered on a permanent record sheet. The data items that were recorded are listed in Table B-1 along with the numerical values associated with run number 235 for the Cummins engine; the sample calculations which follow will be based upon the numerical values shown.

The differences between the data items recorded for the Cummins and Detroit Diesel engines were minor. The Detroit Diesel engine was equipped with four turbochargers; therefore, the number of turbocharger-related temperatures and pressures was doubled by comparison with the Cummins engine. Also, air box pressure, rather than fuel rail pressure, was recorded for the Detroit Diesel engine.

Recorded engine test data were entered into a computer program, and several calculation routines were executed. The following discussion describes the details of the calculation procedure, and the numerical values for Cummins run number 235 are presented as an example.

### **Humidity Calculations**

The air supplied to the engine contained some moisture, and the further addition of water to the fuel affected the exhaust moisture. The following equation was used for the calculation of the saturation vapor pressure of water:  $^{12}$ 

$$P_{B} = \exp \left[ B \ln T + \sum_{i=0}^{9} F_{i} T^{i-2} \right], \qquad (1)$$

CHARLEST CARRESTS THE

where  $P_R$  = saturation vapor pressure, pascals

T = temperature, °K

B = -12.150799

 $F_0 = -8.49922 \times 10^3$ 

 $F_1 = -7.4231865 \times 10^3$ 

 $F_2 = 96.1635147$ 

TABLE B-1. TEST DATA

Date: 24 July 1979	Value for Cummins	
Data Item	Run 235	
	1	
Dynamometer Constant	-	3000
Nominal Water Concentration	Percent	20
Barometric Pressure	Inches - Hg	29.03
Wet Bulb Temperature	°F	77
Dry Bulb Temperature	°F	87
Engine Hour Meter	Hours	5222.3
Engine Speed	RPM	1800
Ream Load	Pounds	718
Coolant Inlet Temperature	°F	175
Coolant Outlet Temperature	°F	186
Oil Sump Temperature	°F	219
Fuel Temperature at Emulsifier	°F	112
Diesel Fuel Inlet Temperature	°F	95
Return Fuel Temperature	°F	151
Return Fuel Temperature After Cooler	°F	113
Intake Air Temperature	°F	90
Exhaust Stack Temperature (Left)	°F	762
Exhaust Stack Temperature (Right)	°F	777
Turbine Inlet Temperature (Left)	°F	905
Turbine Inlet Temperature (Right)	°F	917
Compressor Outlet Temperature (Left)	°F	224
Compressor Outlet Temperature (Right)	°F	222
Charge Air Temperature (Left)	°F	189
Charge Air Temperature (Right)	°F	189
Tap Water Inlet Temperature	°F	101
Cell Air Temperature	°F	91
Engine Oil Pressure	psi	76
Fuel Rail Pressure	psi	96

TABLE B-1. TEST DATA, continued

Data Item	Units	Value For Cummins Run 235
Boost Pressure (Right)	psi	9.9
Boost Pressure (Left)	psi	10.0
Turbine Inlet Pressure (Left)	psi	9.0
Turbine Inlet Pressure (Right)	psi	10.0
Inlet Vacuum	In. H <sub>2</sub> O	13.9
Exhaust Pressure (Right)	Inches - Hg	0.2
Exhaust Pressure (Left)	Inches - Hg	0.5
Pressure Drop, LFE Filter	In. H <sub>2</sub> O	5.40
Pressure Drop, Laminar Flow Element	In. H <sub>2</sub> O	4.25
Exhaust Temperature, Cylinder 1R	°F	905
Exhaust Temperature, Cylinder 2R	°F	890
Exhaust Temperature, Cylinder 3R	°F	897
Exhaust Temperature, Cylinder 4R	°F	873
Exhaust Temperature, Cylinder 5R	°F	890
Exhaust Temperature, Cylinder 6R	°F	898
Exhaust Temperature, Cylinder 1L	°F	939
Exhaust Temperature, Cylinder 2L	°F	913
Exhaust Temperature, Cylinder 3L	°F	880
Exhaust Temperature, Cylinder 4L	° <sub>F</sub>	882
Exhaust Temperature, Cylinder 5L	°F	892
Exhaust Temperature, Cylinder 6L	° <sub>F</sub>	904
Water Flowmeter 1, Glass Float	mm	150+
Water Flowmeter 2, SS Float	mm	115
Water Flowmeter 3, SS Float	mm	0
Fuel Pressure, Tank	psi	20
Pressure, Emulsifier Inlet	psi	100
Pressure, Fuel at Engine	psi	1.6
Water Supply Pressure	psi	65
Emission Concentrations		
Hydrocarbons	ppmc	56
Carbon Monoxide	ррш	148

TABLE B-1. TEST DATA, continued

Data Item	Units	Value For Cummins Run 235
Nitric Oxide	ppm	773
Oxides of Nitrogen	ppm	770
Carbon Dioxide	Percent	7.8
Oxygen	Percent	12.8
Smoke	Percent	3.3
Fuel Flow Measurements	Pounds	5.0
Times: 1	sec	102.3
2	sec	102.6
3	sec	102.4
4	sec	102.7

 $F_3$  = 2.4917646 x 10<sup>-2</sup>  $F_4$  = -1.3160119 x 10<sup>-5</sup>  $F_5$  = -1.1460454 x 10<sup>-8</sup>  $F_6$  = 2.1701289 x 10<sup>-11</sup>  $F_7$  = -3.610258 x 10<sup>-15</sup>  $F_8$  = 3.8504519 x 10<sup>-18</sup>  $F_9$  = -1.4317 x 10<sup>-21</sup>.

Application of this equation to the dry and wet bulb temperatures for run 235 yields the following:

$$P_{WB}$$
 = 3168.62 pascals (at 298.15°K)  
 $P_{DR}$  = 4382.41 pascals (at 303.71°K).

The vapor pressure at the wet bulb temperature was obtained from "Ferrels equation",

$$P_V = P_{WB} - 0.000660 (T_{DB} - T_{WB}) P_{BARO} [1 + 0.0015 (T_{WB} - 273.15)],$$
 (2)

where  $P_V$  = vapor pressure, pascals

 $T_{DB}$  = dry bulb temperature,  ${}^{\circ}K$ 

TWB = wet bulb temperature, °K

 $P_{BARO}$  = barometric pressure, 98307.2 pascals.

Using this relationship, the vapor pressure was found to be

$$P_V = 2797.50 \text{ pascals}.$$

The relative humidity, by definition, was calculated as:

RH = 
$$\frac{P_V}{P_{DR}} \times 100 = 63.8\%$$
, (3)

and the specific humidity was calculated from:

$$H = \frac{(K)(^{P}V)}{P_{BARO} - P_{V}}, \qquad (4)$$

where H = specific humidity, gm  $H_2O/gm$  dry air K = 0.6220 gm  $H_2O/gm$  dry air,

for the test case,

 $H = 0.0182 \text{ gm H}_2\text{O/gm dry air}$ (or pounds moisture/pound dry air).

The volume concentration of the water vapor was calculated on a dry basis as:

$$Y = \frac{(H)(MAIR)}{MH_{2}O} = 0.0293,$$
 (5)

where Y = water vapor volume concentration

MAIR = molecular weight of air = 28.9645

 $M_{H_2O}$  = molecular weight of water = 18.01534/

#### Water Flow Rate Calculations

Although a nominal water concentration was associated with each test run, the actual flow to the engine fuel system was measured using variable area flowmeters. The flowmeters were calibrated with water prior to the initiation of testing, and a fifth order curve was fitted to the calibration data. Two flowmeters were utilized, and one of the two units contained two floats; thus three calibrated flow ranges were available. The matrix of flowmeter constants is shown in Table B-2. The water flow determination was based upon the flow range most applicable to the particular test run; the procedure was

WFR = 
$$W_1 + W_2(X) + W_3(X^2) + W_4(X^3) + W_5(X^4) + W_6(X^5)$$
, (6)

where WFR = water flow rate, cc/min

TABLE B-2. WATER FLOWMETER CURVE COEFFICIENTS

	Meter 1 Glass Float	Meter l Stainless Steel Float	Meter 2 Stainless Steel Float
W <sub>1</sub>	$0.1124503 \times 10^2$	$-0.3398302 \times 10^{1}$	0.1701297 x 10 <sup>1</sup>
W <sub>2</sub>	-0.1180202 x 10 <sup>1</sup>	0.8969192	0.7123502
W <sub>3</sub>	$0.6830435 \times 10^{-1}$	$0.7994353 \times 10^{-1}$	0.1005951
W4	$-0.7587800 \times 10^{-3}$	$-0.1017442 \times 10^{-2}$	$-0.1434834 \times 10^{-2}$
W <sub>5</sub>	$0.3808533 \times 10^{-5}$	$0.5968658 \times 10^{-5}$	0.8912745 x 10 <sup>-5</sup>
W6	-0-6943106 x 10 <sup>-8</sup>	$-0.1340098 \times 10^{-7}$	-0.2025536 x 10 <sup>-7</sup>

W = calibration coefficient, Table B-2

X = flowmeter scale reading, mm.

The calculated water flow rate was utilized in subsequent calculations for the measured water concentration and the corrected exhaust humidity. For the example situation (run 235), the water flow rate was:

WFR = 
$$384 \text{ cc/min}$$
.

# Engine Performance Calculations

The observed brake horsepower was calculated from the engine speed and load:

$$BHP = \frac{(N)(L)}{K}, \qquad (7)$$

where BHP = brake horsepower

N = engine speed, rpm

L = dynamometer beam load, pounds

K = dynamometer constant

= 3000 for Cummins tests

= 2000 for DDAD runs 1 - 106

= 3000 for DDAD runs 107 - 150.

For the case of Cummins run 235,

BHP = 
$$\frac{(1800)(718)}{3000}$$
 = 431.

The observed torque was obtained from

$$T = \frac{(5252)(L)}{K} = 1257 \text{ lb. ft.},$$
 (8)

where T = torque, 1b. ft.

L = beam load, 1b.

### K = dynamometer constant.

Correction factors for the observed engine performance were developed on the basis of atmospheric conditions. <sup>13</sup> The dry barometric pressure was calculated from

$$P_{B, DRY} = P_{BARO} - \frac{P_{V}}{K_{P}} = 28.20 \text{ in. Hg},$$
 (9)

where  $P_{B, DRY}$  = dry barometric pressure, in. Hg Kp = 3386.4 pascal/in. Hg.

the value of the correction factor was then obtained

$$c_{D} = \left(\frac{29.00}{P_{B}, DRY}\right) \left(\frac{T_{test}}{545}\right)^{0.7}, \tag{10}$$

where  $C_D$  = correction factor  $t_{test}$  = intake air absolute temperature, °R.

For the specific test case,

$$C_{\rm D} = \left(\frac{29.00}{28.20}\right) \left(\frac{90 + 460}{545}\right)^{0.7} = 1.035,$$

therefore, the corrected horsepower was

$$CBHP = (431)(1.035) = 446.$$

The mean effective pressure is a useful parameter that describes engine output per unit area of piston surface. In the calculation routine, values were obtained from the relationship

bmep = 
$$\frac{K_{m} (CBHP)}{(D)(N)}$$
 = 115, (11)

where bmep \* brake mean effective pressure, psi

 $K_m$  = constant, 792,000 for 4-stroke cycle

396,000 for 2-stroke cycle

D = engine displacement, cubic inches

= 1710 for Cummins engine

= 1788 for Detroit Diesel engine

N = engine speed, rpm.

# Air Flow Calculations

During the Cummins engine tests, the air flow to the engine was measured using a Meriam Laminar Flow Element. During test runs, the pressure drop across the element, the pressure drop across the filter, and the temperature of the incoming air were recorded. The volume flow rate was obtained from a curve fitted to the flow element calibration curve by means of the following relationship:

$$CFM = AC + BC (dp) + CC (dp)^2 + DC (dp)^3,$$
 (12)

where CFM = volume flow, cubic feet per minute

dp = pressure drop across element, inches of water

AC = 0.0

BC = 298.825

CC = 5.88898

DC = 0.19913.

For the example calculation, it may be observed that

CFM = 1179.

Two correction factors were used to adjust the volume flow rate to the calibration basis of standard cubic feet per minute. The pressure correction was

$$PCF = \frac{P_{BARO} - (dp \ filter)(0.07355)}{29.92} = 0.957,$$
 (13)

where PCF = pressure correction factor

dp filter = pressure drop across filter, inches of water.

The correction for temperature was obtained from a curve fitted to data supplied with the instrument (Table B-3). 14

$$TCF = X_1 + X_2(T_1) + X_3(T_1)^2 + X_4(T_1)^3 = 0.937,$$
 (14)

where TCF = temperature correction factor

T<sub>1</sub> = inlet air temperature, °F

 $x_1 = 1.28345$ 

 $X_2 = -0.0048289$ 

 $X_3 = 1.227782 \times 10^{-5}$ 

 $X_4 = -1.618912 \times 10^{-8}$ .

The air mass flow rate was then established in terms of air density at the calibration condition  $(70^{\circ}F)$  as:

AMF = 
$$(CFM)(PCF)(TCF)(\rho s) = 79.2,$$
 (15)

where AMF = air mass flow, pounds per minute

 $\rho$ s = density of air at 70°F and 29.92 inches of mercury, pounds per cubic foot.

The air flow rate was adjusted using the previously calculated moisture concentration:

$$DAMF = AMF (1.0 - H), \qquad (16)$$

· similar income

where DAMF = mass flow rate of dry air, pounds per minute
H = moisture, pounds water per pound dry air.

For the example calculation,

DAMF = 79.2 (1.0 - 0.0182) = 77.8.

# TABLE B-3. AIR TEMPERATURE CORRECTION FACTORS

# AIR TEMPERATURE CORRECTION FACTORS FOR SCFM AIR BASE TEMPERATURE 70°F, VISCOSITY 181.87 MICROPOISE REFERENCE NBS CIRCULAR 564 CORRECTION FACTOR = 529.67 459.67+°F × 181.87 µg\*

		μg <b>*</b> \	Viscosity o	f Air at flow	ving temper	ature				
ТЕМР <sup>О</sup> F	+0	+1	+2	+3	+4	+5	+6	+7	+8	+9
50	1.0707	1.0670	1.0633	1.0596	1.0559	1.0523	1.0487	1.0451	1.0415	1.0379
60	1.0344	1.0303	1.0273	1.0238	1.0204	1.0169	1.0135	1.0101	1.0067	1.0033
70	1.0000	.9966	.9933	.9900	. 9867	. 9834	. 9802	. 9770	.9737	. 9705
80	.9674	.9642	.9611	.9579	. 9548	. 9517	. 9486	. 9456	. 9425	ردوو.
90	. 9365	.9335	.9305	.9275	.9246	.9216	.9187	.9158	.9129	9100
100	. 9072	.9043	.9015	. 8987	.8959	.8931	. 8903	. 8875	.8848	.8820
110	.8793	.8766	.8739	.8712	.8686	.8659	.8633	. 8606	. 8580	.8554
120	.8528	.8503	. 8477	.8452	.8426	.8401	.8376	.8351	.8326	. 8301
130	. 8276	.8252	. 8227	.8203	.8179	.8155	.8131	.8107	.8083	.8060
140	. 8036	.8013	. 7990	. 7966	.7943	. 7920	. 7898	. 7875	. 7852	.7-30
150	. 7807	. 7755	. 7763	. 7741	.7713	.7697	. 7675	. 7653	. 7632	.76+3

### Fuel Flow Calculations

During each test run, several measurements of the mass flow rate of diesel fuel were performed. The determinations were made by observing the time required for consumption of a known mass of fuel from a container on a scale; the fuel masses were varied to permit time measurements on the order of two minutes. Each fuel mass flow rate was calculated, and an average was obtained. For the case of Cummins run 235, the following data apply:

Observation	1_	2	_3_	_4
Fuel mass, pounds	5.0	5.0	5.0	5.0
Time, seconds	102.3	102.6	102.4	102.7
Fuel rate, pounds per hour	175.95	175.44	175.78	175.27
Average Fuel Rate	= F =	175 61 pou	nde ner ho	117

The brake specific fuel consumption was calculated from the average fuel rate and the corrected brake horsepower:

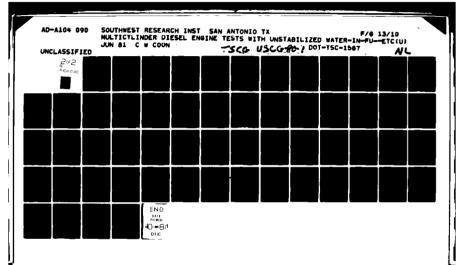
BSFC = 
$$\frac{F}{CBHP}$$
 = 0.3939 pounds fuel per brake horsepower hour. (17)

As a consequence of the fuel and air flow determinations, the observed fuelair ratio was calculated:

$$\left(\frac{F}{A}\right)_{MEAS} = \frac{F}{(DAMF)(60)} = 0.0376. \tag{18}$$

In order to obtain the fuel volume flow rate, a hydrometer measurement of the API gravity of the fuel was obtained and corrected to  $60^{\circ}F$  through the use of ASTM IP Table  $5.^{15}$  The value at  $60^{\circ}F$  was then used in the context of ASTM IP Table  $3^{15}$  to determine the specific gravity of the fuel; for the test case, the specific garvity of the fuel at  $60^{\circ}F$  compared to water at  $60^{\circ}F$  was:

$$SG_{60/60} = 0.8483$$
,



the fuel density was then calculated as

$$\rho_{\text{fuel}} = (\rho_{\text{water}}) (SG_{60/60}) \left( \frac{\text{ft}^3}{1728 \text{ in}^3} \right) \left( \frac{231 \text{ in}^3}{\text{gallon}} \right) = 7.074, \quad (19)$$

where  $\rho_{fuel}$  = fuel density, pounds per gallon

ρwater = density of water at 60°F

= 62.38 pounds per cubic foot

A temperature correction for the difference between the  $60^{\circ}F$  standard and the observed fuel temperature was established from ASTM IP Table  $6^{-15}$  and using the  $60^{\circ}F$  value of the API gravity. Values of the volume reduction factor were selected from the table for the temperature range  $60^{\circ}F - 120^{\circ}F$ , and a curve was fitted to the data. The temperature correction was:

$$TF = Z_1 + Z_2(T_f) + Z_3(T_f)^2 + Z_4(T_f)^3, \qquad (20)$$

where TF = temperature correction factor for fuel volume

T<sub>f</sub> = fuel temperature, °F

 $Z_1 = 0.102678 \times 10$ 

 $Z_2 = -0.446429 \times 10^{-3}$ 

 $Z_3 = -0.14715 \times 10^{-12}$ 

 $Z_4 = 0.10462 \times 10^{-14}$ 

for the example case of run 235,

TF = 0.9844.

The fuel density at the test condition was, therefore,

 $\rho_{ft} = (7.074)(TF) = 6.963$  pounds per gallon,

and the fuel volume flow rate was

$$V_F = \frac{(F)(A)}{(\rho_{F+})(B)} = 1591,$$
 (21)

where  $V_F$  = fuel volume flow rate, cc per minute

A = conversion factor, 3785 cc per gallon

B = conversion factor, 60 minutes per hour.

As a result of the fuel volume flow determination, the water concentration in the fuel mixture was calculated:

$$W = \frac{WFR}{WFR + V_F} \times 100 = 19.4\%,$$
 (22)

where W = water concentration, percent

WFR = water flow rate, cc per minute

 $V_F$  = fuel flow rate, cc per minute.

In order to facilitate subsequent calculations, the water content of the exhaust was modified to include the water introduced with the fuel along with the water entrained in the inlet air. Assuming a density of one gram per cubic centimeter for water,

$$WF = \frac{WFR}{453.6} = 0.8466, \qquad (23)$$

where WF = water flow rate, pounds per minute

WFR = water flow rate, cc per minute,

then,

$$PR = \frac{WF}{DAMF} = 0.0109,$$
 (24)

where PR = moisture added with fuel, pounds water per pound dry air

and

$$H' = H + PR = 0.0217,$$
 (25)

where H' = specific humidity of exhaust, pounds water per pound dry air.

The corrected volume concentration was

$$Y' = \frac{(H')(MAIR)}{MH_{20}} = 0.0349.$$
 (26)

# Exhaust Calculations

From the fuel analysis data shown in Appendix A, the hydrogen/carbon ratio of the fuel was calculated as follows:

HCR = 
$$\left(\frac{\text{HD}}{\text{CA}}\right)\left(\frac{\text{M}_{\text{C}}}{\text{M}_{\text{H}}}\right)$$
 = 1.78, (27)

where HCR = fuel hydrogen/carbon ratio

HD = 12.83 = hydrogen content, percent by weight

CA = 85.75 = carbon content, percent by weight

 $M_C = 12.001 = molecular weight of carbon$ 

 $M_{\rm H}$  = 1.008 = molecular weight of hydrogen.

The concentrations of various exhaust constituents were measured using instruments appropriate to the type of gas and the level present in the exhaust stream. Each relevant range of each instrument was calibrated monthly using at least four gas mixtures within the range, and both zero and span gases were applied to each relevant range before and after testing on each test day. During each test run, scale readings from the instruments were compared to curves developed from the monthly calibrations, and concentrations were reported in parts per million (ppm) or percent. Three gas species, unburned hydrocarbons, carbon monoxide, and carbon dioxide, were of particular importance for the calculation of an air-fuel ratio. The hydrocarbon measurements were made on a wet basis, and the concentrations were corrected to a dry basis during the calculation procedure.

The air-fuel ratio was calculated from exhaust constituent levels using relationships described in the Federal Register. The initial

calculation was for the stoichiometric fuel-air ratio:

$$\left(\frac{F}{A}\right)_{\text{STOICH}} = \frac{M_{\text{C}} + (\text{HCR})}{138.18} \left(1 + \frac{\text{HCR}}{4}\right) = 0.0691.$$
 (28)

The equivalence ratio was then calculated from

$$\phi = \frac{\left(\frac{F}{A}\right)_{MEAS}}{\left(\frac{F}{A}\right)_{STOICH}} = 0.544.$$
 (29)

For convenience, the following ratios were calculated:

$$R_1 = \frac{HCC}{10^6}$$

$$R_2 = \frac{CO}{10^6}$$

$$R_3 = \frac{CO_2}{10^2} ,$$

CO = measured carbon monoxide concentration, parts per
million

CO<sub>2</sub> = measured carbon dioxide concentration, percent.

The wet-to-dry correction factor was then obtained from:

$$K_{W} = \frac{1}{1 + \left[\frac{HCR(R_{2} + R_{3}) + \frac{2Y^{\dagger}}{\phi} (R_{1} + R_{2} + R_{3}) (1 + \frac{HCR}{4})}{2 + \frac{R_{2}}{(R_{3})(K)}}\right]} = 0.929, \quad (30)$$

where  $K_w$  = wet-to-dry correction factor K = water-gas equilibrium constant = 3.5.

then the hydrocarbon concentration that would exist in a dry stream was calculated:

$$HCD = \frac{HCC}{K_w} = 60, \qquad (31)$$

and

$$R_4 = \frac{HCD}{10^5}$$
 (32)

It was convenient to define the parameter

$$\bar{X} = R_2 + R_3 + R_4$$
, (33)

for use in subsequent calculations

The exhaust fuel-air ratio was obtained from the relationship

$$\left(\frac{F}{A}\right)_{\text{calc}} = \frac{4.77 \left(1 + \frac{\text{HCR}}{4}\right) \left(\frac{F}{A}\right) \text{STOICH}}{\frac{1}{X} - \left(\frac{R_2}{2X}\right) - \left(\frac{R_4}{X}\right) + \left(\frac{\text{HCR}}{4}\right) \left(1 - \frac{R_4}{X}\right) - \frac{(0.75) (\text{HCR})}{\left(\frac{R_2}{X}\right) + \left(\frac{1 - K}{1 - \frac{R_4}{X}}\right)}}$$
(34)

For the data representing Cummins run 235, the calculated fuel-air ratio was

$$\left(\frac{F}{A}\right)_{\text{calc}} = 0.0358.$$

The difference between the calculated and measured values of the fuel-air ratio was obtained from

$$D = \frac{\left(\frac{F}{A}\right)_{calc} - \left(\frac{F}{A}\right)_{meas}}{\left(\frac{F}{A}\right)_{meas}} \quad (100) = -4.9, \quad (35)$$

where D = percentage difference between measured and calculated
 fuel-air ratios.

According to reference (12), the absolute value of D should be less than 10 for most engine operating conditions.

The measured concentrations of nitric oxide were corrected for humidity using relationships described in reference (12). The calculation of the correction factor depends upon inlet air temperature, exhaust stream humidity, and the measured dry fuel-air ratio:

$$K_{NO_X} = \frac{1}{1 + A(G - 75) + B(T - 85)} = 1.19,$$
 (36)

where A = 0.044 
$$\left(\frac{F}{A}\right)_{meas}$$
 - 0.0038  
B = -0.116 $\left(\frac{F}{A}\right)_{meas}$  + 0.0053

G = humidity in grains per pound dry air = (7000)(H')

T = inlet air temperature, °F,

then

$$DNO = (NO)(K_{NO_x}), \qquad (37)$$

where DNO = corrected nitric oxide concentration
NO = measured dry nitric oxide concentration.

The above correction is based upon the use of a water-ice bath for condensation of the water vapor present in the exhaust stream. The specific instrument used for this program employed a methanol-dry ice bath for this purpose; the bath temperature was about -150°F. Thus, an additional correction for moisture removal was used:

$$DKNO = \frac{DNO}{1.00678}.$$
 (38)

The dry values of the exhaust constituent concentrations were used for the calculation of mass emissions. Again, several ratios were defined for convenience:

$$R_{5} = \frac{HCD}{10^{4}}$$

$$R_{6} = \frac{CO}{10^{4}}$$

$$R_{7} = R_{6} + CO_{2} + R_{5}$$

$$R_{8} \approx \frac{DKNO}{10^{4}}.$$

The mass emissions, in grams per hour, were obtained from the following relationships:

$$W_{HC} = \frac{(R_5)(W_F)}{R_7} = 61$$
 (39)

$$W_{CO} = \frac{\binom{M_{CO}(R_6)(W_F)}{(M_C + (HCR)(M_H))(R_7)}}{(M_C + (HCR)(M_H))(R_7)} = 306$$
 (40)

$$W_{NO_{X}} = \frac{\binom{M_{NO_{2}}(R_{8})(W_{F})}{(M_{C} + (HCR)(M_{H}))(R_{7})} = 2940, \qquad (41)$$

where  $W_{HC}$ ,  $W_{CO}$ ,  $W_{NO_X}$  = mass emissions of exhaust constituent, grams per hour

W<sub>F</sub> = mass flow rate of diesel fuel, grams per hour = (453.6)(F)

 $M_{CO}$  = molecular weight of CO = 28.0

 $M_{NO_X}$  = molecular weight of  $NO_2$  = 46.0  $M_C$  = molecular weight of carbon  $M_H$  = molecular weight of hydrogen.

The specific emissions were calculated on the basis of the corrected brake horsepower:

$$S_{HC} = \frac{W_{HC}}{CBHP} = 0.14 \tag{42}$$

$$S_{CO} = \frac{W_{CO}}{CBHP} + 0.69 \tag{43}$$

$$S_{NO_{x}} = \frac{W_{NO_{x}}}{CBHP} = 6.59, \qquad (44)$$

where  $S_{HC}$ ,  $S_{CO}$ ,  $S_{NO_X}$  = specific emissions, grams per brake horsepower hour.

## Statistical Calculations

During the Cummins engine tests, statistical procedures were used to evaluate the confidence in certain measured results and to assess the probable effect of the addition of water to the fuel. The performance of the statistical tests required that test procedures be repeated several times under the same conditions in order to provide suitable samples.

As an example of the statistical techniques, two sets of test data will be considered. Table B-4 contains a list of all of the diesel fuel consumption rates observed for the Cummins engine with no water addition and with 20 percent water addition. Sample 1, for no water addition, was regarded as a sample of the entire population of test runs that could be performed at the specified engine setting without water addition. Similarly, Sample 2 was considered to be representative of all of the test runs that might be conducted at the specified engine condition with 20 percent water addition.

The mean of each sample was calculated according to the relationship

$$\bar{X} = \frac{1}{\eta} \sum_{i=1}^{\eta} X, \qquad (45)$$

TABLE B-4. POPULATION SAMPLES

# Diesel Fuel Flow Rates at 1200 RPM Cummins Engine Tests

Sample 1		Sample 2		
0%	. Water	20% Water		
Run	Fuel Rate	Run	Fuel Rate	
178	54.72	182	53.71	
188	55.30	192	53.04	
194	54.43	228	53.92	
224	54.84	242	53.80	
230	55.04	256	54.77	
238	54.94	Mean	53.85	
244	55.58	S	0.6184	
252	56.01	90% band	53.85 ± 0.5896	
258	55.94	ļ		
Mean	55.20			
S	0.5482			
90% band	1 55.20 ± 0.340			

A ......

where  $\bar{X}$  = sample mean

 $\eta$  = number of items in sample

X = value of each fuel rate in the sample.

The calculated mean value for each sample is shown in Table B-4.

The standard deviation for each sample was calculated according to:

$$S = \sqrt{\frac{\sum X^2 - \frac{(\sum X)^2}{\eta}}{n-1}}, \qquad (46)$$

where the individual terms are defined above. The standard deviation for each sample is also shown in Table B-4.

One statistical test was applied to each sample as an individual entity. The Student's t-distribution was used to attach a confidence band to each sample mean. Values of the t-distribution are shown in Table B-5. For a desired confidence level, say 90 percent, it can be argued that the true population mean lies within the band defined by

$$\bar{X} \pm t_{0.95} (\eta - 1) \sqrt{\frac{S}{\eta}}$$
, (47)

where the values of t are obtained from Table B-5. For the example data, the values of the upper and lower limits of the 90 percent confidence band are shown in Table B-4. Thus, it is possible to state with 90 percent confidence that the fuel rate for an additional test at 1200 rpm without water addition would lie between 54.86 and 55.54 pounds per hour.

Since the effect of water addition is desired, it is also desirable to employ a test that compares the two samples. It is possible that the two samples selected are a part of the same population; in that case no definite statement could be made concerning the effect of water addition. The goal of the second statistical procedure is a confidence level for the statement that the means of the two populations (without and with water addition) are different.

As a first step, it was assumed that the two population means were equal. The pooled standard deviation was calculated:

TABLE B-5. CUMULATIVE DISTRIBUTION

<u>v</u>	0.75	0.80	0.85	0.90	0.95	0.975	0.995	0.9995
1	1.0005	1.376	1.963	3.078	6.314	12.706	63.657	636.619
2	0.816	1.061	1.386	1.886	2.920	4.303	8.925	31.598
3	0.765	0.978	1.250	1.638	2.353	3.182	5.841	12.941
4	0.741	0.941	1.190	1.533	2.132	2.776	4.604	8.610
5	0.727	0.920	1.156	1.476	2.015	2.571	4.032	6.859
6	0.718	0.906	1.134	1.440	1.943	2.447	3.707	5.959
7	0.711	0.896	1.119	1.415	1.895	2.365	3.499	5.405
8	0.706	0.889	1.108	1.397	1.860	2.306	3.355	5.041
9	0.703	0.883	1.100	1.383	1.833	2.262	3.250	4.781
10	0.700	0.879	1.093	1.372	1.812	2.228	3.169	4.587
11	0.697	0.876	1.088	1.363	1.796	2.201	3.106	4.437
12	0.695	0.873	1.083	1.356	1.782	2.179	3.055	4.318
13	0.694	0.870	1.079	1.350	1.771	2.160	3.012	4.221
14	0.692	0.868	1.076	1.345	1.761	2.145	2.977	4.140
15	0.691	0.866	1.074	1.341	1.753	2.131	2.947	4.073
16	0.690	0.866	1.071	1.337	1.746	2.120	2.921	4.015
17	0.689	0.863	1.069	1.333	1.740	2.110	2.898	3.965
18	0.688	0.862	1.067	1.330	1.734	2.101	2.878	3.922
19	0.688	0.861	1.066	1.328	1.729	2.093	2.861	3.883
20	0.687	0.860	1.064	1.325	1.725	2.086	2.845	3.850
21	0.686	0.859	1.063	1.323	1.721	2.080	2.831	3.819
22	0.686	0.858	1.061	1.321	1.717	2.074	2.819	3.792
23	0.685	0.858	1.060	1.319	1.714	2.069	2.807	3.767
24	0.685	0.857	1.059	1.318	1.711	2.064	2.797	3.745
25	0.684	0.856	1.058	1.316	1.708	2.060	2.787	3.725
26 27 28 29 30	0.684 0.684 0.683 0.683	0.856 0.855 0.855 0.854 0.854	1.058 1.057 1.056 1.055 1.055	1.315 1.314 1.313 1.311 1.310	1.706 1.703 1.701 1.699 1.697	2.056 2.052 2.048 2.045 2.042	2.779 2.771 2.763 2.756 2.750	3.707 3.690 3.674 3.659 3.646
35 40 45 50 55	0.682 0.681 0.680 0.680 0.679	0.852 0.851 1.048 0.849 0.849	1.052 1.050 1.048 1.047	1.306 1.303 1.301 1.299 1.297	1.690 1.684 1.680 1.676 1.673	2.030 2.021 2.014 2.008 2.004	2.724 2.704 2.690 2.678 2.669	3.591 3.551 3.520 3.496 3.476
60	0.679	0.848	1.046	1.296	1.671	2.000	2.660	3.460
70	0.678	0.847	1.045	1.294	1.667	1.994	2.648	3.435
80	0.678	0.847	1.044	1.293	1.665	1.990	2.638	3.416
90	0.678	0.846	1.043	1.291	1.662	1.987	2.632	3.402
100	0.677	0.846	1.042	1.290	1.661	1.984	2.626	3.390
200	0.676	0.844	1.039	1.286	1.653	1.972	2.601	3.340
300	0.676	0.843	1.038	1.285	1.650	1.968	2.592	3.323
400	0.676	0.843	1.038	1.284	1.649	1.966	2.588	3.315
500	0.676	0.843	1.037	1.284	1.684	1.965	2.586	3.310
1000	0.675	0.842	1.037	1.283	1.647	1.962	2.581	3.301
œ	0.67449	0.84162	1.03643	1.28155	1.64485	1.95996	2.57582	3.29053

$$S = \frac{(\eta_1 - 1) S_1^2 + (\eta_2 - 1) S_2^2}{\eta_1 + \eta_2 - 2} = 0.3278, \tag{48}$$

where  $\eta$  = sample size

S = sample standard deviation,

then

$$S_{X_1} - \bar{X}_2 = \sqrt{\frac{S^2}{\eta_1} + \frac{S^2}{\eta_2}} = 0.3194$$
, (49)

and

$$T = \frac{\bar{x}_1 - \bar{x}_2}{s\bar{x}_1 - \bar{x}_2} = 4.2272, \qquad (50)$$

now, if

$$T \leq -t (1 - \frac{\alpha}{2}) (\eta_1 + \eta_2 - 2),$$
 (51)

or

$$T \stackrel{>}{=} t(1 - \frac{\alpha}{2})(\eta_1 + \eta_2 - 2),$$
 (52)

where  $\alpha$  is the probability of rejecting a true hypothesis, then the hypothesis of equal sample means can be rejected. For the present case, using Table B-5,

$$T > t_{(.995)(12)}$$

and

$$1 - \frac{\alpha}{2} = 0.995$$
,

imply that  $\alpha = 0.01$ .

Thus, it is possible to state with 99 percent confidence that the two samples represent different populations and that significance can be attached to the difference between the means.

APPENDIX C TEST RESULTS

TABLE C-1. ENGINE TEST RESULTS, CUMMINS ENGINE, 900 RPM, BASELINE

DYNAMOMETER CONSTANT: 30 H/C RATIO: 1.78	00 API	GRAVITY	OF DIESE	EL FUEL:	35.3 AT	60F	
KUN NUMBER NOM. WATER PCT.	25 <b>9</b> . 0 .	262 0	268 0	269 0	275. 0.	276 . V .	282 0
ENGINE SPEED RPM OBS. TORQUE LB-FT	900. 257.	900. 257.	900. 257.	900. 257.	900. 257.	900. 257.	900 : 257 :
BAR PRESS. 1N-HG DRY BULB DEG F WET BULB DEG F REL HUMIDITY PCT CORR BHP HP CORR BMEP PSI	28.95 83. 69. 49. 44.9 23.1	29.14 77. 66. 56. 44.1 22.7	29.12 87. 66. 32. 44.8 23.1	29.32 78. 66. 53.9 22.6	29.25 91. 68. 30. 44.8 23.0	29.30 77. 61. 39. 43.6 22.5	29.21 89. 65. 26. 44.6 23.0
FUEL FLOW LB/HR WATER FLOW CC/MIN CALC. VOL. % PCT BSFC LB/BHP-HR AIR FLOW LB/MIN	25.12 0.0 0.0 5590 26.1	25.11 0.0 0.0 5692 26.5	24.73 0.0 0.0 .5516 26.0	25.52 0.0 0.0 .5817 26.3	25.24 0.0 0.0 .5635 26.1	24.70 0.0 0.0 .5662 26.6	24.85 0.0 0.0 5570 26.1
CORR. BHP PSI  FUEL FLOW LB/HR WATER FLOW CC/MIN CALC. VOL. X BSFC AIR FLOW LB/HR COOLANT IN DEG F COOLANT IN DEG F FUEL RETURN DEG F F F F F F F F F F F F F F F F F F F	384071737014474510939364895012 1188581114474510939364895012 441111445 54454545454	1111 11 44 1144 544545454545454545454545	953318107272466430128995503960 789950996801774399643084644369 44111144 544545454454	744699777318853220832707811 44 1144 5445454542227	11775 11775 11775 11772 11772 11772 1172 11	1111 44 54454545454545454545454545454545	631257004877241070712070378988 7879407967007743795321946333588 111 44111144 54454545454
RAIL PRESSURE PSI BOOST (R) PSI BOOST (L) PSI INLET VAC. (R) IN-H20 EXH. PRESS (R)PSI EXH. PRESS (L)PSI TUKB IN (R) IN-HG	455 1200000 200000 100000000000000000000000	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	466 1200000 1200000 20030	486 1200000 20020	486 1.5000 1.50000 00.000 200.000 100.30	50 4 50 00 00 00 00 00 00 00 00 00 00 00 00	46 1200000 1200000 20 30

TABLE C-2. ENGINE TEST RESULTS, CUMMINS ENGINE, 900 RPM, 5% WATER

DYNAMOMETER CONSTANT: 3000 H/C RATIO: 1.78	API	GRAVITY	OF DIESE	L FUEL:	35.3 AT 60F
KUN NUMBER NOM. WATER PCT	260 . S .	263. 5.	270 . S	277. 5.	
ENGINE SPEED KPM OBS. TORQUE LB-FT	900 257	900. 257.	900 257	900 257.	
BAK. PRESS. IN-HG DRY BULB DEG F WET BULB DEG F REL. HUMIDITY PCT CURR. BHP HP CORR. BMEP PSI	28.90 85. 70. 47. 45.3 23.3	29.15 77. 66. 56. 44.3 22.8	29.34 78. 66. 53. 44.1 22.7	29.31 77. 61. 39. 43.8	
FUEL FLOW LE/HR WATER FLOW CC/MIN CALC. VOL. % PCT. BSFC LB/BHP-HR	24.82 11.5 4.9 .5483	25.06 10.5 4.4 .5652	24.89 11.1 4.7 5646	24.73 10.5 4.5 .5 <b>6</b> 47	
COOLANT IN DEG F COOLANT OUT DEG F COOLANT OUT DEG F COOLANT OUT DEG F FUEL SUMP FUEL RETURN DEG F FUEL COOLER DEG F INTAKE AIR DEG F TURB. INLET (R) DEG F TURB. INLET (R) DEG F COMP. OUT (R) DEG F COMP. OUT (R) DEG F CHARGE AIR (L) DEG F EXH. STACK (L) DEG F EXH. STACK (L) DEG F EXHAUST 1R DEG F EXHAUST 1R DEG F EXHAUST 3R DEG F EXHAUST 5R DEG F EXHAUST 5C DEG F	274914678747474946350710432555 889951984600773198633974633247 44111144 54444545455	111 789940998668250550039940656133 789940975799774288632983633337 1144 54445454454	942267108977354641806106450561 942267108977354641806106450561 94459723237	111 11 44 1144 54445454558 7899409746997731874219725236 740257088655143749949577183638	
OIL PRESSURE PSI KALL PRESSURE PSI BOOST (R) PSI BOUST (L) PSI INLET VAC. (R) IN-H20 EXH. PRESS. (L)PSI TUKB. IN. (R) IN-HG TURB. IN. (R) IN-HG FURE PRESS. PSI EMULSION PRESS. PSI WATER PRESS. PSI	486 1200000 486 1200000 200000 10 60	48 6 120000 00000000000000000000000000000000	486 120000 055220000 00000 2003	96 1200000 20030 100030	

TABLE C-3. ENGINE TEST RESULTS, CUMMINS ENGINE, 900 RPM, 10% WATER

DYNAMOMETER CONSTANT: 30 H/C RATIO: 1.78					35.3	AT	60F
RUN NUMBER NOM. WATER PCT.	261. 10.	264. 10.	271. 10.	278. 10.			
ENGINE SPEED RPM OBS. TORQUE LB-FT	900 : 257 :	900. 257.	900 257	900. 257.			
BAR. PRESS. IN-HG DRY BULB DEG F WHT BULB DEG F REL. HUMIDITY PCT CUKR. BHP HP CORR. BMEP PSI	28.89 85. 70. 48. 45.4 23.4	29.15 82. 66. 43.5 44.5 22.9	29.35 82. 67. 46.2 22.7	29.32 80. 64. 41. 44.0 22.6			
FUEL FLOW LEA/HR WATER FLOW CC/MIN CALC. VOL. % PCT. BSFC LE/BHP-HR	24.87 17.3 7.1 5479	25.03 19.7 8.0 .5627	25.31 19.7 7.9 .5731	24.89 19.7 8.1 .5660			
COOLANT IN DEG F COULANT OUT DEG F OIL SUMP DEG F FUEL IN DEG F FUEL RETURN DEG F FUEL SUPPLY DEG F FUEL COMP DEG F COMP OUT (R) DEG F COMP OUT (R) DEG F CHARGE AIR (R) DEG F CHARGE AIR (R) DEG F EXHAUST 1R EXHAUST 1R EXHAUST 1R EXHAUST 3R DEG F EXHAUST 5R EXHAUST 1L EXHAUST 1L EXHAUST 3L EXHAUST 5L	495254192760683228187457655450 889051084601773198531972612137 11111 441111144 54445454554	841217035394240561043603442912 789914984690773188532572623236 11144 544545454	942481220199357943471251096184 789941220199357943471251096184 544445454	407369209766814820853946490803 788940983599672088321871512186 1144 544445454454			
OIL PRESSURE PSI RAIL PRESSURE PSI BOOST (R) PSI HOUST (L) PSI INLET VAC. (R) IN-H20 EXH. PRESS. (L)PSI EXH. PRESS. (L)PSI TURB. IN. (R) IN-HG TURB. IN. (L) IN-HG FUEL PRESS. PSI EMULSION PRESS. PSI FUEL SUPPLY PSI WATER PRESS. PSI	477 1200000 200000 200000000000000000000000	97 12000000 4 12000000000000000000000000000000000000	487 12.000.00 00.00 10.00 10.00 10.00 10.00 10.00 10.00	05520000 1200000 2030 1			

TABLE C-4. ENGINE TEST RESULTS, CUMMINS ENGINE, 900 RPM, 15% WATER

NUMBER CONCIONS AND	Τ <b>Φ</b> Δ	CPAUITY	OF DIESEL	FUEL:	35.3	AT	60F
DYNAMOMETER CONSTANT: 3000 H/C RATIO: 1.78							
RUN NUMBER NOM, WATER PCT.	265. 15.	272. 15.	279. 15.				
ENGINE SPEED RPM OBS. TORQUE LB-FT	900. 257.						
BAR PRESS IN-HG DRY BULB DEG F WET BULB DEG F REL HUMIDITY PCT CORR BHP HP CORR BMEP PSI	29.15 83. 67. 44.6 22.9	29 34 87 68 37 44 4 22 8	29.30 84. 66. 38. 44.3 22.8				
FUEL FLOW LB/HR WATER FLOW CC/MIN CALC. VOL. % PCT. BSFC LB/BHP-HR Alk FLOW LF/MIN	24.50 33.8 13.2 5498 26.1	24.29 33.8 13.3 5474 26.3	24.46 33.2 13.2 156.3				
COOLANT IN DEG F COULANT OUT DEG F COULANT OUT DEG F COULANT OUT DEG F COULANT OUT DEG F COULANT DEG F FUEL IN DEG F FUEL SUPPLY DEG F FUEL SUPPLY DEG F FUEL SUPPLY DEG F FUEL COOLER DEG F FUEL COOLER DEG F COMP DEG F COMP OUT (R) DEG F COMP	952229040905465983090174319505 789914984500772088422961611235 441111144 544445454554	052702565722464696876919887161 88995198350277220883221871501226 44111144 544454545461	840929745500353665775807700394 78995098350077208832187154226 44111144 S4444545454				
OIL PRESSURE PSI RAIL PRESSURE PSI BOOST (R) PSI BOUST (L) PSI INLET VAC. (R) IN-H20 EXH. PRESS. (L)PSI EXH. PRESS. (L)PSI UKB. IN. (R) IN-HG TURB. IN. (R) IN-HG FUEL PRESS. PSI EMULSION PRESS. PSI FUEL SUPPLY PSI WATER PRESS. PSI	487 20 00 00 00 00 00 00 00 00 00 00 00 00	487 120 00 00 00 00 00 00 00 00 00 00 00 00 0	497 120000 20030 20030				

TABLE C-5. ENGINE TEST RESULTS, CUMMINS ENGINE, 900 RPM, 20% WATER

DYNAMOMETER CONSTANT: 3000 H/C RATIO: 1.78				FUEL:	35.3	AT	60F
KUN NUMBER NOM. WATER PCT.	266 . 20 .	273. 20.	280 . 20 .				
ENGINE SPEED RPM OBS. TORQUE LB-FT							
BAR. PRESS. 1N-HG DRY BULB DEG F WET BULB DEG F REL. HUMIDITY PCT CUKR. BHP HP CORR. BMEP PSI	29.14 83. 67. 43. 44.7 23.0	29 32 87 68 37 44 5 22 9	29, 28 84. 66. 38. 44.4 22.8				
FUEL FLOW LB/HR WATER FLOW CC/MIN CALC. VOL. % PCT. BSFC LB/BHP-HR AIR FLOW LB/MIN	24.45 49.8 18.4 .5469 25.7	24.81 49.8 18.1 5573 26.2	24.68 49.8 18.2 .556.2				
COOLANT IN DEG F CUOLANT OUT DEG F FUEL SUMP FUEL RETURN DEG F FUEL COOLER DEG F INTAKE AIR TURB. INLET (R) DEG F INTAKE AIR TURB. INLET (R) DEG F COMP. OUT (R) DEG F COMP. OUT (R) DEG F CHARGE AIR (L) DEG F CHARGE AIR (R) DEG F EXH. STACK (L) DEG F EXH. STACK (L) DEG F EXHAUST 1R CELLA AIR CELLA AIR CELLA AIR CEXHAUST 2R CEXHAUST 3R CEXHAUST 4R CEXHAUST 5R CEXHAU	942415262827465205012372588450 789951983400771098211850580114 5444544454	0.62922886844575799849205247991 88995298240077198881008605800091 111 44111143 54444544454	6305033564302042687696731065172 789951982400771938100760580103 111 441111143 54444544454				
OIL PRESSURE PSI KAIL PRESSURE PSI BOOST (R) PSI HUUST (L) PSI INLET VAC. (R) IN-H20 EXH. PRESS. (R)PSI EXH. PRESS. (L)PSI TURB. IN. (R) IN-HG TURB. IN. (L) IN-HG FURL PRESS. PSI EMULSION PRESS. PSI FUEL SUPPLY WATER PRESS. PSI	487 200000 20030 10000	48 120.000 0.000 200.30	98 1200000 2030 1				

TABLE C-6. ENGINE TEST RESULTS, CUMMINS ENGINE, 900 RPM, 25% WATER

DYNAMOMETER CO H/C RATIO: 1.7	NSTANT: 3000 B				FUEL:	35.3	ΓA	60F
RUN NUMBER NOM. WATER PCT		267 . 25 .	274. 25.	28 <u>1</u> . 25 .				
ENCINE SPEED ORS: TORQUE	RPM LBFT	900 257	900. 257.	900. 257.				
BAR. PRESS DRY BULB WET BULB REL. HUMIDITY CORR. BHP CORR. BMEP	IN-HG DEG F DEG F PCT HP PSI	29.13 87. 66. 32. 44.7 23.0	29.29 90. 69. 34. 44.7 23.0	29 . 25				
FUEL FLOW WATER FLOW CALC. VOL. % RSFC AIR FLOW	LB/HR CC/MIN PCI LB/BHPHR LB/MIN	25.16 66.4 22.6 5628 26.0	24.84 66.4 22.7 .5560 26.2	24.73 66.4 22.9 5562 26.1				
COOLANT IN CUOLANT OUT OIL SUMP FUEL SUMPLY FUEL SUMPLY FUEL COOLER INTAKE AIR TURB. INLET (R COMP. OUT (L) CHARGE AIR (R) CHARGE AIR (R) EXH. STACK (R) EXH. STACK (L) CHARGE AIR EXHAUST IR EXHAUST IR EXHAUST IR EXHAUST SR EXHAUST SR EXHAUST SR EXHAUST SR EXHAUST SR EXHAUST SL	DDDDDDDDDDDDDDDDDDDDDDDDDDDDDDDDDDDDDD	384542388350681098762305031162 88995198240177108800017595831162 111 441111144 544444444444444	173054006577682800031119012339 88905109240077149991031119012339 5444444454	492. 446. 470. 409. 491. 432.				
OIL PRESSURE RAIL PRESSURE BOOST (R) HOOST (L) INLET VAC. (R) EXH. PRESS. (L) IUKB. IN. (R) TURB. IN. (L) FUEL PRESS. EMULSION PRESS. FUEL SUPPLY WATER PRESS.	PSI PSI PSI PSI PSI PSI PSI PSI PSI PSI	488 200000 20000 10030	489 120000 05520000 20030	05500000000000000000000000000000000000				

TABLE C-7. ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, BASELINE

DYNAMOMETER CONSTANT: 3000 H/C RATIO: 1 78	API	GRAVITY	OF DIES	EL FUEL:	35.3 AT	60F			
RUN NUMBER NOM: WATER PCT:	178. 0.	188 . 0 .	194. 0.	224 0	230 0	238 0	244 . 0 .	252 . 0 .	258 . 0 .
ENGINE SPEED RPM OBS. TORQUE LB-FT	1200 508	1200 . 500 .	1200 508	1200 508	1200 . 508 .	1200. 508.	1200 508	1200 . 508 .	1200. 508.
BAR PRESS IN-HG DRY BULB DEG F WIT PULB DEG F REL HUMIDITY PCT CURR BHP HP CORR BHEP PSI	29.33 78. 63. 43. 115.4 44.6	29 11 73 68 116 2 44 8	29.17 81. 72. 65. 117.5 45.4	29.03 82. 76. 76. 118.9 45.9	29.03 89. 76. 119.9 46.3	28.95 86. 76. 63. 119.8 46.2	28.92 100 78 38 121 6 46.9	29.01 82: 77: 80: 119:4 46:1	29.10 81. 74. 117.8 45.5
FUEL FLOW LB/HR WATER FLOW CC/MIN CALC VOL. % PCT BSFC VOL. % PCT LB/BHP-HR A1R FLOW LB/HIN	54.72 0.0 0.0 .4740 38.4	55.30 0.0 0.0 4761 38.0	54.43 0.0 0.0 .4631 37.3	54.84 0.0 0.0 4613 36.5	55.04 0.0 0.0 4589 36.8	54.94 0.0 0.0 4586 37.3	55.58 0.0 0.0 4570 36.7	56.01 0.0 0.0 4689 36.9	55.94 0.0 0.0 4248 37.3
STOICH, F/A MEAS: F/A CALC: F/A % DIFF. PCT	0691 0237 0256 8 07	0691 0242 0248 2 37	0691 0243 0250 2.74	0691 0250 0246 -1 67	.0691 .0249 .0245 -1.80	.0691 .0245 .0250 2.07	0691 0252 0255 1 18	0691 0253 0235 -7 14	0691 0250 0252 61
COOLANT IN DEG F COOLANT OUT DEG F COOLANT OUT DEG F COOLANT OUT DEG F FUEL SUMP FUEL RETURN DEG F FUEL COOLER DEG F FUEL COOLER DEG F TURB. INLET (L)DEG F TURB. INLET (L)DEG F TURB. INLET (L)DEG F COMP. OUT (R) DEG F COMP. OUT (R) DEG F CHARGE AIR (R) DEG F EXHAUST STACK (R) DEG F EXHAUST STACK (R) WATER INLET EXHAUST SR DEG F	189949971099997728353740 18994997437116689876097728353740 1999437116689876097728353740	214294029765899456380941581899 78994997250066977719806515701397 6611169777756667666666	18975222117022901960181015223600 1199509837022901960181015253600 661116707880181015253606	11111198651320285651382390211093 66511177908811888590211093 666111177908871888590211093 75566766676	12151329618432242942517015362302 75032961843224729942517015362302 756643802	111997446788889900970984034026649911165899709840340266499	1285309807711190413183893297340111111111111111111111111111111111111	1198449449135576584737460564 09865111773557658473746056767676676	7596054972111130060450231710860 7696054972111130060450231710860 756667676221
OIL PRESSURE PSI RAIL PRESSURE PSI BOOST (R) PSI BOOST (R) PSI INLET VAC (R) IN-H20 EXH PRESS (R)PSI EXH PRESS (L)PSI EXH PRESS (L)PSI EXH PRESS (L) IN-HG TURB IN (R) IN-HG TURB IN (L) IN-HG FUEL PRESS PSI EMULSION PRESS PSI	65 9 8 6 6 1 1 1 5 4 1 1 2 2 3 2 1 1 8 2 0	66 0858 11554 147 23 21154 200 20	0859 1155 21020	08520155 11240 210 20020	08650155 08650155 21020 20020	2.5	085800055 112401210 20020	08400155 615-1250 21-10020 10020	0804-200 40-00 0-000 61 000 000
HYDROCARBONS PPHC CARBON MONOXIDEPPH NITRIC OXIDE PPH NITROGEN OXIDESPPH CARBON DIOXIDE PCT OXYGEN PCT SMOKE OPACITY PCT	96. 488. 340. 355. 5.4 14.3	54 218 313 315 15 8	94 5063 268 25.3 15.8 8.7	77 : 201 : 213 : 234 : 5 : 3 : 11 : 6 : 9 : 6	124 179 179 2251 255 9 12	120 233 237 230 250 13.9 10.2	237 237 258 5 15 6 10 5	70. 239. 174. 190. 5.0 13.6	235 235 240 5 2 15 5
HC MASS GM-MR CU MASS GM-MR NOX GM-MR BSHC GM-MR BSHC GM-MHP-HR BSHO GM-MHP-HR	45.656 446.11 521.28 3.955 3.8642 4.5146	26 649 203 00 535 294 1 7475 4 6137	40 645 475 09 484 32 3459 4 0426 4 1211	37.936 268.15 431.38 2.2558 3.6290	62.050 192.15 441.54 1.6023 3.6818	58.433 218.68 404.88 4878 1.8255 3.3799	56.697 223.13 419.61 1.8348 3.4504	37 212 244 86 383 03 2 0501 3 2069	29 219 221 67 427 38 2480 1 8813 3 6272

TABLE C-8. ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, 5% WATER

DYNAMOMETER CO H/C RATIO: 1.7	005 : THATEN	0 AP	GRAVITY	OF DIES	SEL FUEL:	35.3 AT 601	f
RUN NUMBER NOM. WATER PCT		179. 5.	189	225 . 5 .	23 <b>9</b> .	25 <u>3</u>	
ENGINE SPEED OBS. TORQUE	RPM LB-FT	1200 508	1200 508	1200 508	1200 508	1200 508	
BAR PRESS DRY BULB WET BULB REL HUMIDITY CURR BHP CORR BMEP	IN-HG DEG F DEG F PCT HP PSI	29.32 78. 63. 43. 115.6 44.6	29.14 73. 68. 78. 116.2 44.8	29.04 82. 76. 76. 118.7 45.8	28.97 86 76 63 120.0 46.3	29.02 82. 77 80. 119.5 46.1	
FUEL FLOW WATER FLOW CALC. VOL. % BSFC AIR FLOW	LB/HR CC/MIN PCT LB/BHP-HR LB/MIN	54.66 23.7 4.6 4727 38.4	54.55 23.7 4.6 4694 38.0	54.86 25.0 4.8 .4622 35.7	54.48 23.7 4.6 4539 37.1	\$5, 53 18, 5 3, 5 4645 36, 9	
STOICH, F/A MEAS, F/A CALC, F/A % DIFF.	PCT	0691 0237 0252 6 05	0691 0240 0246 2 61	0691 0256 0244 -4.78	0691 0245 0250 2.34	0691 0251 -0245 -2-35	
COOLANT IN CUOLANT OUT OUT OUT OUT OUT OUT OUT OUT OUT OU	D D D D D D D D D D D D D D D D D D D	895583206822684138392037587071 67994998422166479877740527582388 66111667987740527582388	1111973399558914554769917899492 661106694770866666666666666666666666666666666666	1119040571382212754435146487444477 789093713822127588881906334744477 7566667666666	111607669/7010030620243/4777 111607669/7010030620243/4777 66111156 6566676766676	6448992553267026756575542621220 111994698551177998898575542621220 651117798898575542621220 650566666666666666666666666666666666	
DIL PRESSURE kAIL PRESSURE BOOST (R) KUUST (L) INLET VAC (R) EXH PRESS (R EXH PRESS (L TURB IN (L) TUR	PSI PSI PSI IN-H2O PSI OPSI	08671153 218020 10020	08580054 01123111212000 00000	0 850 0 155 0 210 20 100 20 100 20	08540055 111240055 10020	65.08334254 22.54 2100 1000	
HYDROCARBONS CARBON MONOXIDE NITRIC OXIDE NITROGEN OXIDE: LARBON DIOXIDE OXYGEN SMUKE UPACITY	PPMC PPPM PPM PPCT PCT PCT	1059 339 350 355 13.7	70 488 275 300 5 16 0	107 267 210 231 5 2 11 6	142 219 214 235 3 14 0	38 219 178 178 5.2 13.7 6.5	
HC MASS CO MASS NOX MASS ESHC BSCO ESNO	GM-HR GM-HK GM-HR GM/BHP-HR GM/BHP-HR GM/BHP-HK	50.691 222.69 537.09 4384 1.9258 4.6447	34.560 464.32 527.91 2974 3.9961 4.5434	54.561 257.26 444.20 .4597 2.1678 3.7430	68.819 204.27 423.10 5734 1.7020 3.5254	44.503 213.32 370.01 -3723 1.7844 3.2623	

TABLE C-9. ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, 10% WATER

DYNAMOMETER CO H/C RATIO: 1.7	NSTANT: 3000	) API	GRAVITY	OF DIES	EL FUEL:	35.3 AT 60	ıF
RUN NUMBER NOM. WATER PCT		180. 10.	190 10:	226 . 10 .	240. 10.	254 . 10	
ENGINE SPEED OBS: TORQUE	RPM LB-FT	1200. 508.	1200. 508.	1200. 500.	1200. 508.	1200. 508.	
BAR. PRESS. DRY BULB WET BULB REL. HUMIDITY CORR. BHP CORR. BHEP	IN-HG DEG F DEG F PCT HP PSI	29.31 80. 65. 44. 116.1 44.8	29 14 74 68 74 116 1 44 8	29.05 82. 75. 72. 118.5 45.7	28.96 92. 78. 54. 120.6 46.5	29.04 86. 79. 74. 119.6 46.2	
FUEL FLOW WATER FLOW CALC. VOL. X BSFC AIR FLOW	LB/HR CC/MIN PCT LB/BHP-HR LB/MIN	53.63 49.8 9.3 .4617 37.2	53.71 49.8 9.3 4624 37.9	54.38 49.8 9.2 .4591 37.1	53.90 49.8 9.2 .4470 36.5	55,02 49.0 4599 36.7	
STOICH F/A MEAS F/A CALC F/A % DIFF	PCT	0691 0240 0249 3.57	0691 0236 0242 2 52	.0691 .0245 .0240 -2.03	0691 0246 0252 2 37	.0691 .0250 .0235 -6.06	
COOLANT OUT CUOLANT OUT OIL SUMP FUEL SUMP FUEL RETURN FUEL SUPPLY FUEL COOLER INTAKE AIR TURB. OUT (R) COMP. OUT (R) CHARGE AIR (R) EXH. STACK (L) WATER INLET EXHAUST 2R EXHAUST 2R EXHAUST 3R EXHAUST 5R EXHAUST 5R EXHAUST 5R EXHAUST 5R EXHAUST 5R EXHAUST 5R EXHAUST 5L	00000000000000000000000000000000000000	111970662314389885112885139317 661116667886112885139317 66111666788655931747093176	111940993311044895794483000573212 6611046675777483000573212 661106675777483000573212	199084618522027254421328481880 199084618522027254421328481880 66111177878888791328481880 6611117558656666666666666666666666666666		44808245245570238640880039908285 119708889894495880176 55111155 6534666666666666666666666666666666666	
OIL PRESSURE RAIL PRESSURE BOOST (R) BOOST (A) INLET VAC (R) EXH PRESS (R EXH PRESS (L TURB IN (L) TURB IN (L) FUEL PRESS EMULSION PRESS FUEL SUPPLY WATER PRESS	PSI PSI PSI PSI PSI PSI PSI PSI PSI PSI	25 25 25 25 25 25 25 25 25 25 25 25 25 2	285811554 2123 212840 2125 01	07500-53 671240 210020 20020	07556 11246 0 2104 20020 10020	067771077 611000 210000 6000	
HYDROCARBONS CARBON MONOXIDI NITRIC OXIDE NITROGEN OXIDE CARBON DIOXIDE OXYGEN SMOKE OPACITY	PPMC EPPM PPM BPPM BPCT PCT PCT	290 320 350 151 13 2	5075 2775 16 1 16 2	1740 1740 1700 114 14 15	22255 22255 14 22222 14 14	96. 2215. 240. 5.0 15.1	
RENO BRCO BRCO HARS CO MARS HC MARS	GM-HR GM-HR GM-HR GM-BHP-HR GM/BHP-HR GM/BHP-HR	47,349 275,89 557,52 4077 2,3754 4,8002	45.610 493.12 496.37 3927 4.1595 4.2736	60 .147 266 .69 450 .12 -5077 2 .2513 3 .7997	58.619 204.09 419.09 .4061 1.6923 3.4753	50,255 523,53 516,01 4200 1,8683 4,3130	

TABLE C-10. ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, 15% WATER

DYNAMOMETER CONSTANT: 3000 H/C RATIO: 1.78	API	GRAVITY	OF DIESEI	_ FUEL: :	35.3 AT 60F
H/C RATIO: 1.78  RUN NUMBER NOM. WATER PCT.	181	191 15	227 . 15	24i 15	255. 15
ENGINE SPEED RPM OBS. TORQUE LB-FT	1200. 508.	1280 508	1200 508	1200 . 508	1200. 508.
BAR PRESS IN-HG DRY BULB DEG F WEI BULB DEG F REL HUMIDITY CURR BHP HP CORR BHEP PSI	29.29 81. 64. 39. 116.2 44.8	29.15 74. 68. 74. 116.4 44.9	29.05 84. 76. 118.7 45.8	28.96 93. 79. 54. 121.0 46.7	29.06 85. 77. 70. 119.1 46.0
FUEL FLOW LE/HR WATER FLOW CC/MIN CALC. VOL. % PCT BSFC Alr FLOW LB/BHP-HR LB/MIN	54 19 79 6 13 9 4664 37 7	53.25 82.8 14.6 4575 37.6	53.70 82.8 14.5 4524 36.9	53.06 81.2 14.4 4384 36.3	53.38 82.8 14.6 .4483 36.6
STOICH. F/A MEAS. F/A CALC. F/A % DIFF. PCT	.0691 .0240 .0249 3.74	.0691 .0236 .0240 1.55	0691 0242 0242 - 26	.0691 .0244 .0251 2.80	0691 0243 0250 2.66
COOLANT IN DEG F FOUL SUMP DEG F F F F F F F F F F F F F F F F F F F	00500068372244891563134912294256 111111 66111166688558217359054 111111 66111155 65566665666	326799558755894726890703393747 1119499791006664875593166148832 56111155 655666685566	66111759889701999999999999999999999999999999999	427399945833125155319965424273 7890409921227766799668306570075 66111155 65566666666666666666666666666	17859 08 22 74 58 47 23 4 28 6 6 22 23 4 6 111 115 5 8 8 7 8 2 2 2 3 4 6 6 5 9 6 6 6 9 9 6 6 9 9 6 6 9 9 6 6 6 9 9 6 6 9 6 9 9 6 6 6 9 9 6 9 6 9 6 9 6 9 9 6 9 6 9 9 6 9 6 9 9 6 9 9 6 9 9 6 9 9 6 9
OIL PRESSURE PSI RAIL PRESSURE PSI BOOST (R) PSI BOUST (L) PSI INLET VAC (R) IN-H20 EXH PRESS (L) PSI TURB IN (R) IN-HG TURB IN (L) IN-HG FUEL PRESS PSI ENULSION PRESS PSI FUEL SUPPLY WATER PRESS PSI WATER PRESS PSI	28887 11552 190 20	55.5858 1152 1023 1025 1025	085200053 10124001210 20020	6576015760 210020 100260	08231255 210000
HYDROCARBONS PPMC CARBON MONOXIDEPPM NITRICOXIDEPPM NITROGEN OXIDEPPM CARBON DIOXIDE PCT OXYGEN SMUKE OPACITY PCT	125 2460 3555 1334	108 5243 300 16 1 16 3	124 246 2034 235 4	153922 253922 15 4 6	1050000 1050000 105000 105000 105000 105000 105000 105000 105000 105000 1050000 105000 105000 105000 105000 105000 105000 105000 105000 1050000 105000 105000 105000 105000 105000 105000 105000 105000 1050000 105000 105000 105000 105000 105000 105000 105000 105000 1050000 105
HC MASS GM-HR LU MASS GM-HR NOX MASS GM-HR SHC GM/BHP-HR BSCO GM/BHP-HR BSCO GM/BHP-HR	60 609 230 08 579 26 5216 1 9800 4 9849	53.197 544.39 568.46 4570 4.6764 4.6832	61 402 234 59 476 76 5173 1 9764 4 0167	71.818 217.19 444.63 5934 1.7944 3.6735	50.092 219.32 467.22 4206 1.8417 3.9233

TABLE C-11. ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, 20% WATER

DYNAHOMETER CONSTANT: 3000 H/C RATIO: 1.78	API	GRAVITY	OF DIES	EL FUEL:	35.3 AT 60F
RUN NUMBER	182.	192.	228 .	242 .	256 .
NOM. WATER PCT.	20.	20.	20	20 .	20 .
ENGINE SPEED RPM	1200.	1200 .	1200	1200 .	1200.
OBS. TORQUE LB-FT	508.	508 .	508	508	508.
BAR PRESS IN-HG DRY BULB DEG F WEI BULB DEG F REL HUMIDITY PCT CUKR BHP HP CORR BMEP PSI	29.28 81. 64. 39. 116.5 45.0	29.16 70. 70. 71. 116.7 45.0	29.06 84. 76. 70. 119.3 46.0	28.96 93. 79. 54. 121.2 46.8	29.12 82. 75. 72. 118.2
FUEL FLOW LB/HR WATER FLOW CC/MIN CALC: VOL. % PCT BSFC Alr FLOW LB/MIN	53.71	53.04	53.92	53.80	54.77
	114.0	111.0	114.0	114.0	111.0
	18.9	18.7	18.9	18.9	18.3
	.4609	.4546	.4521	.4439	.4635
	37.5	37.4	36.7	36.2	36.7
STOICH, F/A	0691	.0691	0691	0691	0691
MLAS: F/A	0239	.0236	0245	0249	0249
CALC: F/A	0249	.0243	0250	0250	0250
% DIFF. PCT	4 36	2.82	1.76	82	35
COOLANT IN DEG F COOLANT OUT DEG F COOLANT OUT DEG F COOLANT OUT DEG F COOLANT OUT DEG F FUEL SUPPLY DEG F FUEL SUPPLY DEG F FUEL SUPPLY DEG F FUEL COOLER DEG F INTAKE AIR (L) DEG F CHARGE AIR (R) DEG F EXH STACK (L) DEG	1111196951344892154420215477645 6611116655883420215477645 6556666655561	11199768177128817445822279421314558107712881744582227942134	7890186666555523168865776259995940 119088655552316886577625995940 561111155586659337995940	178275911059463423675687499854688852 6658499854688852	# 11
OIL PRESSURE PSI RAIL PRESSURE PSI BOOST (R) PSI BUOST (L) PSI IN-H20 EXH. PRESS (R)PSI EXH. PRESS (R)PSI TURB IN (L) IN-HG TURB IN (L) IN-HG FUEL PRESS PSI EHULSTON PRESS PSI EHULSTON PRESS PSI HATER PRESS PSI	557711552	8858 11152	040000000	0555601552	0047-4457
	218020	2123 2120 20	621240 210000	6581240 21021	49-445 A1-0000
	60020	20 20 20	100000	20021	00 6
HYDROCARBONS PPHC CARBON MONOXIDEPPH NITRIC OXIDE PPH NITRIGEN OXIDESPPH CARBON DIOXIDE PCT OXYGEN PCT SMUKE OPACITY PCT	175 263 330 360 53.6 13.6	149 613 275 313 5 1 16 0 3 6	229 221 240 5 3 9 4 0	146. 263. 215. 249. 5.4 15.4	267 267 2880 2855 1916 115
HC MASS GM-HR CO MASS GM-HR NOX HASS GM-HR USHC GM/BHP-HR BSCO GM/BHP-HR USHC GM/BHP-HR	84.052	72.323	71.784	70 145	46.490
	243.65	574.23	212.32	242 91	250.79
	607.14	611.45	514.93	509 42	520.79
	.7212	.6198	6019	5788	520.79
	2.0906	4.9214	1.7802	2 0043	2.1223
	5.2097	5.2405	4.3176	4 2035	4.4018

TABLE C-12. ENGINE TEST RESULTS, CUMMINS ENGINE, 1200 RPM, 25% WATER

DYNAHOMETER CO H/C RATIO: 1.7						35 3 A	1 60f
RUN NUMBER NOM. WATER PCT		193. 25.	229 25	243 25	257 . 25 .		
ENGINE SPEED OBS. TORQUE		1200 508	1200. 508	1200 508	1200 508		
BAR PRESS. DRY BULB WEI BULB REL HUMIDITY CORR BHP CORR BMEP	IN-HG DEG F DEG F PCT HP PSI	29 17 77 70 71 117 1 45 2	29.04 98. 76. 58 119.5 46.1	28.95 94. 80. 55. 121.5 46.9	29.11 82. 73. 65. 117.9 45.5		
FUEL FLOW WATER FLOW CALC: VOL: X BSFC AIR FLOW	LB/HR CC/MIN PCT. LB/BHP-HR LB/MIN	53.33 143.8 22.9 4555 37.2	53.66 159.0 24.6 .4491 36.5	53.75 159.0 24.5 4423 36.0	53.57 159.0 24.6 4544 36.5		
STOICH, F/A MEAS F/A CALC, F/A % DIFF	PCT	0691 0239 0248 3.84	0691 .0245 .0240 -1.99	0691 0249 0252 1.18	.0691 .0244 .0236 -3.42		
COULANT IN COULANT OUT	DECEMBER FFFFFFFFFFFFFFFFFFFFFFFFFFFFFFFFFFFF	111979409259012224802245364531085879049259011775387358092906777855873587555555555555555555555555555	7858277888276239101117753890581150239101155589658067181	1847592265845239758480230342622 9992227774458480230342622 11111999922239758480230342622	1/840 1/840 1/840 1/97 1/97 1/97 1/97 1/97 1/97 1/97 1/97		
OIL PRESSURE KAIL PRESSURE BOOST (R) BOUST (L) INLET VAC (R) EXH PRESS (R TURB IN (R) LURB IN (R) LURB IN (R) LURB IN (L) LURB	PSI PSI PSI PSI IN-H20 )PSI )PSI IN-HG IN-HG PSI PSI PSI PSI	0.6581150 0.6581150 0.000 1.000 1.000 1.000	055520140 401040 010000 60 100 100 100 100 100 100 100 10	05460152 12460152 10020	051251251 210020 21020		
HYDROCARBONS CARBON MONOXID NITRIC OXIDE NITROGEN OXIDE CARBON DIOXIDE OXYGEN SMOKE OPACITY	PPMC EPPM PPM BPPM PCT PCT PCT	250 749 275 325 36 0 9	193 316 216 250, 513 9.5 3.1	211 417 210 250 15 9 15 3	132 383 215 25 15 15 1		
HC MASS LO MASS NOX MASS ESHO ESHO HSNO	GM-HR GM-HR GM-HR GM/HHP-HR GM/BHP-HR GM/BHP-HR	120 00 690 83 657 90 1 0248 5 8998 5 6186	96 372 303 46 555 93 8067 2 5400 4 6525	100 63 381 53 556 28 8281 3 1394 4 5773	66.775 372.81 557.48 5666 3.1626 4.7291		

TABLE C-13. ENGINE TEST RESULTS, CUMMINS ENGINE,  $1800\ \text{RPM}$ , BASELINE

DYNAMOMETER CO H/C RATIO: 1.7	NSTANT: 3000	AP1	GRAVITY	OF DIES	EL FUEL:	35.3 AT	60F	
RUN NUMBER NOM. WATER PCT		171. 0	177	183	188	195.	201	216 0
ENGINE SPEED	RPM LB-FT	1800 1257	1800 1257	1800 1257	1200 508	1800 1257	1800 1257	1800 1257
BAR PRESS DRY BULB WET BULB REL HUMIDITY CURR BHP CORR BMEP	IN-HG DEG F PCT HP PSI	29 29 92 73 65 436 2	29 16 91 73 42 441 7 113 7	28 96 81 76 80 443 9 114 2	29 11 73 68 78 116 2 44 8	29 22 79 74 79 436 1 112 2	29 26 78 72 75 433 9 111 6	29 10 89 76 55 444 3 114 3
FUEL FLOW WATER FLOW CALC: VOL: % BSFC Alk FLOW	LB/HR CC/MIN PCI LB/BHP-HR LB/MIN	181 93 0 0 0 0 4171 83 5	179 14 0 0 0 0 4056 82 6	180 80 0 0 0 0 4073 83 7	55.30 0.0 0.0 4761 38.0	179 19 0 0 0 0 4109 85 0	180.07 0.0 0.0 4151 84.5	177 08 0 0 0 0 3786 81 4
STOICH, F/A MEAS, F/A CALC, F/A % DIFF.	PCT	.0691 .0363 .0335 -7.63	0691 0361 0324 -10.37	.0691 .0360 .0341 -5.24	0691 0242 0248 2 37	.0691 .0351 .0347 -1.15	0691 0355 0348 -1.86	.0691 .0363 .0342 -5.58
COOLANT OUT FUEL RETURN FUEL SUPPLY FUEL GOOLER TURB. INLET (L) COMP. OUT (L) CHARGE AIR (R) COMP. OUT CHARGE AIR (R) CHARGE AIR	DEG F DEG F DEG F DEG F DEG F	781539963571844668866884525516925	1121119967087743055647533939249 192111996743797211055647533939249 19211988199999999999999999999999999999	18687529490202452945194668752999999999999999999999999999999999999	1781 1781 1781 1781 1782 1781 1782 1783 1783 1783 1783 1783 1783 1783 1783	147520597354900020585053008126 11119907221190888302053008126 999999999999999999999999999999999999	11893295821431207880841185710185788888341118579989999999999999999999999999999999	787-07-69-49-54-22-23-15-29-45-19-5-387-83-1 781-159-89-54-22-23-15-29-45-1-25-5-1-25-5-1-25-5-1-28-5-5-1-25-5-1-28-5-5-1-28-5-5-29-5-5-1-28-5-5-29-5-5-2-2-1-28-5-5-29-5-5-2-2-1-28-5-5-2-2-1-28-5-2-2-2-2-2-2-2-2-2-2-2-2-2-2-2-2-2-
OIL PRESSURE KAIL PRESSURE BOOST (A) INLET VAC. (R) EXH. PRESS. (L. IVAN IN (L.) INTER IN (L.) FUEL PRESS. EMULSION PRESS FUEL SUPPLY WATER PRESS.	PSI PSI PSI PSI IN-H20 PSI PSI IN-HG PSI PSI PSI PSI PSI	76 0 0 110 5 3 3 6 0 0 120 20 10 20 0	76. 0 75 3 10 3 11 3 12 7 3 6 0 10 0 10 0	76 0 25 0 3 5 0 0 10 0 20 0 10 0 0 0 0 0 0 0 0 0 0 0	21 4 100 2 100 2 100 2	76 0 2 B 110 B 13 15 20 100 20 20	76 0 75 0 11 1 13 9 13 9 10 0 100 100	744 100 110 11 100 100 100 20 100 20
HYDROCAKBONS CARBON MONOXIDI NITRIC OXIDE NITROGEN OXIDE: CAKBON DIOXIDE OXYGEN SMUKE OPACITY	PPMC EPPM PPM SPPM PCT PCT PCT	100. 306. 900. 863. 7.2 10.5	40 163 950 950 7.0 10.3 0.0	130 635 870 880 7.3 11.5 6.9	54 212 288 313 5 3 15 8	78 281 850 838 7 12 5	70 550 850 72 125 9	82 1288 730 74 150 6
HC MASS CU MASS NOX MASS BSHC BSCO BSNO	GM-HR GM-HR GM-HR GM/BHP-HR GM/BHP-HR GM/BHP-HR	121 64 704 86 3527 4 2789 1 6161 8 0876	49 507 384 81 3789 1 1121 8712 8 5787	154.76 1431.4 3719.8 3486 3.2248 8.3803	26.649 203.00 535.94 2294 1.7475 4.6137	89.849 615.43 3371.8 2060 1.4113 7.7323	81.249 1206.3 3339.2 1873 2.7802 7.6960	95.154 274.58 2949.6 2142 6181 6.6393

TABLE C-13. ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, BASELINE (CONT'D)

DYNAMOMETER CON H/C RATIO: 1.78	STANT: 3000	API	GRAVITY	OF DIES	EL FUEL:	35.3 A1	60F
NUN NUMBER		217	223	231 .	237.	245	251
NOM. WATER PCT.		0	0	0	0	0	0
ENGINE SPEED	RPM	1800	1800	1800	1800	1800	1800
OBS. TORQUE	LB-FT	1257	1257	1257	1257	1257	1257
HAR PRESS. DRY BULB WET BULB REL HUMIDITY CURR BHP CURR BMEP	IN-HG	29, 19	29.24	28 95	29 00	28.94	26 91
	DEG F	80,	89.	83	92.	88.	78
	DEG F	73,	76.	75	80.	76	78
	PCT	73,	55	69	60	58	59
	HP	436, 3	440.9	442 5	448.6	445 3	447.3
	PSI	112, 3	113.5	113 9	115.4	114 6	115.1
FUEL FLOW	LB/HR	179 60	179.37	180.41	180 68	182 23	181 73
WATER FLOW	CC/MIN	0 0	0.0	0 0	0 0	9 0	0 0
CALC. VOL %	PCT	0 0	0.0	0 0	0 0	0 0	0 0
ESFC	LB/BHPHR	4116	4068	4077	4028	4093	4063
AIR FLOW	LB/MIN	83 7	83.1	81.9	81 2	82 1	80 6
STOICH F/A	PCT	.0691	0691	0691	.0691	0671	0691
MEAS F/A		.0358	0360	0367	.0371	0376	0376
CALC F/A		.0343	0350	0336	.0354	0358	0347
% DIFF		-4.12	-2.85	-B 39	-4.51	-3 19	-7 62
COOLANT IN CUGLANT OUT OIL SUMP FUEL SUPPLY FUEL SUPPLY FUEL COOLER 1NTAKE AIK 1NTAKE INLET (R) COMP. OUT (R) COMP. OUT (R) COMP. OUT (R) CHARGE AIK (R) EXH. STACK (L) WATER INLET UELL AIK EXHAUST 3R EXHAUST 3R EXHAUST 5L EXHAUST 6L	DEG F	114680601953566441620487345363 1211 992218878883129087445363 997821877	10602/279874: 188/29036-10/18450-109	781059184622299008842312382120695 78105918462229900884231238211214 999999999999999999999999999999999	458956026586332095854141966106 112111	11211198752112039583874632503 198045198752112039583874632503 1997339921985241874632503	048946098742013222800127253886 112111 1 992211881 9934235062225
OIL PRESSURE KAIL PRESSURE BOOST (R) POUST (L) INLET VAC (R) EXH PRESS (R) EXH PRESS (L) TURB IN (R) TURB IN (L) FUEL PRESS EMULSION PRESS EMULSION PRESS EMULSION PRESS EMULSION PRESS	PS1 PS1 PS1 PS1 PS1 HS1 HG IN-HC IN-HC PS1 PS1 PS1	78 0 2211260110 110 120 100 3 0	77500 14 1100 100 20 100 20	76571500 11000 1000 1000	75 10 10 14 11 19 20 10 20 20	76 0 0 0 1 10 0 0 1 0 0 0 0 0 0 0 0 0 0 0	775006 1190020 10020
HYDROCAKEUNS CARBON MUNOXIDE NITRIC OXIDE NITROGEN OXIDES CANBUN DIOXIDE OXYGEN		61 246 650 738 7 4 14 0	2535 638 638 7 11 9	50 246 605 623 7 12.7	281 675 698 7 6 12 8	250 573 578 7 7 10 3	2845 2845 2860 2860 2860 2872 125
SMUKE OPACITY  HC MASS LU MASS NOX MASS HSHC BSCO RSNU	GM-HR	71 696	63.377	60 .193	69 172	107 48	34.118
	GM-HR	547 33	551.08	561 .61	607 75	538 52	632.33
	GM-HR	2949 4	2483.0	2589 .0	2842 1	2322 1	2372.5
	GM/EHP-HR	1643	.1437	.1360	1542	2414	0763
	GM/EHP-HR	1 2544	1.2498	1.2692	1 3547	1 2094	1.4137
	GM/EHP-HR	6 7594	5.6311	5.8488	6 3354	5 2151	5.3042

TABLE C-14. ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, 5% WATER

DYNAMOMETER CO H/C RATIO: 1.7	NSTANT: 3000	API	GRAVITY	OF DIES	EL FUEL:	35.3 AT	60F
RUN NUMBER NOM. WATER PCT		172 5		196			
ENGINE SPEED OBS TORQUE	RPM LB-FT	1800 1257	1800 1257	1800 1257	1800 1257	1800	1800 1257
BAR. PRESS. DRY BULB WET BULB REL. HUMIDITY CORR. BHP CORR. BHEP	IN-HG DEG F DEG F PCT HP PSI	29 .27 82 . 73 . 65 . 437 .6 112 .6	28.95 94. 82. 60. 453.4 116.7	29 23 79 74 74 437 0 112 5	29 21 80 73 72 72 437 2	28.96 83. 75. 69. 442.9 114.0	28.95 89. 76. 55. 446.6 114.9
FUEL FLOW WATER FLOW CALC: VOL: % BSFC AIR FLOW	LP/HR CC/MIN PCT LB/BHP-HR LB/MIN	179.89 82.8 4.8 4111 82.9	168.98 82.8 5.1 .3727 80.8	178 13 82 8 4 9 4076 83 5	178.26 82.8 4.9 .4078 83.2	179.46 82.8 4.8 4052 81.7	181 31 82 8 4 8 4060 79 7
STOICH, F/A MEAS, F/A CALC, F/A X DIFF	PCT	.0691 .0362 .0326 -9.98	0691 0348 0345 - 98	0691 0355 0343 -3.49	0691 0357 0343 -4.00		0691 0379 0358 -5 61
COOLANT IN COOLANT OUT OIL SUMP FUEL SUPPLY FUEL SUPPLY FUEL COOLER INFAKE AIR INLET (R INFAKE AIR INF	DEG F DEG F DEG F DEG F DEG F DEG F	882631952919529187 882057988329509883748500529114 992221187 999999999999999	1121159378500342394709169175535 1211594795644791089423149169175535 121159178500342394709169175535	247722914354904042595461554679 247722914354904042595961554679	326792127453473243955499160937 1121119722211778882015499160937	1846275013196435307/36238 9922219930999999999999999999999999999999	46025711128552237762615688582378 1182157111285522377762615688582378 279222188 2792224
OIL PRESSURE KAIL PRESSURE BOOST (R) LOOST (L) INLET VAC (R) EXH. PRESS (R EXH. PRESS (L TURB IN (L) FUEL PRESS EMULSION PRESS EMULSION PRESS EMULSION PRESS EMULSION PRESS EMULSION PRESS EMULSION PRESS	PSI PSI PSI IN-H2D PSI PSI IN-HG	758 0 9 7 3 3 5 0 8 11 9 22 1 0 20 20 60	76 07913508 1103 1198 1198 15020	76 0 0 0 0 0 1 1 0 0 0 0 0 0 0 0 0 0 0 0	77 80 11 10 10 13 8 14 6	76 76 10 10 14 15 11	76 0 80 6 10 8 15 8
TURB IN (L) FUEL PRESS EMULSION PRESS FUEL SUPPLY WATER PRESS	IN-HG PSI PSI PSI PSI PSI	100 100 60	9.8 19. 150. 50.	9 8 20 100 50	11.0 10.0 20 100 50	*6 à	15 8 10 55 20 100 20
HYDROCARBONS CARBON HONOXIDI NITRIC OXIDE NITRIGEN OXIDE: CARBON DIOXIDE OXYGEN SMOKE OPACITY	PPMC PPM PPM PPM PCT PCT PCT	110 263 925 875 10 0	150 592 870 915 10.5	95 246 850 875 12.5 4.2	229 742 632 13.4 13.4	219 625 645 13.2	90 219 640 648 77 9 4 2
HC MASS CU MASS NOX MASS BSHC BSCO FSNO	GM-HR GM-HR GM-HR GM/BHP-HK GM/BHP-HR GM/BHP-HR	136.23 619.35 3790.3 3113 1.4154 8.6618	165.36 1231.8 3869.4 3647 2.7169 8.5346	110.81 542.95 3681.6 2535 1.2423 0.4237	50.392 505.92 2606.0 .1336 1.1573 5.9613	65.327 497.47 2774.6 .1475 1.1232 6.2647	102.49 470.61 2544.7 2295 1.0537 5.7870

TABLE C-15. ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, 10% WATER

DYNAMOMETER CONS H/C RATIO: 1.78	TANT: 3000	API G	RAVITY C	F DIESEL	FUEL: 3	5.3 AT 6	0 F
RUN NUMBER NOM, WATER PCT.		173. 10	185. 10.	197. 10.	219. 10.	233.	247 10
ENGINE SPEED R	PM B-FT	1800 1257	1800 1257			1800 1257	1800 1257
DRY BULB D WET BULB D REL HUMIDITY P CORR BHP H	N-HG EG F ECT P SI	29 25 82 73 65 438 5 112 8	28 92 93 80 57 450 9 116 0	29.23 79. 74. 79. 437.0 112.5	29, 23 82, 73, 65, 437, 1 112, 5		28.95 89 76. 55. 447 8 115.2
BSFC L	B/HR C/MIN CT B/BHP-HR B/MIN	179.18 1 175.1 9.7 4087 81.7	179.68 171.8 9.5 3985 80.8	177 73 1 175 1 9 8 4067 82 6	78.31 1 171.8 9.6 4080 82.2	80.7	79 91 175 1 9 6 . 4018 79 1
	PCT	0691 0365 0332 -9 21	0691 0370 0349 -5.76	0491 0358 0348 -2.85	0691 0362 0343 -5.24	.0691 .0367 .0346 -5.67	0671 0379 0369 -2 71
EXHAUST 2L EXHAUST 3L EXHAUST 4L EXHAUST 5L EXHAUST 6L	DEG F DEG F DEG F DEG F	882932065586335578426167271065 182159165586335578426167271065	158468440954221966061643515112 194433990889302913840113 1992211871 99989999971	24792221983961194956325378 112111 1 1992228883961194956325378 119992211983961194956325378	126792034242665074710766828669 19234242665074710766828669 19792211777 999889998889	768911343585750118687253230300653 18342299118687253230300653 1974221177 9998880002	1 1211 1 9922111881 99999999999999999999
OIL PRESSURE KAIL PRESSURE BOOST (R) BOOST (L) INLET VAC (R)	PSI PSI PSI PSI IN-H2O	76 84 10 10 12 13	76. 83.0 10.5 11.0 13.0	76 83 0 10 6 10 8 13 4	77 85 10 10 13 13	76 83 10 10 14 2	76 85 0 10 2 11 15 7
OIL PRESSURE KAIL PRESSURE BOOST (R) FUOST (L) (R) EXH PRESS (R) EXH PRESS (L) TURB IN (L) FUEL PRESS EMULSION PRESS FUEL SUPPLY WATER PRESS HYDROCARBONS	FSI IN-HG IN-HG PSI PSI PSI PSI PSI	10 20 20 20 10 20 10 20	10 9 920 100 50	11 0 9 8 20 100 50	10 8 20 100 50	10 5 20 100 20 100 65	100 20 100 65
HYDROCARBONS CARBON MONOXIDE NITRIC OXIDE NITROGEN OXIDES CARBON DIOXIDE OXYGEN SMUKE OPACITY	PPMC EPPM PPM SPPM PCT PCT PCT	38 476 938 900 7 1 10 5	166 592 930 940 70 10 4 5	91 488 888 900 7 5 12 0	58 212 775 630 7 4 14 0 3 4	193 663 695 7 5 12 8	70 202 683 683 683 8 0 11 8
HC MASS CO MASS NOX MASS BSHC BSCO BSNO	GM-HR GM-HR GM-HR GM/XHP-HR GM/BHP-HR GM/BHP-HR	46.217 1093.0 3978.3 1054 2.4929 9.0734	192.62 1293.9 4194.5 4272 2.8698 9.3034	104 78 1057 5 3892 7 2397 2 4197 8 9068	67,210 469,53 2678,2 1538 1,0720 6,1276	61 831 419 70 2998 4 1395 9471 6 7662	76 989 418 81 2744 6 1719 9353 6 1294

TABLE C-16. ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, 15% WATER

DYNAMOMETER CON H/C RATIO: 1.78	ISTANT: 3000	API	GRAVITY	OF DIESI	LL FUEL:	35 3 AT	60F
RUN NUMBER NOM. WATER PCT.		174. 15.	186 15	198. 15.	220 . 15	234 15	248 15
ENGINE SPEED OBS. TORQUE	RPM LB-FT	1800 1257	1800. 1257.	1800 1257	1800 1257	1800 1257	1800 1257
BAR PRESS. DRY BULB WET BULB REL HUMIDITY CORR BHPP CORR BMEP	IN-HG DEG F PCT HP PSI	29, 22 89, 73, 47, 439, 4 113, 1	28,90 100. 94. 80. 462.6	29 24 92 76 76 438 2 112 8	29.25 86. 76. 63. 440.2 113.3	29.00 67. 77. 64. 444.0 114.2	28.96 95. 77. 44 448.9 115.5
FUEL FLOW WATER FLOW CALC: VOL: % BSFC AIR FLOW	LB/HR CC/HIN PCT LB/BHP-HR LB/HIN	178.39 284.9 14.9 4060 81.0	178.32 288.5 15.0 3855 79.3	176 90 288 5 15 2 4037 81 4	176 82 288 5 15 2 4017 80 2	177 37 277 7 14 6 3995 79 5	179.00 284.9 14.8 .3987 77.6
STOICH, F/A MEAS, F/A CALC, F/A % DIFF.	РСТ	0691 0367 0332 -9.50	0691 0375 0353 -5.76	0691 0362 0348 -3 83	.0691 .0368 .0349 -5.00	.0691 .0372 .0349 -6.09	.0671 .0384 .0372 -3.19
COOLANT IN COOLANT OUT OIL SUMP FUEL SUPPLY FUEL RETUPPLY FUEL COOLER TINES INLET (R) COMP. OUT (R) CHARGE AIR (R) EXH. STACK (R) EXHAUST 3R EXHAUST 3R EXHAUST 3R EXHAUST 5R EXHAUST 5L EXHAUST 5L EXHAUST 5L	00000000000000000000000000000000000000	892023191186338511530324871821 782159182322998709191824871821 14211 1 992211771 989899998899	45947875994323328951158381442 781158192933339998192832538381442 1121111 992221771 999899999999	781157122487781662960682948 7811571861111387508979680497789 11211 1 972211771 8888899998888	4691115276452501846763136962831 78111591812228981846763136962831 12111 1 P922211771 9898899788891	56922536411;33906428484253400277 784315918411;33906428484253400277 12111 1 9922211771 989891529991	4504495655420000086019617186072 782159195477990909311910739113 11211 1 0922011871 90999999999
DIL PRESSURE RAIL PRESSURE BOOST (R) BOOST (L) INLET (AC. (R) EXH. PRESS. (R)	PSI PS1 PS1 PS1 IN-H20 PSI	75. 90.0 10.3 10.9 12.0	76. 88.0 10.2 10.8 12.8	76 03812551 900113 2551	76 90.0 10.4 10.5 13.5	76 88.0 10.2 10.1 14.1	76.9 90.9 10.5 15.4
OIL PRESSURE RAIL PRESSURE BOOST (R) INLE! VAC (R) EXH. PRESS (R) EXH. PRESS (L) TURB IN (L) FUEL PRESS EMULSION PRESS EMULSION PRESS FUEL SUPPLY WATER PRESS	PSI IN-HG IN-HG PSI PSI PSI PSI PSI	100 200 100 60	1900000 100000 100000	18 5 9 1 20 100 50	100 20 100 40	100 100 100 100 100	1000
HYDROCARE'NS CARBON MONDXIDE NITRIC OXIDE NITROGEN OXIDES CARKON DIOXIDE OXYGEN SMUKE OPACITY	PPMC PPM PPM PPM PCT PCT PCT PCT	46. 212. 1018. 783. 7.2 10.3	150 571 988 975 9 8 7 8 7	94 468 950 963 72 12 10	54 186 675 813 7.6 11.9	65. 695. 708. 73.6 13.5	67 163 700 740 8 1 11 6 3 0
HC MASS NOX MASS BSHC BSHC BSHC BSHC BSHC BSHC BSHC BS	GM-HR GM-HR GM-HR GM/BHP-HR GM/BHP-HR GM/BHP-HR	55.774 485.17 4362.4 1269 1.1041 9.9733	171.43 1222.3 6476.6 2.6425 14.009	107.30 1009.5 4423.2 2449 2.3039 10.208	60.978 399.94 3662.0 1385 9086 8.3195	73.797 279.54 3228.9 1662 6296 7.2722	72.733 333.28 3051.5 3424 6.7974

TABLE C-17. ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, 20% WATER

DYNAMONETER CO H/C RATIO: 1 7	NSTANT: 3000	AP I	GRAVITY	OF DIES	EL FUEL	35 3 AT	60F
RUN NUMBER NOM WATER PCT		175	187 20.	199 20	221 20	235 20	249 20
ENGINE SPEED OBS. FORQUE	RPM LB-FT	1880 1257	1800 1257	1900 1257	1800 1257	1800 1257	1800 1257
BAR PRESS DRY BULB WET BULB REL HUMIDITY CORR BHPP CORR BMEP	IN-HG DEG F DEG F PCI HP PSI	29,21 90 73, 44, 440 5 113 4	28.88 98. 84. 56. 455.4 117.2	29 24 82 76 76 76 437 6 112 6	29.25 86. 76. 63. 439.6 113.1	29 03 87 77 64 445 8 114 7	28,95 79 79 45 451 2 116 1
FUEL FLOW WATER FLOW CALC. VOL. % BSFC AIR FLOW	LB/HR CC/MIN PCT LB/BHP-HR LB/MIN	177 48 398 0 19 7 4029 79 6	178 82 394 5 19.4 3927 78 0	176 82 394 5 19 7 4040 80 8	176.64 394.5 19.7 .4018 80.4	175 61 384 0 19 3 3939 77 8	178, 35 374, 5 19, 5 3752 76, 2
STOICH, F/A MEAS, F/A CALC, F/A % D1FF	PCT	0691 0371 0334 -10 00	0691 0382 0356 -6.80	9691 0365 0357 -2.20	0691 0366 0356 -2 77	0691 0376 0358 -4 88	.0691 .0390 .0372 -4 66
STOICH F/A HEAS F/A CALC F/A Z DIFF COOLANT IN CUOLANT DUT OIL SUMP FUEL RETURN FUEL RETURN FUEL RETURN FUEL COOLER TURB INLET (L' COMP OUT (L) COMP OUT (L) COMP OUT (L) COMP INLET (L' COMP INLET (L' COMP INLET (L') COMP I	DEGG F DEGG F DEGG F DEGG F DEGG F DEGG F	7831024216967238424110322368344 783159190122997609988795168344 19922112771 98988999988890	1421111 99221177119898999988999	157882017965670593089318869233 12111 1 88221177 88887588295678 88888888888888888888888888888888888	417175168H18448218651844267242	1 12115 1 90 1222 1 150 2 30 88 9 9 1222 1 150 2 30 88 9 9 9 88 9 9 9 88 9 9 9 88 9 9 9 88 9 9 9 88 9 9 9 88 9 9 9 88 9 9 9 88 9	451530681532124878192832781440 11211111 992211271 787881529982
OIL PRESSURE RAIL PRESSURE BOOST (R) MIDST (R) INLET VAC (R) EXH PRESS (R EXH PRESS (L) TURB IN (L) FUEL PRESS EMULSION PRESS	PSI PSI PSI IN-H20 PSI IN-HG IN-HG PSI	75 97 10 10 11 10 10 10 10 10 10 10 10 10 10	76 98 10 12 10 10 20 100 100	76 96 10 110 12 9 25 1 1 9 20	77 95 0 10 0 13 3 15 10 0 20	76 96 90 10 13 2 10 9 0	76799031500 10900 1000
FUEL SUPPLY WATER PRESS	PSI PSI	60	65.	1 0 0 6 0	1 0 0 60	62 100 20	65.
HYDROCARBONS CARBON MONOXIDE NIRIC OXIDE NIRICOXIDE CARBON DIOXIDE DXYGEN SMUKE OPACITY	PPMC PPM PPM PCI PCI PCT PCT	47 163 1125 1100 7 2 10 2 0 0	144 281 1025 1050 7 8 8 4 0	95 448 1000 1013 7 7 11 9 2 7	55 164 880 705 7 11 8 2 4	56 148 733 770 7 8 12 8 3 3	69 154 688 8 1 11 6 2 8
HC MASS CI MASS NOX MASS ESHC BSCO BSCO BSCO	GM-HR GM-HR GM-HR GM/PHP-HR GM/PHP-HR GM/PHP-HR	55.778 368.94 5115.1 .1266 .8375 11.611	163.08 598.46 5482.2 3581 1.3143 12.039	106 31 942 87 4872 0 2429 2 1545 11 133	61.570 344.31 3304.0 1401 2832 7.5156	61.964 307.07 3601.3 1390 6888 8.0784	74.740 312.68 3149.8 1656 6930 7.0691

TABLE C-18. ENGINE TEST RESULTS, CUMMINS ENGINE, 1800 RPM, 25% WATER

DYNAHOMETER CO	NSTANT: 3000	API	GRAVITY	OF DIES	EL FUEL:	35.3 AT 60	F
RUN NUMBER NOM. WATER PCT		176 25	200 . 25 .	222 25	236 25	250 25	
ENGINE SPEED	RPM LB-FT	1800 1257	1800 1257	1800 1257	1800 1257	1800 1257	
HAR PRESS DRY BULB WET BULB REL HUMIDITY CORR BHP CORR BMEP	IN-HG DEG F DEG F PCT HP PSI	29 19 90 73 44 440 3 113 3	29 .26 81 . 75 . 76 . 436 .9 112 .4	29 25 86 76 63 441 9 113 7	29.03 91 79 59 445.9 114.7	28 92 98 77 39 450 7 116 0	
FUEL FLOW WATER FLOW CALC: VOL: % BSFC Alk FLOW	LB/HR CC/MIN PCT LB/BHP-HR LB/MIN	178.04 478.9 22.8 4044 79.8	176.54 494.2 23.5 4041 80.1	176 64 501 7 23 7 3998 77 7	176 71 501 7 23 7 3963 77 9	179 43 501 7 23 4 .3981 76.1	
STOICH F/A MEAS F/A CALC F/A Z DIFF	PCT	.0691 .0372 .0334 -10.05	.0691 .0368 .0356 -3 00	0691 0379 0358 -5 39	0691 0378 0362 -4 07	06/1 0393 0369 -6 10	
COOLANT OUT COOLANT OUT COOLANT OUT COOLANT OUT COOLANT COOLANT COUT COUT COUT COUT COUT COUT COUT COMP. COM	DECOMPANY FERRER FOR FERRER FER	7830050022552119464343493078454 199022117711 88888899888	11211126327893807983112643278939027397985221879931126457789387554778798522187938798522	10727720144194482201889755550 107277201441944822018975578 1072778	1785929 11149539401988560246897291885602498888892057724	6617427994230227190749775997948 112111117922271900897990317790	
OIL PRESSURE KAIL PRESSURE BOOSI (R) BOOSI (R) FOR FRESS (R) FRESS (R) FOR FRESS (R) FUEL PRESS (R) FUEL FUEL PRESS (R) FUEL FUEL FUEL FUEL FUEL FUEL FUEL FUEL	PSI PSI PSI IN-H20 )PSI IN-HG IN-HG PSI PSI PSI PSI	752 1029 9 7 3500 10 220 10 20 10 60	76 0 10 0 0 10 0 0 12 7 15 10 0 0 10 0 10 0 0 0 10 0 0 0 10 0 0 0 10 0 0 0 10 0 0 0 10 0 0 0 10 0 0 0 10 0 0 0 0 10 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	77 0 107 0 10 0 10 0 10 0 20 1 10 0 20 1	76 0 112 0 7 10 0 6 1 1 1 0 0 0 8 8 1 1 0 0 0 1 1 0 0 0 1 1 0 0 0 1 1 0 0 0 1 1 0 0 0 1 1 0 0 0 1 1 0 0 0 1 1 0 0 0 0 1 0	765995115008 10708111508 10708111508	
HYDROCARBONS CARBON HONDXID NITRIC OXIDE NITRIGEN OXIDE CARBON DIOXIDE OXYGEN SMUKE OPACITY	PPMC EPPM SPPM SPPM PCT PCT PCT	163 1188 1163 10.0 0.0	100 1113 1113 1113 7 7 11 9	55 1770 750 7 5 12 5 12 2	151 788 795 7 9 13 . 3	69: 148: 763: 8:0 11:3	
HC MASS CU MASS NOX MASS HSHC BSCO HSNO	GM-HR GM-HR GM-HR GM/BHP-HR GM/BHP-HR GM/BHP-HR	55.392 370.11 5662.8 1258 6406 12.862	111.82 900.50 5600.0 .2560 2.0613 12.819	61.187 308.49 3712.8 .1385 .6982 8.4027	60.594 311.64 4021.8 .1359 .6989 9.0196	75.734 303.71 3517.1 .1680 .6743 7.8030	

TABLE C-19. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 500 RPM

DYNAMOMETER CON H/C RATIO: 1.8	NSTANT: 2000.	API	GRAVITY	OF DIES	EL FUEL:	33.9 AT	60F	
RUN NUMBER NOM. WATER PCT		1 0	7 0	13.	47	48 5	49 10	50 15
ENGINE SPEED OBS. TORQUE	RPM LB-FT	600 386	600 386	600 386	600. 386.	600. 386.	600 386	600 386
BAR PRESS. DRY BULB WET BULL REL HUMIDITY CORR BHP CORR BMEP	IN-HG DEG F PCT HP PSI	29,26 72, 64, 65, 43,6 16,1	29.30 70. 63. 68. 43.6 16.1	29.07 72. 66. 73. 44.6 16.5	29.02 72. 59. 46. 44.5 16.4	29.03 78. 60. 34. 44.4 16.4	28.99 79. 61. 35. 44.6 16.5	28.94 59. 56. 44. 44.4 16.4
FUEL FLOW WATER FLOW CALC. VOL. % BSFC AIR FLOW L AIR FLOW R	LB/HR CC/MIN PCT LB/BHP-HR LB/MIN LB/MIN	24.89 0.0 0.0 .5712 24.0 24.7	24.41 0.0 0.0 .5604 23.1 23.6	24.60 0.0 0.0 .5515 22.6 23.0	23 82 0 0 0 0 5358 22 5	23.43 11.5 5279 21.2	24 . 22 20 . 6 8 . 7 . 5431 22 . 1	24.60 33.8 13.3 5540 21.2
CUOLANT IN TOUT OUT OUT OUT OUT OUT OUT OUT OUT OU		1784 1794 1794 1798 11983 1198	194 194 199 199 199 199 199 199 199 199	9322.6205888989191804056872421070 33333333333333333333333333333333333	0522716521122231702058466784028 1892231702058466784028 333333333333333333333333333333333333	78413720443434523126552322788879993126552322788879993333333333333333333333333333	7442252054554464014454131196553 111988859798882101196553 3333333333333333333333333333333333	2748925542222331740857445306651 287911178888879779888657445306651 1117888887977988865744530
OIL PRESSURE FUEL SPILL BOOST (RF) BOOST (RR) HOUST (LF) BOOST (LR) AIR BOOST (LR) INLET VAC. (RF INLET VAC. (LR INLET VAC. (L	PPSSS 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	20.00 0.00 0.00 0.00 0.00 0.00 0.00 0.0	18.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.0	19200000555800000000000000000000000000000	19 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	19000000000000000000000000000000000000	90000000000000000000000000000000000000	19000000000000000000000000000000000000

TABLE C-20. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 800 RPM, BASELINE

DYNAMOMETER CONSTANT: 8000. H/C RATIO: 1.82	API	GRAVITY	OF DIES	BEL FUEL:	33.9 AT 60	F
RUN NUMBER Nom. Water PCT.	2. 0.	8. 0.	14. 0.	40. 0.	46 0	
ENGINE SPEED RPM OBS. TORQUE LB-FT	800. 591.	800. 591.	800. 591.	800. 591.	800. 591.	
RAR. PRESS. IN-HG DRY BULB DEG F WEI BULK DEG F REL. HUMIDITY PCT CIRR. BHP HP CORR. BMEP PSI	29.20 72. 64. 65. 89.8 24.9	29 29 74 64 58 89 5 24 8	29.04 74. 68. 74. 91.0 25.2	29.33 71. 58. 45. 88.4 24.5	29.13 79. 62. 38. 90.4 25.0	
FUEL FLOW LK/HR WATER FLOW CC/MIN CALC. VOL. % PCT. RSFC LB/BHP-HR A)R FLOW L LR/MIN AIR FLOW R LB/MIN	43.10 0.0 0.0 4279 31.9 32.7	43.06 0.0 0.0 4813 31.9 32.6	42.87 0.0 0.0 4710 31.0 31.6	42.15 0.0 0.0 4768 31.0 31.1	42 43 0 0 0 0 4692 30 8	
COOLANT IN DEG F COOLANT OUT DEG F COOLANT OUT DEG F COOLANT OUT DEG F COOLANT OUT DEG F FUEL SUMP DEG F FUEL SUMPLY DEG F FUEL SUMPLY DEG F FUEL SUMPLY DEG F FUEL SUMMER AIR (RF)DEG F INTAKE AIR (LF)DEG F INTAKE AIR (LF)DEG F INTAKE AIR (LR)DEG F EXHAUST SAR DEG F EXHAUST SR DEG F EXHAUST SL DEG F	066905999012334255583748861230 189905999012334255583748861230 33333333444	1896 1996 1997 1998 1998 1998 1998 1998 1998 1998	1887571391805553058350108636446 18875713991805553058350108636446 3 333333344344	188555865652400988771210403160625 88755865652400988771210403160625	175562380764400978722082026080044334433333444334443344433444334444344434443444344434444	
OIL PRESSURE PSI FUEL SPILL PSI RUOST (RF) PSI BOOST (RF) PSI BOOST (LF) PSI BOOST (LF) PSI AIR BOX PSI INLET VAC (RF)IN-H20 INLET VAC (LF)IN-H20 INLET VAC (LF)IN-H3 IN (RF) IN-H3 INLET INLET	268 1010 122000000 5	279 00 00 0 0 0 9 9 7 6 0 0 6 8 8 8 0 7 20 3 0 0 0 1 2 2 0 0 0 0 0 0 0 0 0 0 0 0 0 0	22 00000000000000000000000000000000000	65 0 811122001111120030 20030	0.055.05.00.0880.087.77.0 7	

TABLE C-21. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 800 RPM, WITH WATER ADDITION

DYNAMOMETER CONSTANT: 2000. H/C RATIO: 1.82	API	GRAVITY	OF DIES	EL FUEL:	33.9 AT	60F
KUN NUMBER NOM. WATER PCT.	<b>41</b> 5	42 10	43 15	<b>44</b> . 20 :	45 25	
ENGINE SPEED RPM UBS. TORQUE LB-FT	800 571	800. 591	800. 591.	800. 591.	800. 591.	
BAR. PRESS. IN-HG DRY BULB DEG F WET BULK DEG F REL. HUMIDITY PCT CORR. BHP HP CORR. BMEP PSI	29.32 80. 60. 29.8 89.8 24.9	29.30 80. 61. 32. 90.0 24.9	29.26 80. 61. 32. 90.3 25.0	29 21 79 59 29 89 9 24 9	29 17 79 59 29 90 5 25 0	
FUEL FLOW LB/HR WATER FLOW CC/MIN CALC. VOL. % PCT. BSFC LB/BHP-HR Alk FLOW L LR/MIN AIR FLOW R LB/MIN	42.60 19.7 4.7 4742 31.1 30.0	42.83 40.0 9.4 .4760 30.9 30.0	43.28 66.4 14.6 .4791 30.6 30.6	44.19 90.6 18.6 4916 31.6 30.0	43 69 119 3 23 3 24829 30 6 29 9	
CUOLANT IN DEG F COBLANT OUT DEG F COBLANT OUT DEG F COBLANT OUT DEG F COBLANT OUT DEG F COBLANT CAR	188733312654599098884096092618543 11108888899098884096092618543 133333334408122	1887432229967332053346668637541114 1887767888976733333334422 111088888776787541114	1887781788572210856253631199426 18877481788572210856253631199426	058970855249977332064320556979 1887910855249977332064320556979 11173320643320556979	78754038866822293368443294445179 11199888880688767660179 11113 33333344334	
OIL PRESSURE PSI FUEL SPILL PSI BUOST (RF) PSI BUOST (RR) PSI BUOST (LF) PSI BUOST (LF) PSI BUOST (LR) PSI INLET VAC. (RF)IN-H20 INLET VAC. (RR)IN-H20 INLET VAC. (LF)IN-H20 INLET VAC. (RF)IN-H20 INLET VAC. (RF)IN-H20 INLET VAC. (LF)IN-H20 INLET VAC. (LF)IN-H20 INLET VAC. (LF)IN-HG TURB. IN. (RF) IN-HG TURB. IN. (LF) PSI FUEL SUPPLY WATER PRESS. PSI	265 0 8112200089880 0 1030 0	60000000000000000000000000000000000000	67 0 8111220011110040 20 5	00500000000000000000000000000000000000	0 8-1-12200-1-1-1-3000 67 0 8-1-122000-1-1-1-30000 22 5	

TABLE C-22. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, BASELINE

DYNAMOMETER CONSTANT: 8000. H/C RATIO: 1.82	AP I	GRAVITY	OF DIE	SEL FUEL:	33.9 A	7 60F			
RUN NUMBER NOM. WATER PCT.	3. 0.	9 . 0 .	15. 0.	19. 0.	25. 0.	51. 0.	57. 0.	65. 0.	7 <u>i</u> 0 .
ENGINE SPEED RPM OBS. TORQUE LB-FT	1000. 877.	1000. 877.	1000 877	1000. 877.	1000. 877.	1000. 877.	1000. 877.	1000 877	1000 H77
BAR PRESS. IN-HG DRY BULB DEG F WET BULB DEG F REL HUMIDITY PCT CORR BHP HP CORR BHP PSI	29.17 72. 64. 65.1 167.0 37.0	29.26 72. 64. 65. 165.9 36.7	28.99 74. 68. 74. 169.6 37.6	29.33 73. 66. 66. 166.4 36.9	29.25 73. 67. 73. 168.4 37.3	29.29 62. 55. 64. 162.9 36.1	29.15 68. 62. 166.2 36.8	29.07 62. 62. 76. 168.2 37.3	29.00 71. 64. 69. 167.2 37.0
FUEL FLOW LB/HR WATER FLOW CC/MIN CALC: VDL. % PCT: BSFC LB/BHP-HR Alr FLOW L LB/MIN AIR FLOW R LB/MIN	72.43 0.0 0.0 4336 40.7 41.8	72.17 0.0 0.0 4351 40.7 42.0	71.76 0.0 0.0 4230 39.8 40.8	72.25 0.0 0.0 4341 40.9 41.5	72.43 0.0 0.0 4301 39.9 40.2	71.91 0.0 0.0 .4416 41.4 41.6	73.17 0.0 0.0 4403 41.0 40.8	71.97 0.0 0.0 4278 39.6 40.2	71.32 0.0 0.0 4265 41.0 39.9
COOLANT IN DEG F COOLANT OUT DEG F COOLANT OUT DEG F FUEL SUPP DEG F FUEL SUPPLY DEG F FUEL COOLER (RF) DEG F INTAKE AIR (RR) DEG F INTAKE AIR (LF) DEG F INTAKE AIR (LR) DEG F INTAKE AIR (LR) DEG	9512290119123332246996725551584 78002808878000477756766977903 1120211 1 1 1 1 4 4 4 4 4 5 4 4 4 5 5 5	7809289888821100577864867097013 780928988821100577864867097013 112 1	96191789923117436099576444353778092178992311174360995764443537	7.800.160.884.1267.651.9666.164.250.653.1888.110.067.7866.44.250.888.110.114.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.653.144.7866.164.250.144.	282018999660109792423160490718 1880211899888111109792423160490718 11111478474444554555	11890232882320804195670725477 789028077669999676855670725477 474444554555	1890286877004021552286232790987747642 189028088770099777856685229245 11990280887747642	1848791688138964639637453516398 1997797799996477563518846 199779778637453516398	42800809977889990952982128860443 111111 1289777799990952982128860443 147775522607443
OIL PRESSURE PSI FUEL SPILL PSI BOOST (RF) PSI BOOST (RR) PSI BUOST (LF) PSI BUOST (LF) PSI BUOST (LF) PSI BOOST (RR) PSI INLET VAC. (RF)IN-H20 INLET VAC. (RF)IN-H20 INLET VAC. (LF)IN-H20 INLET VAC. (LF)IN-H20 INLET VAC. (LF)IN-H20 INLET VAC. (LF)IN-H20 INLET VAC. (RF)IN-H20 EXH. PRESS. (R)PSI IURB IN. (RF)IN-HG IURB IN. (LF)IN-HG	34.00 11.00 10.00	35711111211330001 B0 7	3477111141143000111110030 20030	35.6.6.12.0.14.114.30.0.37.7.7.7.0.5 20.30.0.30.0.30.0.30.0.30.0.30.0.30.0.3	0000000065990044440 2 35	95.	35511000000550900111430 7	00000000000000000000000000000000000000	355100101111400553350030 101111400553350030

TABLE C-23. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, 5, 10, 15% WATER

DYNAMOMETER CONS H/C RATIO: 1.82	STANT : 2000.	API	GRAVITY	OF DIES	SEL FUEL:	33.9 A	106 1			
RUN NUMBER NOM, WATER PCT.		20	52. 5.	66 . 5	21 10	53 10	67 . 10 .	22 15	54 15	68 . 15 .
ENGINE SPEED R OBS. TORQUE L	RPM .B-FT	1000. 877.	1000 877	1000 877	1000 822	1000	1000. 877.	1000 877	1000 877	1000 877
	N-HG DEG F DEG F OCT HP 'SI		29.30 62. 55. 64. 163.2 36.2	29.08 67. 62. 76. 167.1 37.0	29.33 73. 66. 69. 168.6 37.3	29.28 68. 58. 55. 164.3	29.09 72. 66. 73. 167.9 37.2	29.31 72. 66. 73. 168.8 37.4	29.25 68. 58. 55. 164.5 36.4	29 .09 72 66 73 167 .9 37 .2
FUEL FLOW L WATER FLOW C CALC. VOL. % P RSFC L A)k FLOW L AIR FLOW R L	B/HR C/MIN CT B/BHP-HR B/MIN B/MIN	72.55 33.8 4.9 .4314 39.8 40.7	71 96 32.3 4.8 .4408 41.4 41.5	71.68 30.8 4.6 4290 40.4 40.2	73.11 66.4 9.2 .4337 39.5 40.1	72.06 69.7 9.7 4385 40.8 41.1	72.35 66.4 9.3 4309 40.5 40.3	73.34 111.0 14.4 .4345 39.5 40.1	72.52 109.55 14.4 .4409 40.9 41.3	72.37 111.0 14.6 1310 40.2 40.0
COOLANT IN DO COOLANT OUT DO UTL SUMP DO FUEL IN DEFUEL SUPPLY DEFUEL COOLER (RE) DINTAKE AIR (LE) DINTAKE AIR (LR) DINTAKE AIR (LR) DINTAKE AIR (LR) DHP AIR (RR) DHP AIR (RR) DHP AIR (LR) DEEXHAUST 2R DEEXHAUST 2R DEEXHAUST 4R DEEXHAUST 5R DEEXHAUST 5L DEEXHAUST 5L DEEXHAUST 5L DEEXHAUST 6L DEEXHAUST 5L DEEXHAUST 6L DEEXHAUST 5L DEEXHAUST 6L DEEXHAUST 6	50000000000000000000000000000000000000	1784989776666457887964927602619757	17897 17897 18897 17897 18897 18897 18897 18897 18897 18897 1889 1889	188019089558644407424373028310149 1887708877855864455554555	7808089999888800007567080828999888000075670800999998888844446005529926	180188504441133343880162589968659	74809706588450865670715335174809706588450865670715335174452297348	283987855666689356598606596548 11271898888000048888598606596548 11124844466596544901	1887 1887 1887 1887 1887 1887 1887 1884 1884	789877955884408196361223574988 11199677096361223574988 111944444455344653
OIL PRESSURF PO		35.0 36.0 1.0 2.0	9000000055781021136 45100101135 035566	3310000056000011020 0 451100010111440055550050	33 33 33 33 33 33 33 33 33 33 33 33 33	4551000101054891020130 0	0000000561000011000 0 5511001101144000333500050	90000000055080075670 7 37 37 200550055080075670 7	35 10000055990011130 7 35 100191133003355003	35 100 100 10 10 10 10 10 3 10 10 10 10 10 10 10 10 10 10 10 10 10

TABLE C-24. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, 20, 25% WATER

DYNAHOMETER CONSTAN	IT: <b>2000.</b> AP.	I GRAVITY	OF DIESE	L FUEL:	33.9 AT	60F
RUN NUMBER NOM, WATER PCT.	23 20	55. 20.	69 . 20 .	24 25	56. 25.	70 25
ENGINE SPEED RPM OBS. TORQUE LB-F	1000 . 877	1000 877	1000 877	1000. 877.	1000 877	1000. 877.
BAR. PRESS. IN-H DRY BULB DEG WET BULK DEG REL. HUMIDITY PCT CORR. BHP HP CORR. BMEP PSI	1G 29.29 F 72. F 66. 73. 167.6	29 .21 60 59 . 165 .0 36 .5	29.09 72. 66. 167.9 37.2	29 27 72 . 66 . 167 . 7 37 . 1	29 . 18 69 . 69 . 165 . 2 36 . 6	29.04 72. 66. 73. 168.2 37.3
FUEL FLOW LB/H WATER FLOW CC/M CALC: VOL. % PCT. BSFC LB/B AIR FLOW L LB/M AIR FLOW R LB/M	R 73.83 IN 159.0 19.0 HP-HR .4405 IN 40.1 IN 40.3	73.68 155.9 17.1 .4466 48.8 41.5	73.02 155.92 194.9 40.7	73.91 208.0 23.9 .4407 39.9 40.3	73.77 209.0 23.9 .4467 41.0 41.2	73.53 211.0 24.2 4372 40.0 40.2
	9516586770199449506333618576729 78091898880000388075555985890 11111488075555985890	628648776325553650372525559251 78991897777999944 7443433009251 1111 1897777799994 4444435555555555	74075677697697696103113092461 74075677697697696103113092461 74075677697697696103113092461	112 18988880743030115833935277 78091898880000388755833935277 1112 18988881000388755833935277	199777753360373739056605 199777753360373739056605	630647676095629016513526446103 78091199888700095876545446007612 1112 1 4 44444554455
UIL PRESSURE PSI FUEL SPILL PSI BOOST (RR) PSI BOOST (LR) PSI BUOST (LR) PSI BUOST (LR) PSI INLET VAC (RF) IN-H INLET VAC (RF) IN-H INLET VAC (LF) IN-H EXH PRESS (L) PSI TURB IN (RF) IN-H TURB IN (RF) IN-H TURB IN (RF) IN-H TURB IN (LR) IN-H TURB	00000000000000000000000000000000000000	35 10010414500555500040	35 10 00 00 00 00 00 00 00 00 00 00 00 00	58 12010141330003335005	356-10-0-0-0-5-69-9-0-1-1-20-2-2-2-2-2-2-2-2-2-2-2-2-2-2-2-	35 461000000000000000000000000000000000000

TABLE C-25. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, FUEL INJECTION TIMING RETARDED 2.4 DEGREES

DYNAMOMETER CONSTANT: 2000. H/C RATIO: 1.82	API	GRAV1TY	OF DIES	SEL FUEL:	33.9 A	1 60F	
RUN NUMBER NUM. WATER PCT.	72	78 0	<b>73</b> . 5 .	74 10	75 15	76 20.	27 25
ENGINE SPEED RPM UBS: TURQUE LB-FT	1000 827	1000 877	1000 877	1000 877	1000 877	1000	1600 877
BAR PRESS IN-HG DRY BULB DEG F WET BULK DEG F REL HUMIDITY PCT CURR BHP HP CORR BMEP PSI	29.23 74. 62. 51. 166.2 36.8	29 21 82 60 26 1 166 8 36 9	29.25 74. 62. 166.5 36.9	29 25 80 61 32 166 4 36 9	29 24 80 61 32 166 7 36 9	29 32 89 61 61 32 61 166 4 36 9	29 23 86 61. 32 166 5 36.9
FUEL FLOW LE/HR WATER FLOW CC/MIN CALC. VOL. % PCT. BSFC LE/BHP-HR AJK FLOW L LE/HIN AIR FLOW R LE/MIN	72 29 0 0 0 0 4349 40 8	72.23 0.0 0.0 4331 40.1 40.7	71.72 32.3 4.8 4307 40.7	72.17 68.1 9.5 4337 40.4 40.5	72.32 109.5 14.4 .43.3 40.3 40.7	72.64 157.4 19.4 4365 40.3	73.22 208.0 24.0 4397 40.4 40.5
COOLANT IN DEGGEFF FOR THE PROPERTY OF THE PRO	118998877087669867190868610053318418 1899887708766986778876633135667190868610053318418	840021833562387772982806382793 1780021833562387772982806382793 1110888884444663313555555	748990900109921778031039803256 7899179998800007778557552303355 11114 444455555555	7380909092333312533340828204224587 1189099923333125333408282042245387 1111144 873445542245387	76387607324441264651113558690000 118991999881126465111355869000 11114 44445574555	9507.60.65.45.623.86.842.22.27.57.93.52.44 78091.999.88811.00.68875.45.45.85.3812.34 1111.148875.45.45.53.45.55	110051540655443444445534555 11005133337357245 111051345454454555555
OIL PRESSURE PSI FUEL SP(LL PSI BUOST (RF) PSI BUOST (RR) PSI BUOST (LF) IN-H20 INLET VAC. (RF) IN-H20 INLET VAC. (LF) IN-H20 INLET VAC. (LF) IN-H20 INLET VAC. (LF) IN-H20 INLET VAC. (LF) IN-H20 INLET VAC. (LPSI EXH. PRESS. (L)PSI EXH. PRESS. (L)PSI TURB. IN. (RF) IN-HG TURB. IN. (LF) IN-HG TURB. IN. (LF) IN-HG TURB. IN. (LR) IN-HG FUEL PRESS. PSI EMULSION PRESS PSI EMULSION PRESS PSI EMULSION PRESS PSI HATER PRESS. PSI	00000045500000111200 05511101211144000375500000	000000056100011120 54111010111440033350030 20030	35 35 35 36 37 37 37 37 37 37 37 37 37 37 37 37 37	35411010111010 S 3541101011111010 S 3541101011111010 S	35 1 1 0 1 2 1 1 4 4 0 0 3 3 3 5 0 0 3 0 5 0 5 0 5 0 5 0 5 0	334110121114400333330030	451101611179400879790000 35

TABLE C-26. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, FUEL INJECTION TIMING RETARDED 4.1 DEGREES

DYNAHDMETER CO H/C RATIO: 1.8	NSTANT : 2000.	AP1	GRAVITY	OF DIES	FL FUEL:	33.9 AT	60F	
KUN NUMBER NOM WATER PCT		79 . 0 .	8 <b>5</b> . 0 .	80 . 5.	81 10	82. 15.	83 20	84 25
ENGINE SPEED OBS. TURQUE	RPM LB-FT	1000 877	1000 877	1000 877	1000 877	1000 877	1000 877	1000 877
HAR PRESS DRY BULB WET BULB REL HUMIDITY CURR BHP CORR BMEP	IN-HG DEG F DEG F PCT HP PSI	29.04 80. 70. 61. 170.3	28.93 94. 70. 30. 172.6 38.2	29.06 80. 70. 61. 170.6 37.8	29.06 83. 71. 56. 170.6 37.8	29.03 83. 71 56 171 2 37.9	29.00 84. 71. 53. 172.6 38.2	28.96 92. 71. 35. 172.8 38.3
FUEL FLOW WATER FLOW CALC. VOL. % BSFC AIR FLOW L AIR FLOW R	LB/HR CC/MIN PCT- LB/BHP-HR LB/MIN LB/MIN	72.58 0.0 0.0 4263 39.3 40.4	72.52 0.0 0.0 4201 39.6 39.5	72.09 33.8 4.9 4226 39.1 39.8	72 93 69 7 96 4275 39 1 39 8	73.51 108.0 14.0 4293 40.7 39.7	73.82 154.4 18.8 4276 38.0 39.5	74.57 211.7 23.9 .4315 39.5
COOLANT IN COOLANT OUT COOLANT	00000000000000000000000000000000000000	847435399559996324027463492224 111102908888100698807684634922224 11111111114 5448478	78786000000141496502796245882 78978600000141496502796245882 58479	1848547522662208236483017243279 1848547522662208236483017243279 1111114 44444555555	749759655884519386037061695346 119029099881111088899637061695356 1111111 1 44475555555	840741655333464028441533032878 1800201009999111189888441533032878 1111114 444455535555	7495234005689063542241400336879 111000999112179986565229134 1111149988455229134	74041441167009794694340208920374041441167009794694340208920397945552189203
UIL PRESSURE FUEL SPILL MOUST (RF) BOOST (RR) BUOST (LR) AIK BOX INLET VAC (RF INLET VAC (LF INLET VAC (LF INLET VAC (LR LXM. PRESS (LL TURB IN. (RF) TURB IN. (RF) TURB IN. (LF) FUEL PRESS EMULSION PLES WATER PRESS	PPROSENT COO 99999999999999999999999999999999999	5551100005551100111020 0 5551100121114400535530050	355110101114400333336030 2030	0000000005500000110000 0	0000000056010011020 2 551100101114400555536030	000000055210011020 5 351100121114400533336030	555110000056210000010 101011440005753360530 20050	00000000056210000010 7

TABLE C-27. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, FUEL INJECTION TIMING ADVANCED 2.8 DEGREES

DYNAMOMETER CO H/C RATIO: 1.8	NSTANT: 2000.	API	GRAVITY	OF DIES	EL FUEL:	33.9 AT	40 <i>F</i>	
RUN NUMBER		93.	99.	94.	95.	96	97 .	98
NOM. WATER PCT		0.	0.	5.	10.	15	20 .	25
ENGINE SPEED	RPM	1000.	1000.	1000.	1000.	1000	1000	1000
OBS. TORQUE	LB-FT	877.	877.	877.	877.	877	877	
BAR PRESS	IN-HG	29.24	29.04	29.22	29.20	29.16	29 12	29.09
DRY BULB	DEG F	72.	85.	72.	80.	80.	84	84
WET BULK	DEG F	60.	65.	60.	61.	61.	64	64
REL HUMIDITY	PCT	49.	33.	49.	32.	32.	33	33
CURR BHP	HP	165.1	169.8	166.0	166.5	167.4	168 7	168 9
CURR BMEP	PSI	36.6	37.6	36.8	36.9	37.1	37 4	37.4
FUEL FLOW	LB/HR	72.32	71.68	72.35	72.91	73.59	74 10	74.60
WATER FLOW	CC/MIN	0.0	0.0	30.8	68.1	111.0	157 4	208.0
CALC: VOL: %	PCT	0.0	0.0	4.5	9.4	14.3	19 1	23.7
BSFC	LB/BHP-HR	4382	.4222	4357	4380	4397	4392	.4416
AIK FLOW L	LB/MIN	41.0	39.0	40.9	40.5	40.0	39 4	39.4
AIR FLOW R	LB/MIN	40.7	39.2	40.3	40.5	39.8	39 6	39.8
COOLANT IN COOLANT OUT OIL SUMP FUEL SUPPLY FUEL RETURN FUEL SUPPLY FUEL COOLER INTAKE AIR (RF INTAKE AIR (LF I	00000000000000000000000000000000000000	738684654554476122301386554611 18988770099778651386554611 4445554611	1899445355124442608522615747357 1899445355124442608522615747357 1111114 988522615747357 4445545555	748117987006721966261173345456 11190117987006721966261173345456 111114 4444554555	## 112101000099229943249612760902038	852202122450076372456235540414 780029099881100688754544118124 111114 4444445555	7520830008901109686366323071656 1121011909089011058887543071656 111114844444455855	#52984922002221196757338621185 7809199999911111588654544007008 111114 44444554555
OIL PRESSURE FUEL SPILL BOOST (RF) BOOST (RR) BOOST (LR) AIR BOX INLET VAC (RF INLET VAC (LR INLET V	PSSI 1 2200 PSSI 1 2200 PSSI 1 + H200 PSSI 1 + PSSI 1 + PS	33 100121114300313300 2	33 11001211144003333300000 20030	44 100000055000010010 4 44 1000121114400555500000000000000000000000	44100000000000000000000000000000000000	44 100000056900010010 5 35 1000121154005555N0050	44100010111440000000000000000000000000	4441001040144400333330030 35

TABLE C-28. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM, FUEL INJECTION TIMING ADVANCED 5.5 DEGREES

DYNAMOMETER CO H/C RATIO: 1.8	NSTANT : <b>2000</b> .	API	GRAVITY	OF DIES	EL FUEL:	33.9 AT	60F	
RUN NUMBER NOM, WATER PCT		100. 0.	106. 0.	101. 5.	102. 10.	103	104. 20.	105 25
ENGINE SPEED OBS. TORQUE	RPM LB-FT	1000 877	1000 877	1000. 877.	1000 877	1000. 877.	1000. 877.	1000
BAR. PRESS. DRY BULB WET BULB REL. HUMIDITY CORR. BHP CORR. BMEP	IN-HG DEG F DEG F PCT HP PSI	28.98 75. 69. 74. 169.4 37.5	28.95 82. 70. 55. 171.5 38.0	28.99 75. 67. 74. 170.2 37.7	29.00 75. 78. 169.9 37.6	29.00 82. 70. 55. 170.8 37.8	28.99 82. 70. 55. 170.6 37.8	28.97 82. 70. 55. 171.0
FUEL FLOW WATER FLOW CALC. VOL. % BSFC AIR FLOW L AIR FLOW R	LB/HR CC/MIN PCT LB/BHP-HR LB/MIN LB/MIN	73.41 0.0 0.0 .4333 40.0 40.1	72.85 0.0 0.0 4247 38.7 39.2	73.77 32.3 4.6 4334 39.5 39.9	74.04 68.1 9.3 4357 39.7 40.2	75.47 112.5 14.2 .4419 40.9 39.8	76.11 160.6 19.0 4460 39.2	76.50 215.5 23.8 4474 39.0 40.1
COOLANT IN COOLANT OUT OIL SUMP FUEL SUMP FUEL RETPLY FUEL COOLER INTAKE AIR (LR INTAKE AIR (LR INTAKE AIR (LR INTAKE AIR (LR INTAKE AIR INTAKE INT		740460333981319988659195288367 11211 1 4 445545235	18024564013221121121114888440183306987	1782453455214542692707041014120 12102112119188881100426927070441014120 112104534552145355	18021321345773487230672667770863 190221345773487230672667770863 1121024 1111148 4455441170863	## 120 122 28 6 6 4 2 4 6 3 3 2 8 6 1 0 1 2 2 1 1 1 1 1 1 6 8 8 7 6 5 5 4 5 5 5 5 5 5 6 6 6 6 6 6 6 6 6 6	853304399998900385553756863820 180029088888900385553756863820 111148887653756863820	8542952122302331961461685372977800190797971111148887655449574757
UIL PRESSURE FUEL SPILL BUOST (RF) BUOST (RF) BUOST (LF) BUOST (LF) BUOST (LF) AIK BOX AC (RF INLET VAC (RF) INLET VAC	PPS1112200 PPS1114220 PPS1114220 PPS114220 PPS1214220 PPS1214231 PPS1214231 PPS121431 PPS121431 PPS121431 PPS131	33 10000055100010110 0 35 10001015110 0 35 1000101110 0	34.00000550000010 121114.000000000000000000000000000000000	34 1000121114 40033535000 20000000000000000000000000000	34 1 0 0 0 0 0 0 0 1 0 1 0 0 1 0 0 1 0 1	34 10001041144003333340 2 20330 5	33 10001011111440003333340330 4	33 33 33 34 35 35 35 35 35 35 35 35 35 35 35 35 35

TABLE C-29. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1200 RPM, BASELINE

DYNAMOMETER CO H/C RATIO: 1.8	NSTANT: 2000	L AP	I GRAVIT	Y OF DIE	SEL FUEL	: 33.9 AT	60F
RUN NUMBER NOM, WATER PCT		<b>4</b> : 0 :	10. 0.	16.	26 . 0 .	32. 0.	
ENGINE SPEED OBS. TORQUE	RPM LB-FT	1200 1229	1200 1229	1200. 1229.	1200 1229	1200 1229	
BAR PRESS. DRY BULB WET BULK REL HUMIDITY CUKR BHP CORR BMEP	IN-HG DEG F DEG F PCT HP PSI	29.16 75. 65. 281.3 51.9	29.25 72. 64. 65. 280.1	28.96 75. 70 78. 286.1 52.8	29.31 79. 61. 34. 277.9 51.3	29.18 72. 62. 52. 282.4 52.1	
FUEL FLOW WATER FLOW CALC. VOL. % BSFC AJR FLOW L AIR FLOW R	LB/HR CC/MIN PCT LB/BHP-HR LB/MIN LB/MIN	115.57 0.0 0.0 4108 52.1 51.8	115.85 0.0 0.0 .4136 50.5 52.9	115.13 0.0 0.0 4023 49.1 50.2	115.76 0.0 0.0 4166 50.8 52.3	115 11 0 0 0 0 4077 49 4 52 2	
COOLANT IN COOLANT OUT COOLANT OUT COOLANT OUT COOLANT OUT COOLANT FUEL SUPPLY FUEL IR FUEL SUPPLY FUEL SUPPLY FUEL SUPPLY FUEL SUPPLY FUEL SUPPLY FUEL AIR (RF INTAKE AIR (LF HP AIR (LF) EXHAUST 1R EXHAUST 2R EXHAUST 3R EXHAUST 5R EXHAUST 5L EXHAUST 5L EXHAUST 5L EXHAUST 5L EXHAUST 5L	DEGG FFFFFFFFFFFFFFFFFFFFFFFFFFFFFFFFFFF	0862551563423227876778881445880 112111577866778881445880 11211157786778881445880	976133011007909586838353081869 11211115779780872293349 1121115779780872293349	197243144 188012831444 18888841115 1888881112288 112248805837701115808	188 188 188 198 198 198 198 198 198 198	75492100955555555555555665666112 1 1 11115 655555665666	
OIL PRESSURE FUEL SPILL MUDST (RF) BOOST (RR) BOUST (LF) AJR BOX INLET VAC. (RF INLET VAC. (RF INLET VAC. (LR I	PS1 PS1 PS1 PS1 PS1 PS1 PS1 PS1 PS1 VIN-H20 VIN-H20 VPS1 VPS1 VPS1 VPS1 VPS1 VPS1 VPS1 VPS1	2601110110101015500000000000000000000000	00000000077600044440 1 350011000077600044440 1 20000	001005057650035990 5 25501106005500550035990 5 44	45211272255002222050 40000000000000000000000000	44 44 44 10 10 10 10 10 10 10 10 10 10 10 10 10	

TABLE C-30. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1200 RPM, WITH WATER ADDITION

DYNAMUMETER CONSTANT: 80 00 H/C RATIO: 1.82	D. AP	I GRAVIT	Y OF DIE	SEL FUEL	: 33.9 AT	60F
NUN NUMBER NOM. WATER PCT.	27. S.	28 10	29 . 15 .	30. 20.	31 25	
ENGINE SPEED RPM OBS. TORQUE LB-FT	1200 1229	1200. 1229.	1200 1229	1200 1229	1200 1229	
BAR PRESS. IN-HG DRY BULB DEG F WHT BULB DEG F REL HUMIDITY PCT COKR BHP HP CURR BMEP PSI	29.31 79. 61. 34. 280.0 51.7	29 . 29 72 . 62 . 57 . 280 . 5 51 . 8	29 . 25 72 . 62 . 57 280 . 9 51 . 9	29,20 75, 64, 55, 283,2 52,3	29.18 72. 62. 282.4 52.1	
FUEL FLOW LE/HR WATER FLOW CC/MIN CALC. VOL. % FCT BSFC LE/BHP-HR AIR FLOW L LE/MIN AIR FLOW R LE/MIN	116.26 49.8 4152 4152 50.7	115.70 114.0 9.9 .4124 50.9 51.8	116.43 175.1 14.3 .4144 50.8 51.6	116.22 215.5 17.1 .4104 51.0 51.1	116.43 366.4 25.9 .1123 49.4 51.2	
CUOLANT IN DEG F COOLANT OUT DEG F COOLANT OUT DEG F COOLANT OUT DEG F COOLANT OUT DEG F FUEL SUMP FUEL SUMP FUEL RETURN DEG F FUEL RETURN DEG F FUEL RETURN DEG F FUEL COOLER (RR)DEG F INTAKE AIR (RR)DEG F INTAKE AIR (LR)DEG F INTAKE AIR (LR) DEG F INTAKE AIR (LR) DEG F HP AIR (RR) HP AIR (LR) HP AIR (LF) HP AIR (LF) HP AIR (LF) HP AIR (LF) EXHAUST 1R DEG F EXHAUST 1R DEG F EXHAUST 1R DEG F EXHAUST 1L DEG F EXHAUST 5R DEG F EXHAUST 1L DEG F EXHAUST 5L DEG F EXHAUST 5L DEG F	966234232448921502950146698761 7800280998822224789779988028224 112111 1 5555566566666666666666666666666	966234487442321600732450164329 1888888221600732450164329 188788888779164329	97/632247/643232911405487047114792 1121121111115 5555556566666	9261925888434205444714137494799 1120111818888434205444714137494799 555555556565656	8565520009563422942164470947189780918098882222188464655705080112 1 1 111115 5555565656	
OIL PRESSURE PSI FUEL SPILL PSI BUOST (RF) PSI BUOST (RR) PSI BUOST (LF) PSI BUOST (LF) PSI AJR BOX (RF) IN-H20 INLET VAC (RF) IN-H20 INLET VAC (LF) IN-H20 INLET VAC (LF) IN-H20 INLET VAC (LR) IN-H6 TURB IN (RR) IN-H6 TURB IN (LR) IN-H6 TURB IN (LR) IN-H6 TURB IN (LR) IN-H6 TURB IN (LR) IN-H6 FUEL PRESS PSI EMULSION PRESS PSI EMULSION PRESS PSI EMULSION PRESS PSI EMULSION PRESS PSI	99991110702155000000000000000000000000000000	000500067660035550 4 3501112702550035550030 44	0005050677500333330 5 4421127225500333330030 20 5	000505050567500333330 0 4442112722255003333330 0 2040	00000000000000000000000000000000000000	

TABLE C-31. ENGINE TEST RESULTS. DETROIT DIESEL ENGINE, 1400 RPM, BASELINE

DYNAMUMETER CONSTANT: 2006 H/C RATIO: 1.82	). AP]	GRAVIT	Y OF DIE	SEL FUEL	- 33.9 A	1 60F	
RUN NUMBER NOM, WATER PCT.	5. 0.	11.	17	33. 0.	39. 0	58 0.	5 <b>4</b> 0
ENGINE SPEED RPM OBS. TORQUE LB-FT	1400. 1654.	1400 1654	1400. 1654.	1400. 1654.	1400 1654	1400 1654	1400 1654
BAR PRESS IN-HG DRY BULB DEG F WHT BULB DEG F REL HUMIDITY PCT CORR BHP HP CORR BMEP PSI	29.30 67. 62. 76. 435.0 68.8	29.12 74. 67. 70. 446.5 70.6	29.17 75. 64. 55. 444.1 70.2	29.17 75. 62. 48. 442.5 70.0	29 . 23 21 . 59 . 437 . 5 69 . 2	29 22 65 59 70 431 1 68 2	29.15 68 64 81 438 6 69 4
HUEL FLOW LEYHR WATER FLOW CC/MIN CALC. VOL. % PCT. BSFC LEYBHP-HR AIR FLOW L LEYMIN AIR FLOW R LEYMIN	177.76 0.0 0.0 4087 66.3 68.6	177 08 0 0 0 0 3965 64 8 67 4	177 47 0 0 0 0 3997 66 0 68 4	176 47 0 0 0 0 3988 66 3 69 0	177 44 0 0 0 0 4056 66 0 69 7	176 38 0 0 0 0 4091 66 8 69 5	176 47 0 0 0 0 4023 64.7 68.7
COOLANT IN DEG F COOLANT OUT DEG F COOLANT OUT DEG F COOLANT OUT DEG F FUEL SUMP DEG F FUEL SUPPLY DEG F FUEL COOLER DEG F INTAKE AIR (RF)DEG F INTAKE AIR (LF)DEG F ENHALE (LF)DEG F HP AIR (RF) HP AIR (RF) HP AIR (RF) HP AIR (LF)DEG F EXHAUST 1R DEG F EXHAUST 2R DEG F EXHAUST 3R DEG F EXHAUST 5R DEG F EXHAUST 3L DEG F EXHAUST 5L DEG F	871461567440117799040862874280 871461567440117799040862874280 667766677677	991572443666739452682800681095 11211 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	769560600993473352433140012238 1121181998855553789920983504548 11116 6677683504548	11211281300554443918175112130088029 112118130055444391817511116 867767777	11210464075772273640291174229 1121028075772273687767777 11210464075772273667767777	978025921873249372058934930662 97880259218732493720589777777777777777777777777777777777777	11211279732533294010503178792197 11211279797755336779911787777777777777777777777777777777
OIL PRESSURE PSI FUEL SPILL PSI HUOST (RF) PSI BUOST (RR) PSI HUUST (LF) PSI BUUST (LF) PSI BUUST (LR) PSI INLET VAC. (RF)IN-H20 INLET VAC. (RF)IN-H20 INLET VAC. (LF)IN-H20 INLET VAC. (LP)IN-HG INLET VAC. (LP)IN-HG INLET INLET IN-HG INLET	0004575464400 NUNNOSSO 5	18433484478 NANGE 30	17453484488 6676030 2030	0000050455209887750 5 074334044880099999030	5443340448908079980 9 544334044890807998020	000005000133331154446 5 94433424488 99990050	54433434488 99900040 1040

TABLE C-32. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1400 RPM, WITH WATER ADDITION

DYNAMOMETER CONSTANT: 2000 H/C RATIO: 1.82	. AP	I GRAVIT	Y OF DIE	SEL FUEL	.: 33.9 A	T 60F			
RUN NUMBER NOM. WATER PCT.	34. 5.	59 5	35. 10.	60. 10.	36. 15.	61 . 15 .	37 . 20 .	62 . 20 .	38 . 25 .
ENGINE SPEED RPH OBS. TORQUE LB-FT	1400 1654	1400. 1654.	1400 1654	1400. 1654.	1400. 1654.	1408. 1654.	1400. 1654.	1400. 1654.	1400. 1654.
BAR PRESS IN-HG DRY BULB DEG F WET BULB DEG F REL HUMIDITY PCT CURR BHP HP CORR BHEP PSI	29.18 75. 62. 48. 442.9 70.1	29, 23 65, 60, 75, 433, 2 68, 5	29.19 75. 62. 48. 442.2 70.0	29.23 67. 62. 76. 434.3 68.7	29.19 75. 61. 44.2 69.8	29.21 68. 62. 72. 435.6 68.9	29.18 76. 63. 48. 441.0 69.8	29.19 68. 62. 72. 437.1 69.1	29.18 72. 59. 46. 435.2 68.8
FUEL FLOW LB/HR WATER FLOW CC/MIN CALC. VOL. % PCT BSFC LB/BHP-HR AIR FLOW L LB/MIN AIR FLOW R LB/MIN	175.30 82.8 5.0 .3958 65.2 69.0	177.80 81.2 4.8 4105 66.1 69.9	176.64 173.4 9.6 .3995 64.5 68.7	175.95 170.1 9.7 .4051 65.8 69.5	176.73 292.1 15.5 4006 64.8 68.9	176.82 270.5 14.5 4059 65.4 69.3	176.38 418.9 20.9 4000 65.1 69.2	176.13 380.5 19.4 .4030 64.9 68.5	177 25 501 7 23 9 4073 65 1 70 0
COOLANT IN DEG F COOLANT OUT DEG F COOLANT OUT DEG F COOLANT OUT DEG F FUEL SUPP FUEL RETURN DEG F FUEL COOLER INTAKE AIR (RR)DEG F INTAKE AIR (LR)DEG F INTAKE AIR DEG F EXHAUST 1R DEG F EXHAUST 2R DEG F EXHAUST 5R DEG F EXHAUST 5L DEG F EXHAUST 5L DEG F EXHAUST 6L DEG F	1184156333095555530782901201290660 1184156333095555530782901201290660 11553788901201290660 11553788901201290660	18279087322004361775695187148306677777777777777777777777777777777777	18604533324445507721322128979741 1816281998855507221322128979741 11554298882212897949741	1187977777778187878787878787878787878787	1180022309 1180022309 1180022309 11257453025651946635 11141698977779376635	43898543254830888992225332697 18898543254830888992225332697 144830888992225332697	1401002809999236263822216989999231144057822221698999992311440578222216989999923114405782222169899999231144057822221698999992311440578222221698999999999999999999999999999999	175489 18099 18399 18399 1845 174439 1899 1899 1899 1899 1899 1899 1899 18	7-6-1-98-88-7-7-8-1-9-8-7-5-5-5-5-5-5-5-7-8-1-1-9-8-7-5-5-5-5-5-5-5-5-5-5-7-7-7-7-1-1-2-7-1-1-1-1-1-1-1-1-1-1-1-1
OIL PRESSURE PSI FUEL SPILL PSI BUOST (RF) PSI BUOST (RK) PSI BUOST (LF) PSI BUOST (LF) PSI BUOST (LF) PSI BUOST (LF) PSI BUOST (LF) PSI BUOST (LF) PSI INLET VAC. (RF) IN-H20 INLET VAC. (LF) IN-H3 INLET INLET	000000004455800000000005 0447334044880000000000000000000000000000	00000000044414455450 3 54443434458 955500050 20 51	000005035420087760 5 0544340448800359990030 20 5	5155444343444888 977790030 20030	00000050354200087550 5 05444340448800999990050 20050	0005000024311154440 2 54443434488 99990030	0000000046300076550 5	000050054001145N300 N	000005045100076050 5 0544340448800999990030 10 5

TABLE C-33. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1400 RPM, FUEL INJECTION TIMING RETARDED 4.1 DEGREES

DYNAHOMETER CO	NSTANT: 2000	AP:	I GRAVIT	Y OF DIE	SEL FUEL	: 33.9 A	T 60F	
RUN NUMBER		86.	92 .	<b>87</b> .	88.	89.	90	91
NOM. WATER PCT		0.	0 .	5.	10.	15.	20	25
ENGINE SPEED	RPM	1400.	1400.	1400.	1400.	1400.	1480.	1400.
OBS. TORQUE	LB-FT	1654.	1654.	1654.	1654.	1654.	1654.	1654.
BAR PRESS. DRY BULB WEI BULK REL HUMIDITY CUKR BHP CORR BMEP	IN-HG	29.49	29.43	29.50	29.50	29 . 48	29.46	29,45
	DEG F	74.	80.	74	74.	77 .	77.	80,
	DEG F	52.	54.	52.	52.	64 .	64.	54,
	PCT	18.	13.	18.	18.	49 .	49.	13,
	HP	430.8	436.0	431.8	432.4	437 . 8	439.3	433,5
	PSI	68.2	69.0	68.3	68.4	69 . 3	69.5	68,6
FUEL FLOW	LB/HK	180 .81	181.54	180 .36	179.78	178.66	178.31	178 .44
WATER FLOW	CC/MIN	0 .0	0.0	82 .8	171.8	277.7	387.5	522 .4
CALC: VOL: %	PCT	0 .0	0.0	4 .9	9.6	14.7	19.4	24 .6
BSFC	LB/BHP-HR	.4197	4164	.4177	4153	4081	4059	.4117
AJR FLOW L	LB/MIN	68 .6	66.7	68 .1	67.9	67.2	66.7	66 .7
AIR FLOW R	LB/MIN	70 .9	70.3	70 .8	70.4	69.6	69.7	70 .0
COOLANT IN COOLANT OUT COOLANT OUT COOLANT OUT COOLANT OUT COOLANT OUT COOLANT OUT COOLANT COO		1180 1181 1200 1121 1120 1121 1120 1121 1121	1180734 11201124 1120	112011248898888855577776765571111111111111111111111	1187 1187 1187 1187 1188 1188 1188 1188	112000 1120000 112000 112000 112000 112000 112000 112000 112000 112000 1120000 112000	1120112060144556622756894649294 6483060118099856622756894649294 667127777777777777777777777777777777777	11675555667616090645695736635 112 1189798855544090645695736635 1112 1189798855544090645695736635
OIL PRESSURE FUEL SPILL HOUST (RF) BOUST (RF) HOUST (LF) BOUST (LF) BOUST (LF) AIR BOX INLET VAC (RF INLET VAC (LF INLET VACS (RF INLET VACS (RF INLET VACS (LR EXH PRESS (RF) IURB IN (RF) IURB IN (RF) IURB IN (LF) FUEL PRESS EMULSION PRESS FUEL SUPPLY WATER PRESS	PSSSI 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	517.4333464488 1099960030	5444444644BB 99992030	16443464488 99996030 54235	5154443464488 9999603 510443464488 9999603	545443464488 99996030 515443464488 99996030 51030	54544346448B 99996030 600550065331165440 0	00000000000000000000000000000000000000

TABLE C-34. ENGINE TEST RESULTS, DETROIT DIESEL ENGINE, 1600 RPM, BASELINE

DYNAHOMETER CONSTANT: 2000 H/C RATIO: 1.82	. AP	I GRAVIT	Y OF DIESE	L FUEL:	33.9 AT	60F
RUN NUMBER NOM, WATER PCT.	6 . 0 .	12. 0.	18. 0.			
ENGINE SPEED RPM OBS. FURQUE LB-FT	1600 2143	1600 2143	1600 2143			
BAR. PRESS. IN-HG DRY BULB DEG F WEI BULB DEG F REL. HUMIDITY PCT COKR. BHP HP CORR. BMEP PSI	29.31 75. 65. 648.1 89.7	29.11 78. 69. 64. 660.1 91.4	29.17 75. 64. 55. 660.7 91.5			
FUEL FLOW LE/HR WATER FLOW CC/MIN CALC. VOL. % PCT. BSFC LB/BHP-HR AIR FLOW L LE/MIN AIR FLOW R LB/MIN	259.27 0.0 0.0 4001 85.7 93.1	260.12 0.0 0.0 3940 82.0 91.5	260 37 0 0 0 0 3941 84 4 91 9			
COOLANT IN DEG F F COOLANT OUT DEG F F COOLANT	822 : 817 : 829 :	841 835 844	835. 837. 848.			
JIL PRESSURE PSI FUEL SPILL PSI BOOST (RF) PSI BOOST (RR) PSI BOOST (LR) PSI BOOST (LR) PSI AIK BOX PSI INLET VAC. (RF) IN-H20 INLET VAC. (LF) IN-H20 INLET INLET INLH20 INLET INLET INLH20 INLET	50787817822 22420030 10030	557777827832 44440030 2030	60777828723 44440030 55777828723 44440030			

TABLE C-35. PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE,  $600\ \text{RPM}$ 

DYNAHOMETER CON	STANT : <b>3000</b> .	API	GRAVITY	OF DIES	L FUEL:	33.9	AT	60F
RUN NUMBER NOM. WATER PCT.		142.	143					
ENGINE SPEED OBS. TORQUE	RPM LB-FT	600 385	600 385					
BAR PRESS DRY BULB WET BULB REL. HUMIDITY CURR BHP CORR BMEP	IN-HG DEG F DEG F PCT HP PSI	29.13 88. 74. 52. 45.0 16.6	29.12 90. 73. 44. 45.3 16.7					
FUEL FLOW WATER FLOW CALC. VOL. % BSFC AIR FLOW L AIR FLOW R	LB/HR CC/MIN PCT. LB/BHP-HR LB/MIN LB/MIN	24.10 0.0 0.0 .5351 22.9 23.3	24.32 10.3 4.5 5374 23.0 23.2					
SIDICH. F/A HEAS. F/A CALC. F/A Z DIFF.	PCT	.0689 .0087 .0072 -17.48	0689 0088 0072 -18 37					
COOLANT IN COOLANT OUT COOLANT OUT OUT COOLANT OUT COIL SUMP FUEL SUMP FUEL SUMPLY FUEL COOLER (RE) INTAKE AIR (LE) INTAKE AIR LEXHAUST 3R EXHAUST 3R EXHAUST 3R EXHAUST 5R EXHAUST 5R EXHAUST 5R EXHAUST 5L		149719519216739388953119504248 1889719519216739388953119504248 111 11 1 3 332322732334	3083601926590632038210422688204 879063203926210422688204 111111111111111111111111111111111111					
OIL PRESSURE FUEL SPILL BOOST (RF) BOOST (LF) BOOST (LF	PPSSI PPSSI PPSSI PPSSI PPSSI PPSSI PPSSI PPSSI PPSSI PPSSI PPSSI PPSSI PPSSI	10 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	18 0 3 3 0 0 3 5 4 4 9 8 0 0 8 6 7 9 8 0 0 8 6 7 9 8 0 0 8 6 7 9 8 0 0 8 6 7 9 8 0 0 8 6 7 9 8 0 0 0 8 6 7 9 8 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0					
HYDROCARBONS CARBON MONOXIDE NITRIC DXIDE NITROGEN OXIDES CARBON DIOXIDE UXYGEN PARTICULATE		421 302 361 191 191	491 186 354 186 186 5					
HC MASS CO MASS NOX MASS BSHC BSCO BSNO	GM-HR GM-HR GM-HR GM/BHP-HR GM/BHP-HR GM/BHP-HR	310 260 951 7 06 5 91 21 62	366 276 913 9 31 6 27 20 75					

TABLE C-36. PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE, 800 RPM

DYNAMUMETER CO	NSTANT : <b>3000</b>	AP1	GRAVIT	Y OF DIE	BEL FUEL	33.9 A	100 T	
RUN NUMBER NOM WATER PC		114	120	115. S	116 10	117 15	118	119 25
ENGINE SPEED OBS TORQUE	RPH LB-FT	800 592	800 592	800 592	800 592	800 592	800 542	800 392
BAR PRESS DRY BULB WET BULR REL HUMIDITY CORR BHP CORR BMEP	IN-HG DEG F DEG F PCT HP PSI	29 22 88 77 61 92 4 25 6	29 14 99 78 39 93 8 26 0	29 22 89 77 58 92 6 25 6	29 21 91 78 56 92 9 25 7	29 21 78 78 93 1 25 8	29 19 78 47 93 5 25 9	29 18 99 78 19 93 9 26 0
FUEL FLOW WATER FLOW CALC VOL X BSFC AIR FLOW L AIR FLOW R	LB/HR CC/HIN PCT LB/BHP-HR LB/MIN LB/HIN	42 95 0 0 0 0 4650 31 0 30 9	43 25 0 0 4612 30 9 30 6	43 18 17 3 4 3 4665 30 8 30 9	43 69 46 5 10 5 4702 30 4 30 8	43 80 61 4 13 4 4704 36 4 30 7	44 10 94 0 19 1 4715 30 2 30 6	44.81 124.5 23.5 4773 30.5
STOICH F/A MEAS F/A LALC F/A Z DIFF	PCI	0689 0116 0101 -12 64	0689 0117 0100 -14 39	0689 0117 0102 -13 02	0689 0119 0102 -13 97	0689 0119 .0103 -14.05	0689 0121 0103 -14 35	0689 0123 0104 -15 23
COOLANT 10UT COOLANT 10UT COOLANT 10UT FUEL SINF 1 FUEL COOLER (REF FUEL COOLER (REF 11N AKE AIR (REF 11N AK	DED CONTROL OF THE FEE FEE FEE FEE FEE FEE FEE FEE FEE F	102355147 102355147 10211187 10211188 10211188 10211188 10211188 10211188 10211188 10211188 10211188 10211188 10211188 102188 102188 102188 102188 102188 102188 10218 10218 10218 10218 10218 10218 1	163 127 187 187 164 1103 1107 1105 1107 1116 1116 1116 1116 1116 1116 1116	174 1813 1165 1165 1164 95 95 1165 1164 95 1165 1164 1164 1165 1164 1165 1164 1165 1164 1165 1164 1165 1165	183 189 200 108 167, 1107 107 107 107 108 109 108 108 108 108 108 108 108 108 108 108	173. 184. 184. 163. 164. 164. 164. 167. 167. 167. 167. 167. 167. 167. 167	1810655647 1021024 1021104 1104 1104 1104 1104 110	102724 102724 102721 10271 102
TURB IN (RR) TURB IN (LF) TURB IN (LF) TURB IN (LR) FUEL PRESS. EMULSION PRESS FUEL SUPPLY WATER PRESS.		00570808870845570 3	005007389880065660 2 75 10 1 2200011119030	225 0 1 22000 1 1 1 1 8 8 3 0 0 4 3 1 8 8 3 0 0 1 1 1 1 1 8 8 3 0 0 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	005887788770045570 3	00580238888808655570 3	0058845888880045560 3	0057073888880075560 3
HYDROCARBONS CARBON MONOXID NITRIC DXIDE NITRICEN OXIDE CARBON DIOXIDE UXYGEN PARTICULATE	PPMC EPPM PPM PPM PCT PCT MG/SCF	114 108 405 446 16 16 10	376 108 428 469 201 16.7	4856 1859 1859 161 161	532 172 145 103 15 15 15 3	463 193 376 453 18 18 2	484 2328 3528 111 112	393 494 236 292 171
HC MASS CO MASS NOX HASS BSHC BSCO BSNO	GM-HR GM-HR GM-HR GM/BHP-HR GM/BHP-HR GM/BHP-HR	386 199 1588 4 30 2 21 17 61	356 202 1541 3 95 2 24 17 09	455 1572 1572 1572 1572 179	500 520 1492 5 55 16 55	436 359 1685 4 83 3 98 18 70	455 466 1474 5 05 5 18 16 35	373 750 1101 4 14 8 32 12 22

TABLE C-37. PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE, 1000 RPM

DYNAHUHETER CO	NSTANT : <b>3000</b> .	API	GRAVITY	OF DIES	SEL FUEL:	33.9 A	1 60F	
RUN NUMBER NON. WATER PCT		107.	113. 0.	108	109	110	20	112
ENGINE SPEED OBS. TURQUE	RPM LB-FT	1000 877	1000	1000 877	1000 877	1000 877	1000 877	1006 877
BAR PRESS DRY BULD WET BULL REL HUMIDITY CORR BHP CORR BHEP	IN-HG DEG F DEG F PCT HP PSI	29 23 87 77 6 172 0 38 1	29.12 100. 77. 35. 174.4 38.6	29 22 92 77 51 171 8 38 0	29 21 93 77 49 172 0 30 1	29.18 95 78 47 172.7 38.3	29,16 95, 77, 44,1 174,1 38,6	29 14 98 76 173 3 38 4
FUEL FLOW WATER FLOW CALC. VOL. % BSFC AIR FLOW L AIR FLOW R	LB/HR CC/MIN PCT LB/BHP-HR LB/MIN LB/MIN	73.26 0.0 0.0 4259 40.3 40.1	73.38 0 0 8.0 4207 38 7 39 5	74.14 32.3 4.6 4316 40.3 39.6	73 65 66 4 9 1 4283 40 2 39 6	74.50 109.5 14.0 4314 39.5	74.53 155.9 18.6 4281 37.0 39.2	74 01 211 7 24 1 4270 39 3 39 4
STOICH. F/A MEAS. F/A CALC. F/A % DIFF.	PCT	.0689 .0152 .0140 -7.72	.0689 .0156 .0138 -11.52	0689 0155 0139 -9 95	0689 .0154 .0138 -10.17	0689 0157 0139 -11.11	0689 0159 0139 -12.43	0689 0157 0139 -11 43
HP AIR (RE) HP AIR (RE) HP AIR (RE) EXHER INLET CELLAUST 1R EXHAUST 1R EXHAUST 3R EXHAUST 5R EXHAUST 5R EXHAUST 5R EXHAUST 5R EXHAUST 5L EXHAUST 3L EXHAUST 3L EXHAUST 5L EXHAUST 5L	######################################	30597.26992440063602183180597.269924400636021831835590537.111111449928444435590537.	11111111111111111111111111111111111111	7899024800999712065120653538749321111144 99848733174932	310094958195629373801597278912 1001094958195629373801597278912 110114996473921108013 44744554555	120855720227840565046577 7800855720227840565046577 110082111081112759504444757846602	173 120 1257 107 107 107 107 107 107 107 107 107 10	7100627402110241102411024794102947940808
OIL PRESSURE FUEL SPILL BUOST (RF) BUOST (RF) BUOST (LF) ALR BUOY BUOY BUOY BUOY BUOY BUOY BUOY BUOY	PROPERTY OF THE PROPERTY OF T	431118111114730030W750030	######################################	431101011114400330000000000000000000000	760801845110000918 0 331101011144000330136050	800901854110809915 3311011111440897009915 80390	Bookers Jideooke Jio o Philosophysia o Philosophysia o No h	9000011550000099800 7 75170010111111111111111111111111111111
HYDROCARBONS CARBON MONOXID NITRIC UXIDE NITROGEN OXIDE CARBON DIOXIDE UXYGEN PARTICULATE	PPMC EPPM PPM PPM PCT PCT MG/SCF	422 80 549 576 2 9 15 0	507 94 578 619 2 9 15 8	652 119 527 563 2 9 16 2	720 136 491 543 2 15 15 15	610 145 500 524 16.0	588 145 481 530 15.5 15.3	546. 164. 444. 492. 2.9 16.3
HC MASS CO MASS NUX MASS BSHC BSCO BSNO	GM-HR GM-HR GM-HR GM/BHP-HR GM/BHP-HR GM/BHP-HR	488 180 2513 2 92 1 08 15 05	594 217 2418 3.56 1.30 14.48	767 275 2431 4.59 1.65 14.56	848 314 2390 5 08 1 88 14 25	723 335 2526 4 33 25 13	697 336 2445 4 17 2 81 14 64	644 378 223 3 86 2 26 13 31

TABLE C-38. PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE, 1200  $\ensuremath{\text{RPM}}$ 

DYNAMOMETER CO	NSTANT : <b>3000</b>	. AP:	I GRAVIT	Y OF DIE	SEL FUEL	: 33.9 A	T 60F		
RUN NUMBER NOM. WATER PCT		121. 0.	127 0	144. 0.	150. 0.	122	145. 5.	123. 10.	146 10
ENGINE SPEED OBS. TORQUE	RPM LB-FT	1200 . 1231 .	1200 1231	1200. 1231	1200 1231	1200 1231	1200 1231	1200	1200 1231
BAR PRESS. DRY BULB MEI BULK REL HUMIDITY CORR BHP CORR BMEP	IN-HG DEG F DEG F PCT HP PSI	29.20 89. 77. 58. 289.4 53.4	29.11 101 78. 36. 294.2 54.3	29.11 96. 74. 35. 298.8 53.7	28.96 102. 73. 24.6 54.4	29.19 91. 77. 53. 290.4 53.6	29.06 108 76 33 293.2 54.1	29 19 92 78 54 291 1 53 7	29.84 102. 77. 32. 294.7 54.4
FUEL FLOW WATER FLOW CALC: VOL: % BSFC AIR FLOW L AIR FLOW R	LB/HR CC/MIN PCT: LB/BHP-HR LB/MIN LB/MIN	114.65 0 0 0 0 3962 495 51.8	116 46 0 0 3959 50 9 52 0	115.06 0.0 0.0 395.2 50.2	116.46 0.0 0.0 3953 49.5 51.6	115.76 53.1 4.8 3987 49.9 51.7	116.22 53.1 4.8 3964 49.4 52.1	116 18 109 5 3992 50 6 52 0	116.69 112.5 9.6 .960 50.7 51.6
STOICH F/A MEAS F/A CALC F/A Z DIFF	PCT	8689 0189 0175 -7 22	8689 8189 8175 -7.44	0689 0188 0173 -7 55	.0689 .0192 .0173 -9.97	0689 0190 0176 -7.28	0689 0191 0175 -8.22	0689 0189 0176 -6 52	0689 0190 0177 -7.07
CELL AIR 1 EXHAUST 1 EXHAUST 1 EXHAUST 3 EXHAUST 5 EXHAUST 5 EXHAUST 1 EXHAUST 1 EXHAUST 1 EXHAUST 3 EXHAUST 3 EXHAUST 3 EXHAUST 5 EXHAUST 5 EXHAUST 5 EXHAUST 5 EXHAUST 5 EXHAUST 5 EXHAUST 5 EXHAUST 6 EXHAUST 6 EXHAUST 6	DODD DODD DE EFFERENCE DE LE FERENCE DE LE F	75400663255539491884276730340 78013700999223268995553555666640 1121115 5555555666666	7.79.5.2.14.10.11.11.11.11.11.11.11.11.11.11.11.11.	8644017719322222468663727992266667	89121294810210967838857122 112113084811124197170967854851552 15555555666667	2142110924766661131111111111111111111111111111111	976181809652138102342060775765 1111111111111111151155555566656666	1784521301156099889939975999467467992311111111111111111111111111111111111	000-000-000-000-000-000-000-000-000-00
OIL PRESSURE FUEL SPILL BUDST (RF) BUDST (RF) BUDST (LF) BUDST (LF) BUDST (LF) AIR BUX	PSI PSI PSI PSI PSI PSI PSI PSI PSI PSI	70011110477600111718000	500811257090003345 0	BBBRUNUBANABBBRANANABBRB	900 300 00 00 00 00 00 00 00 00 00 00 00	44 NATIONAL AND	44	80033342568700035470 8	BATTON TO A
HYDROCARBONS CARBON HONOXID NITRIC OXIDE NITROGEN OXIDE CARBON DIOXIDE OXYGEN PARTICULATE	PPMC EPPM PPH SPPM PCI PCI MG/SCF	513 83 674 692 3 7 15 8	510 83 681 714 3.7 16.0	560 91 720 752 3 15	522 84 756 775 3 15 8	791 99 648 676 3 152	774 88 727 764 3 6 14 8	768 98 644 685 3 7 15 5	758 94 763 796 15 15 15
HC MASS CO MASS NUX MASS BSHC HSCO BSNO	GM-HR GH-HR GM-HR GM/BHP-HR GM/BHP-HR GM/BHP-HR	716 3673 13 86	755 3542 2 60 12 60	822 3577 2 92 12 72	777 351 321 12 51	1154 3434 12 42	1139 3754 4 05 13 35	1123	1107 4010 4010 14 26

TABLE C-39. PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE, 1200 RPM, 15, 20, AND 25% WATER

DYNAMUHETER CONSTANT: 3000 H/C RATIO: 1.82	. AP	1 GRAVIT	Y OF DIE	SEL FULL	33.9 A	1 60F
RUN NUMBER NOM. WATER PCT.	124. 15.	147. 15.	125 20	148 20	126	149 25
ENGINE SPEED RPM OBS. TORQUE LB-FT	1200 1231	1200 1231	1200 1231	1200	1200 1231	1200 1231
BAR PRESS. IN-HG DRY BULB DEG F WHIT BULB DEG F REL HUMIDITY PCT CORR BHP HP CORR BHEP PSI	29.17 92. 78. 291.6 53.8	29.02 103 777 31.2 295.5	29.17 92. 80. 59. 293.6 54.2	29.00 105. 75. 24.7 294.7	29 14 97 79 45 293 7 54 2	28.79 105. 75. 24. 295.2
FUEL FLOW LB/HR WATER FLOW CC/MIN CALC VOL X PCJ BBFC ABFC LB/HP-HR AIR FLOW L LB/HIN AIR FLOW R LB/HIN	117 12 180 1 1415 4016 50 4	116.69 180.1 14.6 3954 50.3	117.34 252.3 19.2 .3994 48.4 50.5	117.46 252.3 1986 50.4 51.2	118 13 341 / 24 2 4023 49 6	117.72 341.7 24.3 3987 49.4 51.5
STOICH F/A HEAS F/A CALC: F/A Z DIFF: PCT	9689 0191 0177 -7 58	.0689 .0191 .0175 -8.56	.0689 .0198 .0178 -9.78	.0689 .0193 .0176 -8.73	.0689 .0194 .0180 -7.59	.0689 .0194 .0178 -8.56
LOOLANT IN DEG F COOLANT OUT DEG F COOLANT OUT DEG F COOLANT OUT DEG F FUEL SUMPP FUEL IN DEG F FUEL RETURN DEG F FUEL SUPPLY	174 184 1107 1109 1109 1109 1109 1109 1109 1109	89-2-17-382-6-0-8-8-3-4-3-6-3-4-18-6-7-5-38-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-	76818307043115609963945130092155	6829:53637:11999:3037:52446530008601111111111111511555555566666666666	904954871135230552045494245140117 18009000000334330055355329176017 1111111111111151155555556566666	5826323480900724086873314044287 6700200101044444308534732741181 1121111111111111111111111111111111
OIL PRESSURE PSI FUEL SPILL PSI MOUST (RF) PSI BOOST (RF) PSI INLET VAC (RF)IN-H20 INLET VAC (LF)IN-H20 INLET VAC (LF)IN-H20 INLET IN (RF) IN-HG IUMB IN (RF) IN-HG IUMB IN (LF) IN-HG IUMB	00047.45.687.00.757.00.0 2.500.10.00.00.00.00.00.00.00.00.00.00.00.0	44 NOT TO THE RESIDENCE OF THE PROPERTY OF THE	15001010150055555500 5 1400101010105005555500 5 20 5	2442N-124 NN555005555550 7 0 442N-124 NN555000555550 7 0 1 4 4 4 2 N 5 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1	00061-10569700-101500000000000000000000000000000000	34444141800046666015151515000 0 5
HYDROCARBONS PPHC CARBON HONOXIDEPPH NITROCEN OXIDE PPH NITROGEN OXIDESPPH CARBON DIOXIDE PCT DXYEN DIOXIDE PCT PARTICULATE MG/SCF	785 96 969 706 14 B 1 7	686 748 789 3 6 15 0	789 94 644 677 3 7 15 4 1 8	6/0 89 701 737 3 7 16 0	685 623 658 3.7 15.0	685 84 654 694 3 7 15 3
HC MASS GM-HR CO MASS GM-HR NUX MASS GM-HR BSHC GM/BHP-HR BSHO GM/BHP-HR BSHO GM/BHP-HR	1154 274 4148 4 11 97	1018 263 4127 3 62 14 68	1155 268 4364 4 11 95 15 52	992 258 3759 3 53 13 37	1002 258 4110 3.56 14.62	889 241 3674 3 16 86 1 3 06

TABLE C-40. PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE, 1400  $\ensuremath{\text{RPM}}$ 

DYNAHONEJER CON	STANT : 3000	. AP	I GRAVIT	Y OF DIE	SEL FUEL	: 33.9 A	1 60F	
RUN NUMBER Nom. Water PCT.		128. 0	134. 0.	129 . 5 .	130	131 15	132 20	133 25
ENGINE SPEED OBS TORQUE	RPM LB-FT	1400. 1654.	1400 1654	1400. 1654.	1400 1654	1400 1654	1400	1400 1654
BAR PRESS DRY BULB WET BULB REL HUMIDITY CORR BMPP CORR BMEP	IN-HG DEG F DEG F PCT HP PSI	29 22 87 77 64 453 3 71 7	29 14 102 76 30 459 3 72 7	29 21 89 77 58 453 7 71 8	29 21 94 78 49 457 5 72 4	29 20 94 78 49 458 8 72 6	29.19 94. 78. 49.0 72.6	29 17 100 77 35 458 7 72 6
FUEL FLOW WATER FLOW CALC: VOL: % BSFC AIR FLOW L AIR FLOW R	LB/HR CC/MIN PCT LB/BHP-HR LB/MIN LB/MIN	178.17 0.0 0.0 3931 63.9 68.0	178.39 0.0 0.0 .3884 64.6 66.5	178.39 81.2 4.8 3932 65.5 68.3	177.76 173.4 97.3886 62.3 67.9	177 95 288 5 15 2 3879 61 9	178.22 391.0 19.5 3883 61.9 65.9	179 37 517 5 24 2 3911 63 3 66 0
STOICH. F/A MEAS. F/A CALC. F/A Z DIFF.	PCT	0689 0225 0170 -24 56	0689 0227 0171 -24 57	0689 0222 0172 -22 53	0689 0228 0174 -23 35	0689 0232 0176 -24 19	.0689 .0232 .0175 -24.53	0689 0231 0176 -24 10
		169 1109 11215 113	1789 4 1189 4 1189 4 1189 4 1189 1 1189 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	9-162150225409144 111971225409144 114971569977448 997366666667777448	1887 1887 11142 1142 114	100 110 110 110 110 110 110 110 110 110	1789 1189 80 1118 1118 1118 1118 1118 1118	13107605246507034027646519363707 121111111111111165116866651139707
OIL PRESSURE FUELTS (RF) BOOST (RF) BOOST (LR) AR BOOST (LR) AR BOOST (LR) AR BOOST (LR) FUELT VAC (RF) INLET VAC (LF) INLET VAC (RF) INLET VAC (LF) INLET VAC (RF) INLET V	PAPA PAPA PAPA PAPA PAPA PAPA PAPA PAP	10971-1834[R]:901-0000 1 0 07343454488000999990030 20 42	808070*********************************	N671, N444448800099999000 7 6 6 7 10 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	0080807377454488000988880000 4954	007018243220098880 7 053323588440088810000 7	005018813550084570 6 05535034448800088880000 8 54	006440000074700097700 5 9577507544000000000000005 440000000000000
HYDROCARBONS CAMBON MONOXIDE NITRIC OXIDE NITRICEN OXIDES CAMBON DIOXIDE UXYGEN PARTICULATE		420 141 683 691 15 4 15 5	497 136 764 778 3 6 15 7	646 116 676 709 3.6 14.6	549. 96. 689. 706. 3.6 15.6	684 76 725 744 3 7 15 5	682 67 786 737 3 7 15 4	631 674 678 3 7 14 8
HC Mass CO Mass NUX Mass BSHC BSCO BSNO	GM-HR GM-HR GM-HR GM/BHP-HR GM/BHP-HR GM/BHP-HR	977 648 5940 2 22 1 45 13 47	1147 613 5776 2.60 1.39 13.10	1483. 518. 6180. 3.36 1.18 13.83	1466 422 6055 3 32 96 13 73	1536 334 6640 3 48 76 15 06	1539 293 6906 3.49 15.66	1437 245 6395 3 26 14 50

TABLE C-41. PERFORMANCE AND EMISSION TEST RESULTS, DETROIT DIESEL ENGINE,  $1600\ \text{RPM}$ 

DYNAMUMETER CO	NSTANT <b>3000</b> .	AP	I GRAVIT	Y OF DIE	SEL FUEL	: 33.9 A	1 60F	
RUN NUMBER NOM. WATER PCT		135 0.	141	136 . S	137	130. 15.	139 20	148
FNGINE SPEED OBS. TURQUE	RPM LB-FT	1600. 2143	1600 2143	1600 2143	1600 2143	1600 2143	1600. 2143	1600 2143
BAR PRESS DRY BULB WHI BULB REL HUMIDITY COKR BHP CORR BMEP	IN-HG DEG F PCT HP PSI	29.19 88. 78. 64. 672.5 93.1	29 08 106 77 27 686 3 95 0	29 18 90 78 59 673 9 93 3	29, 17 93, 78, 51, 675, 9	29 17 93 75 43 673 8 93 3	29 15 78 39 677 3 93 8	29 12 97 77 41 681 0 94 3
FUEL FLOW WATER FLOW CALC. VOL % BSFC Alk FLOW L AIR FLOW R	LB/HR CC/MIN PCT LB/BHP-HR LB/MIN LB/MIN	262 97 0 0 0 0 3911 84 5 87 3	263 45 0 0 0 0 3838 80 9 87 9	263 16 120 0 4 8 3905 83 8 89 5	262 68 248 7 9 5 3886 81 3 88 9	262 77 401.5 14.4 3900 81.3 88.4	264 32 570 5 19 2 3902 82 3 88 1	262 97 661 5 21 7 3862 81 2
STOICH. F/A MEAS. F/A CALC. F/A % DIFF.	РСТ	0609 0255 0237 -7 27	0689 0260 0240 -7 73	0689 0253 0236 -6 61	0689 0257 0237 -7.67	0689 0258 0240 -7 12	0689 0259 0241 -6.93	0689 0260 0241 -7 43
COOLANT IN COOLANT OUT OUT OUT OUT OUT OUT OUT OUT OUT OU	######################################	181049993434647899798788888888888888888888888888888	174.6.6.7.8 1186.6.7.8 110.0.9 110.0.0 110.0.0 110.0.0 110.0.0 110.0.0 110.0.0 110.0.0 110.0.0	11379145700887811477878787878991149798878115778988781157789887897977898888781157789888878115778988887811577887877789888878888888888	491548262091620825271196663416888888888888888888888888888888888	78444386 78444396 78444396 78444396 7845 7785 7785 7785 8848 887 888 888 888 888 888 888 888 88	784.6430939658624399888525558420 799888552558420 798888552558420	744631174198093637204521792081411111222271108945217920814
OIL PRESSURE FUEL SPILL NOOST (RF) BOOST (RF) BOUST (LF) BOUST (LF) AIR BOX AI	P 9961 P 9961 P 9961 P 9961 P 1200 P	400513555190065460 5 4776679775200555559020 5476679775200555559020	889871547400030920 5 486657877330055459020 54	58666679773800054250 B	54666679773290032020 B	008030026180020800 > 57676767977320055450017	5466669?7320030907 0 54766669?7320030907 0	546656977320054450020 54656977320054450020
HYDROCARBONS CARBON HONOXID NITRIC DXIDE NITRIC DXIDE CARBON DIOXIDE OXYGEN PARTICULATE	PPMC EPPM SPPM SPPM PCT PCT MG/SCF	588 546 927 927 13 5	476 622 9536 9536 937 13.3	686 397 919 937 4 9 13 3	711 319 942 53 8 13 8	758 254 921 941 5 0 13 4	746 179 900 915 13.6	688 165 927 938 12.9
HC MASS CD MASS NUX MASS BSHC BSCO BSCO	GM-HR GM-HR GM-HR GM-HR GM/BHP-HR GM/BHP-HR GM/BHP-HR	1459 2605 8439 2 25 3 99 12 93	1165 2930 7381 1 79 4 49 11 31	1706 1897 8707 2.61 2.91 13.34	1756 1513 8800 2 69 2 32 13 48	1855 1194 8556 2.84 1.83 13.11	1833 845 8861 2 81 1 29 13 57	1681 775 9272 2 58 1 19 14 20

## APPENDIX D

## REPORT OF NEW TECHNOLOGY

This study documents the unique application of water-in-fuel emulsions to large (900hp to 1200hp) diesel engines. A laboratory system was developed to mix and meter the emulsions to the engine (p. 6 to 11). This system performed well and allowed a determination of the emulsion effects on diesel engine performance.

## **REFERENCES**

- Storment, J. O., and C. W. Coon, "Single-Cylinder Diesel Engine Tests with Unstabilized Water-in-Fuel Emulsions," Report Nos. DOT-TSC-USCG-79-4/ CG-D-13-78, Final Report, U.S. Department of Transportation (August 1978).
- 2. Spadaccini, L. J., and R. Pelmas, "Evaluation of Oil/Water Emulsions for Application in Gas and Turbine Engines," paper presented at Symposium on Evaporation-Combustion of Fuel Droplets (1976).
- 3. Law, C. K., et al, "On the Vapor Pressure, Boiling Point, Burning Characteristics, and Fire Retardancy of Oil/Water Emulsions," Paper No. 79-44, 1979 Fall Meeting of the Western States Section of the Combustion Institute, Lawrence Berkeley Laboratories, Berkeley, CA (15-16 October 1979).
- 4. Dryer, F. L., "Water Addition to Practical Combustion Systems—Concepts and Applications," Sixteenth Symposium (International) on Combustion, The Combustion Institute, Pittsburgh, PA (1977).
- 5. Gollahalli, S. R., M. L. Rasmussen, and S. J. Moussavi, "Combustion of Drops and Sprays of Number 2 Diesel Oil and its Emulsions with Water and Methanol," Report No. DOT/RSPA/DPB-50/80/1, U.S. Department of Transportation, Research and Special Programs Administration, Office of University Research, Washington, DC (January 1980).
- 6. Thompson, R. V., "Application of Emulsified Fuels to Diesel and Boiler Plant," Paper, Institute of Marine Engineers (December 1978).
- 7. Cook, D. H., and C. K. Law, "A Preliminary Study of the Utilization of Water-in-Oil Emulsions in Diesel Engines," Paper, Combustion Science and Technology, Vol. 18 (1978).
- 8. Murayama, T., et al, "Experimental Reduction of  $NO_X$ , Smoke, and BSFC in a Diesel Engine using Uniquely Produced Water (0-80%) to Fuel Emulsion," Society of Automotive Engineers (SAE) Paper 780224 (1978).
- 9. Marshall, W. F., and R. D. Fleming, "Diesel Emissions as Related to Engine Variables and Fuel Characteristics," Society of Automotive Engineers (SAE) Paper 710836 (1971).
- 10. Walter, R. A., "The Emissions and Fuel Economy of a Detroit Diesel 6-71 Engine Burning a 10 Percent Water-in-Fuel Emulsion," Report No. CG-D-10-78, U.S. Department of Transportation, United States Coast Guard, Office of Research and Development (1978).
- 11. Lawson, A., and A. J. Last, "Modified Fuels for Diesel Engines by Application of Unstabilized Emulsions," Society of Automotive Engineers (SAE) Paper 790925 (1979).

## REFERENCES (CONTINUED)

- 12. Federal Register, 42FR174, P-45169.
- 13. Society of Automotive Engineers, "Engine Test Code-Spark Ignition and Diesel," SAE Standard J816b, SAE Handbook (1978).
- 14. Meriam Instrument Division, Scott and Fetger Co., "Installation and Operating Instructions; Meriam Laminar Flow Elements," (1975).
- 15. American Society for Testing and Materials, <u>Petroleum Measurements Tables</u>, American Edition, ASTM (1952).
- 16. Ostle, B., Statistics in Research, 2nd, Iowa State University Press, Ames, IA (1963).

☆U. S. GOVERNMENT PRINTING OFFICE: 1981--701-545--81